STRALIS AT/AD

REPAIR MANUAL MECHANICAL ELECTRIC ELECTRONIC







This publication describes the characteristics, the data, the correct methodology of the repairs that can be made on each individual component of the vehicle.

By complying with the instructions supplied and using the specific tools it is possible to perform any repair intervention correctly, within the specified time frames, while protecting the technicians against incidents.

Before starting any repair work, make sure that all accident prevention devices are ready at hand.

Check and wear the protective personal equipment provided for by the safety standards: goggles, helmet, gloves, shoes.

Check the efficiency of all processing, lifting and transport tools before using them.

The data contained in this publication might fail to reflect the latest changes which the Manufacturer may introduce at any time, for technical or sales purposes, or to meet the requirements of local legislation.

Copy, even partial, of text and drawings is forbidden.

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	SPECIAL REMARKS	
are indicated in the Gene The subjects usually deal Technical data table, Driv Where possible, the sam Diagrams and symbols ha	for mechanical parts have been divided into Sections, each of which eral Specifications. Each section features a main Unit (e.g. engine, g It with in each section are: ving torques, Equipment, Diagnostic, Removal and Fitting in place, ne sequence of procedures has been followed for easy reference. ave been widely used to give a clearer and more immediate illustrat ring descriptions of some operations or procedures.	gears etc.). Repair operations.
Example		
Ø I		hten to torque hten to torque +
Ø2		gular value
digit number. This numbe and in the FAULT CODI	h section, every heading or sub-heading concerning the operations er is the Product Code that is to be found in the repair operation de ES. indication of how to read this code is described below (see the R	escribed in the REPAIR TIMES CHARTS
Product Code:	S O PRODUCT UNIT SUB-ASSEMBLY COMPONENT	
Product 52 =	Frame; Axles; Gears etc.	
Unit Code:	PRODUCT UNIT SUB-ASSEMBLY COMPONENT	
	lentify the ASSEMBLY within the PRODUCT	
Unit 01 =	Frame; Chassis; Bumpers etc .	
Sub-assembly Code:	PRODUCT UNIT SUB-ASSEMBLY COMPONENT	
Unit 01 =	Frame; Chassis; Chassis cross members etc.	

Graphs and symbols

	Removal Disconnection
	Refitting Connection
	Removal Disassembly
	Fitting in place Assembly
	Tighten to torque
$\widehat{\mathfrak{P}}_{a}$	Tighten to torque + angle value
••	Press or caulk
846	Regulation Adjustment
	Warning Note
	Visual inspection Fitting position check
F	Measurement Value to find Check
P	Equipment
<u> </u>	Surface for machining Machine finish
Ś	Interference Strained assembly
	Thickness Clearance
	Lubrication Damp Grease
	Sealant Adhesive
	Air bleeding

	Intake	
Þ	Exhaust	
$\langle \neg \rangle$	Operation	
Q	Compression ratio	
	Tolerance Weight difference	
	Rolling torque	
IVECO	Replacement Original spare parts	
	Rotation	
\triangleleft	Angle Angular value	
	Preload	
	Number of revolutions	
	Temperature	
bar	Pressure	
>	Oversized Higher than Maximum, peak	
<	Undersized Less than Minimum	
A	Selection Classes Oversizing	
	Temperature < 0° Cold Winter	
	Temperature > 0° Hot Summer	

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UPDATE DATA

Section	Description	Page	Revision date

INDEX OF SECTIONS

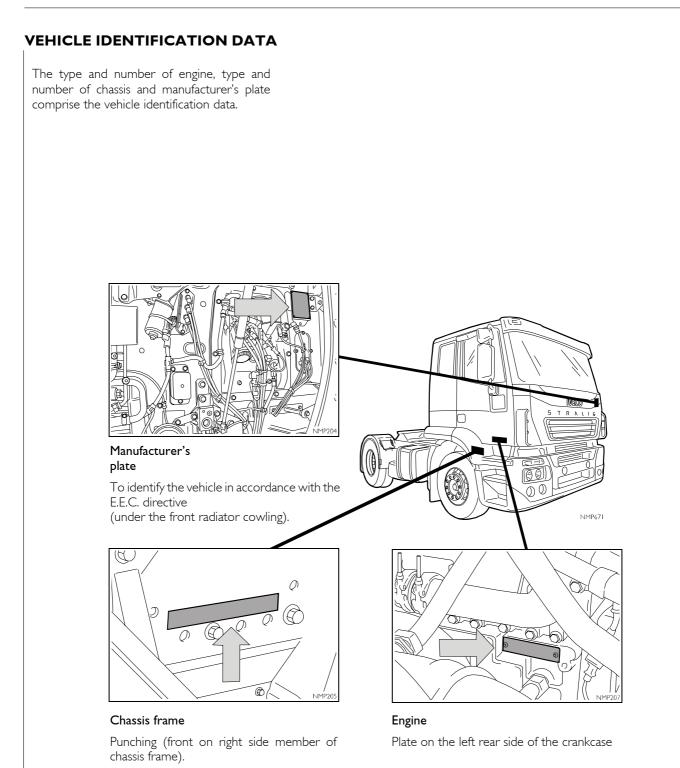
	Section
General information	
Engine	2
Clutch	3
Gearbox	4
Hydraulic retarder	5
Propeller shafts	6
Rear axles	7
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Front and rear suspensions	9
Wheels and tyres	10
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SECTION I

General

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REPLENISHING FLUIDS	13

2 GENERAL



Stralis AT/AD

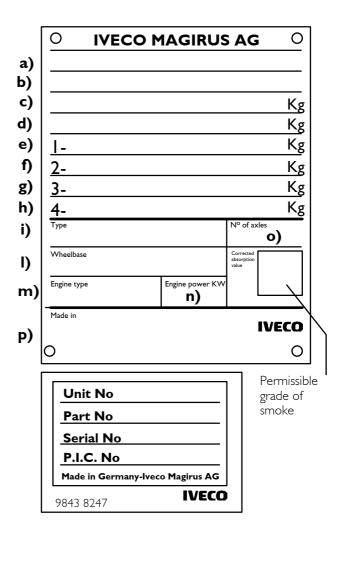
Vehicle Identification Plate

Plate legend

- a) Type-approval number marking (if applicable).
- b) Vehicle identification code number (V.I.N.).
- c) Total tractor weight.
- d) Total weight of tractor + trailer (if applicable).
- e) Permissible weight limit on 1 st axle.
- f) Permissible weight limit on 2nd axle (if applicable).
- g) Permissible weight limit on 3rd axle.
- h) Permissible weight limit on 4th axle (if applicable).
- i) Specific identification of type.
- I) Wheelbase in mm.
- m) Engine type.
- n) Engine power.
- o) No. of axles.
- p) Place of manufacture.



This plate shows the P.I.C. (production identification code number), which is needed when referring to the **spare parts catalogue** (electronic and/or microfiche catalogue). The P.I.C. is also given on the vehicle warranty card. **Note**: When consulting the catalogues, use only the first 8 digits of the product identification code number.



5

COMPOSITION OF MODELS

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ASSEMBLIES			AD 190 S	AT 190 S 27	AD 190 S 27/P	AT 190 S 27/P	AD 190 S	AT 190 S 27/FP-D AD 190 S 30	AD 190 S 30/P	AD 190 S 31	AT 190 S S31	AD 190 S 31/P	AD 190 S 31/FP-D	AT 190 S 31/FP-D	AD 190 S 35	AT 190 S 35	AT 190 S 35/P	AD 190 S 35/FP-D	AI 190 S 35/HP-U AD 190 S 40	AT 190 S 40	AD 190 S	AT 190 S 40/P	AD 190 S 40/FP-D	AT 190 S 40/FP-D	AT 190 S 40/FP-CT AD 190 S 43	AT 190 S 4	AD 190 S 43/P	AT 190 S 43/P	
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⊗ = With brake calliper assembly at 57° without parking brake

□ = With parabolic leaf springs

• With brake calliper assembly at 0° with parking brake

* = TI with drum brakes

With longitudinal and transversal bars

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- = 4x2 tractor

Т

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- TX = 6x2 C tractor (central added axle cannot be lifted)
- TY = 6x2 P tractor (rear added axle can be lifted)
- TN = 6x2 vehicles with mechanical rear suspensions and raisable rigid rear added axle
- TZ = 6x4 tractor (bogie rear axle)
 - = 4x2 6x2P 6x2C vehicles with air suspension on rear axle and 6x2P vehicles with rigid rear axle that can be lifted with single wheels
- PT = 6x2P vehicles with air suspension on rear axle and rigid rear added axle that can be lifted with twin wheels
- PS = 6x2P vehicles with air suspension on rear axle and on steering rear added axle that can be lifted with single wheels
- FP = $4x^2 6x^4 6x^2P 6x^2C$ vehicles with front and rear air suspensions
- FS = 6x2P vehicles with front and rear air suspensions, steering rear added axle can be lifted with single wheels
- 4x2 = Vehicles with two axles with rear driving axle
- 6x2P = Vehicles with three axles with rear driving axle and rear added third axle that can be lifted
- 6x2C = Vehicles with three axles with rear driving axle and central added third axle that cannot be lifted
- 6x4 = Vehicles with three axles with two rear driving axles (in tandem)
- CM = Movable Boxes
- LT = Tractor with lowered chassis frame
- CT = Chassis cab with lowered chassis frame
- RR = Rough Roads
- D = Distribution
- HR = Hub Reduction
- AT = Active Time
- AD = Active Day

Base - January 2003

 \diamond = With longitudinal and transversal bars

COMPOSITION OF MODELS

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Т	=	4x2 tractor
ΤX	=	6x2 C tractor (central added axle cannot be lifted)
ΤY	=	6x2 P tractor (rear added axle can be lifted)
TN	=	6x2 vehicles with mechanical rear suspensions and raisable rigid rear added axle
ΤZ	=	6x4 tractor (bogie rear axle)
Ρ	=	4x2 - 6x2P - 6x2C vehicles with air suspension on rear axle and $6x2P$ vehicles with rigid rear axle that can be lifted with single wheels
PT	=	6x2P vehicles with air suspension on rear axle and rigid rear added axle that can be lifted with twin wheels
PS	=	6x2P vehicles with air suspension on rear axle and on steering rear added axle that can be lifted with single wheels
FP	=	4x2 - 6x4 - 6x2P - 6x2C vehicles with front and rear air suspensions
FS	=	6x2P vehicles with front and rear air suspensions, steering rear added axle can be lifted with single wheels
4x2	=	Vehicles with two axles with rear driving axle
6x2P	=	Vehicles with three axles with rear driving axle and rear added third axle that can be lifted
6x2C	=	Vehicles with three axles with rear driving axle and central added third axle that cannot be lifted
6x4	=	Vehicles with three axles with two rear driving axles (in tandem)
CM	=	Movable Boxes
ΗM	=	Heavy Mission
LT	=	Tractor with lowered chassis frame
CT	=	Chassis cab with lowered chassis frame
RR	=	Rough Roads
D	=	Distribution
AT	=	Active Time
AD	=	Active Day

7

COMPOSITION OF MODELS

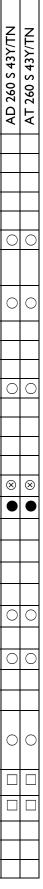
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/ith brake calliper assembly at 57° without parking brake = With brake calliper assembly at 0° with parking brake •

= TI with drum brakes

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= With longitudinal and transversal bars



= 4x2 tractor

Т

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- ΤX $= 6 \times 2$ C tractor (central added axle cannot be lifted)
- ΤY $= 6 \times 2 P \text{ tractor}$ (rear added axle can be lifted)
- ΤN $= 6x^2$ vehicles with mechanical rear suspensions and raisable rigid rear added axle
- ΤZ = 6x4 tractor (bogie rear axle)
 - = $4x^2 6x^2P 6x^2C$ vehicles with air suspension on rear axle and 6x2P vehicles with rigid rear axle that can be lifted with single wheels
- PΤ $= 6 \times 2P$ vehicles with air suspension on rear axle and rigid rear added axle that can be lifted with twin wheels
- PS $= 6 \times 2P$ vehicles with air suspension on rear axle and on steering rear added axle that can be lifted with single wheels
- FP = $4x^2 - 6x^4 - 6x^2P - 6x^2C$ vehicles with front and rear air suspensions
- FS = $6 \times 2P$ vehicles with front and rear air suspensions, steering rear added axle can be lifted with single wheels
- 4x2 = Vehicles with two axles with rear driving axle
- 6x2P = Vehicles with three axles with rear driving axle and rear added third axle that can be lifted
- 6x2C = Vehicles with three axles with rear driving axle and central added third axle that cannot be lifted
- = Vehicles with three axles with two rear 6x4 driving axles (in tandem)
- СМ = Movable Boxes
- LT = Tractor with lowered chassis frame
- CT = Chassis cab with lowered chassis frame
- RR = Rough Roads
- D = Distribution
- HR = Hub Reduction
- AT = Active Time
- AD = Active Day

			AD 440 S 31T/P AT 440 S 31T/P		AD 440 S 35T/P-HR	AT 440 S 35T/P-HR AD 440 S 40T/P	AT 440 S 40T/P	AD 440 S 40T/P-RR AT 440 S 40T/P-RR	AD 440 S 40T/P-HR AT 440 S 40T/P-HR	AT 440 S 40T/FP-C1	AT 440 S 40T/FP-LT AD 440 S 43T/P	AT 440 S 43T/P AD 440 S 43T/P_RR	AT 440 S 43T/P-RR	AD 440 S 431/P-HR AT 440 S 43T/P-HR	AT 440 S 43T/FP-CT AT 440 S 43T/FP-LT	AD 440 S 40TX/P	AT 440 S 40TX/P AD 440 S 43TX/P	AU 440 5 431 X/F AT 440 S 43TXP	1 440 5 431 AF
MBLIES			< <	< <	ג ∢	< <	< -	< <		₹	<u>A</u> A	< <		< <	<u>ک</u> ک	< ∢		[▲	1
	F2BE0681F (270 CV)															_	+		
$\overline{\overline{\mathbf{u}}}$	F2AE0681E (300 CV)															_			
<u></u>	F2BE0681B (310 CV) F2BE0681A (350 CV)		00			\bigcirc													
<u></u>	F3AE0681B (400 CV)								00		\bigcirc					\cap			
_	F3AE0681D (430 CV)				_							00		$\cap \cap$	\cap				$\overline{\mathbb{C}}$
																		-	_
H an t	Single disc 16''																		
 -1	Single disc 17''		00			00	0	00	OC	\circ	OC	00		00	00		00	C	С
	ZF 9S 109 D.D.																		
	ZF 16S 151 O.D.		00	$) \bigcirc ($	O	\bigcirc													
	ZF 16S 181 O.D.								OC					00			\circ \circ \circ	C	0
I.	ZF 16S 181 D.D.					0	\circ	00		0	OC	00			\bigcirc)			
	ZF 16S 221 D.D.																		
	EuroTronic Automated 12 AS 2301 D.D.								OC					00					_
	EuroTronic Automated 12 AS 2301 O.D.		00			00	0	_		0	OC				O(C)				_
	Allison MD 3060 P - MD 3066 P FRONT AXLE:	5876/4 (F 8021)	88) 🛞 (§	⊘ ⊗	\otimes \otimes	\otimes	$\otimes \otimes$	\otimes \otimes	\otimes	\otimes \otimes	88	\otimes	⊗ ⊗	\otimes \otimes	\otimes	8 8	3 8	Ø
		5876/5 (F 8021)	•			••		• •	• •		••			• •	•				ō
		5886/5 (F 9021)																	
Ŀ	ADDED AXLE:																	—	
÷ ,	Steering central	5876/4 (F 8021)														0			$\overline{\mathbb{C}}$
	Rigid rear	55080/DI (N 807I) *																	
	Rigid rear	56082/DI (N 9171) *																	
	Steering rear	57080/DI (N 8072) *																	
	MERITOR MS 13-175/T - MS 13-175/D		00			00	\circ	00		0	OC	00			00			C	J
	MERITOR RT 160/1																		
	451391 HR								OC)				00					
	ZF 8098					00	0	00	00		00			00	00		00		С
5	FRONT MECHANICAL	Front								1								╗┼┍	
ير مورب		Rear								-								+	_
$\overline{}$	PNEUMATIC	Front								0	0		++	_	00		++	+	
		Rear	00	$) \bigcirc ($	O	00	\circ	00	OC	$) \bigcirc$	OC	00		00	\bigcirc	$) \bigcirc$) (5
1. je		Added axle																	_

With longitudinal and transversal bars

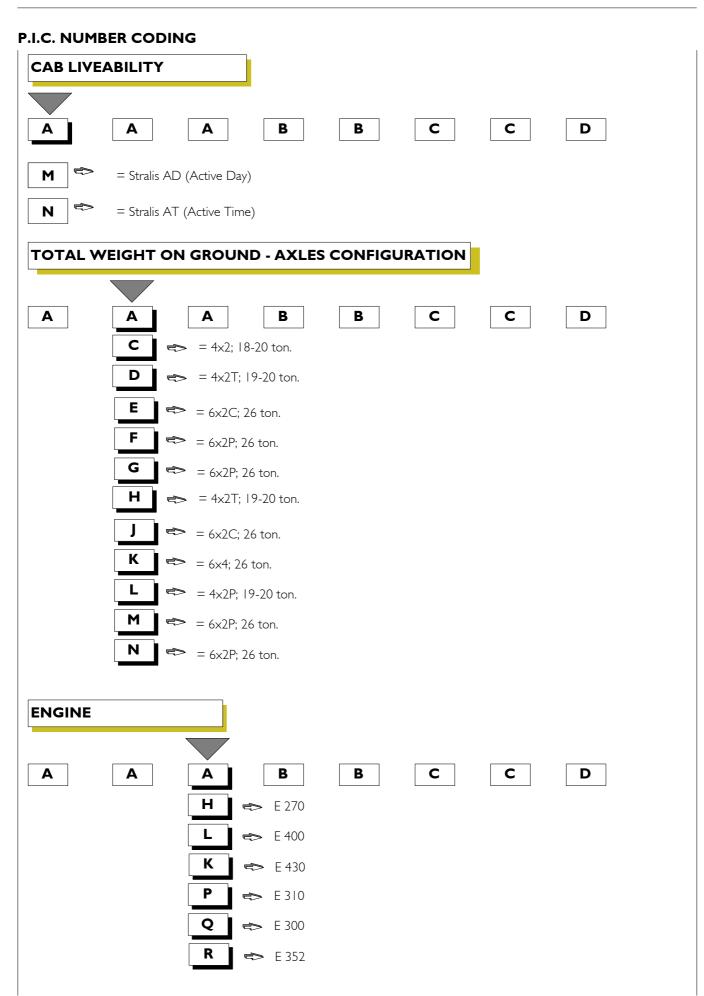
4x2

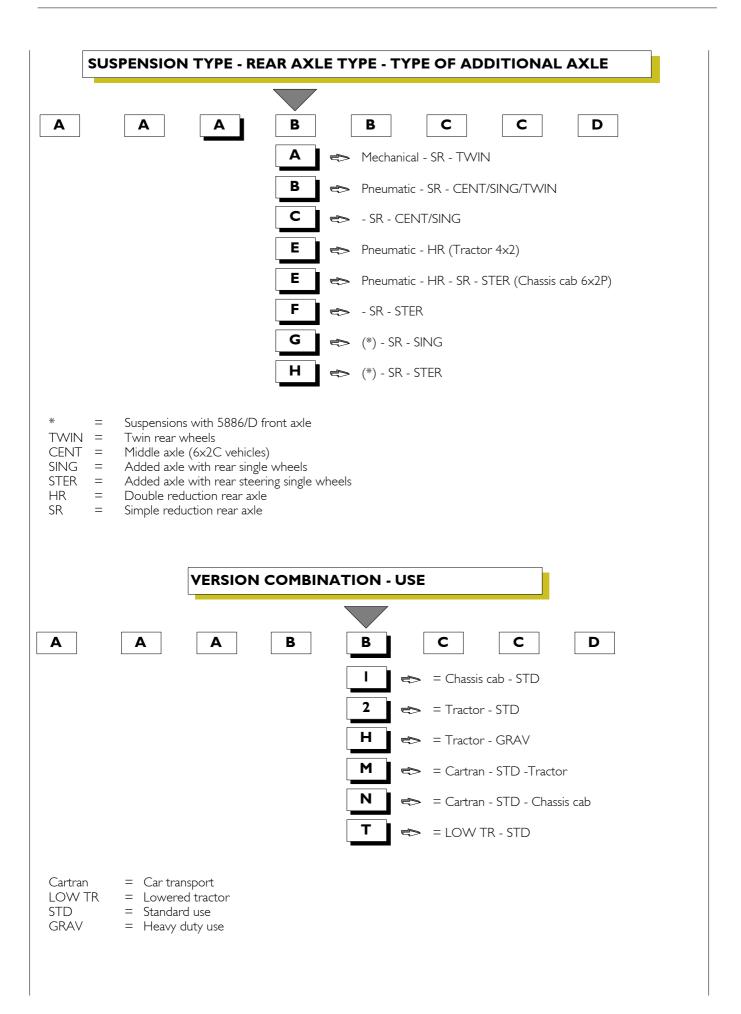
6x2 C 6x4

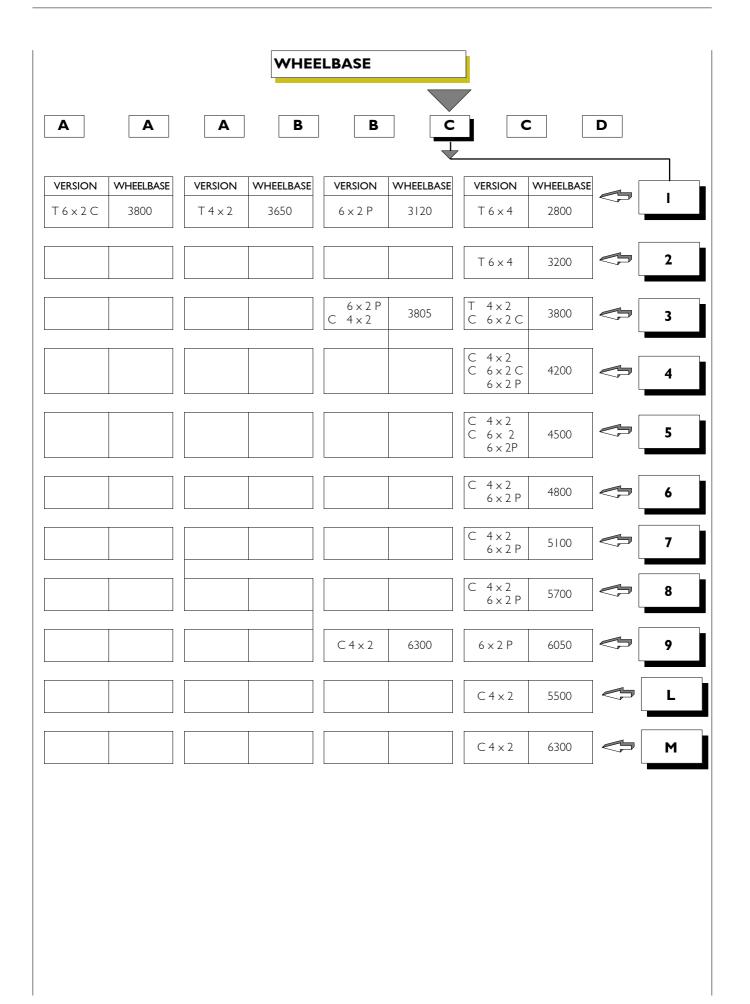
TRACTORS - MODELS

COMPOSITION OF MODELS

Т	=	4x2 tractor
ΤX	=	6x2 C tractor (central added axle cannot be lifted)
ΤY	=	6x2 P tractor (rear added axle can be lifted)
TN	=	6x2 vehicles with mechanical rear suspensions and raisable rigid rear added axle
ΤZ	=	6x4 tractor (bogie rear axle)
Ρ	=	4x2 - 6x2P - 6x2C vehicles with air suspension on rear axle and $6x2P$ vehicles with rigid rear axle that can be lifted with single wheels
PT	=	6x2P vehicles with air suspension on rear axle and rigid rear added axle that can be lifted with twin wheels
PS	=	6x2P vehicles with air suspension on rear axle and on steering rear added axle that can be lifted with single wheels
FP	=	4x2 - 6x4 - 6x2P - 6x2C vehicles with front and rear air suspensions
FS	=	6x2P vehicles with front and rear air suspensions, steering rear added axle can be lifted with single wheels
4x2	=	Vehicles with two axles with rear driving axle
6x2P	=	Vehicles with three axles with rear driving axle and rear added third axle that can be lifted
6x2C	=	Vehicles with three axles with rear driving axle and central added third axle that cannot be lifted
6x4	=	Vehicles with three axles with two rear driving axles (in tandem)
CM	=	Movable Boxes
HM	=	Heavy Mission
LT	=	Tractor with lowered chassis frame
CT	=	Chassis cab with lowered chassis frame
RR	=	Rough Roads
D	=	Distribution
AT	=	Active Time
AD	=	Active Day

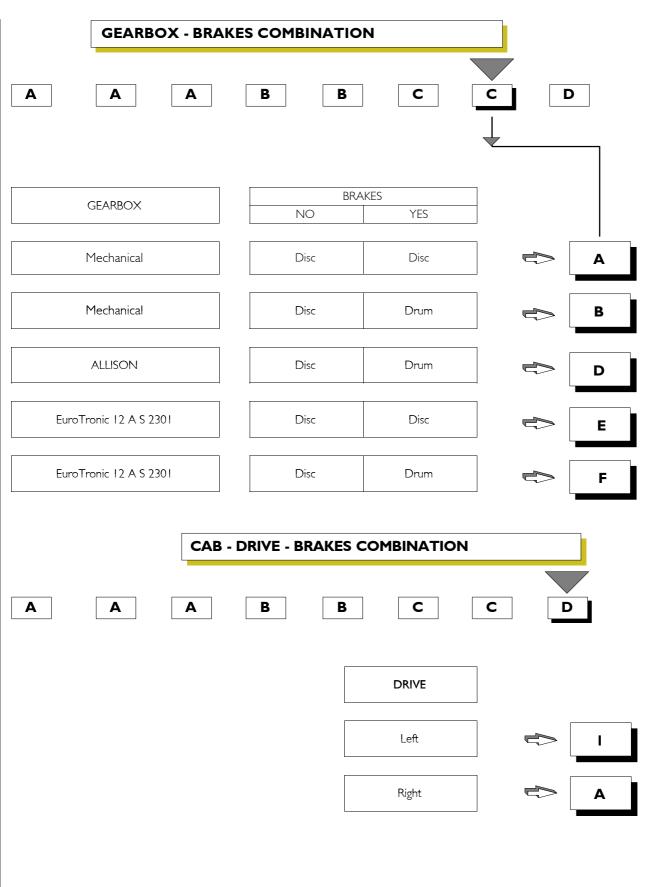






GENERAL

12



REPLENISHING FLUIDS

JBRICANTS RECOMMENDED BY IVECO	PARTS TO BE FILLED UP	Quantity	
		Litres	Kg
Urania FE 5W30(1) Urania LD5 Urania Turbo LD	Engine F2B		
	Total capacity for first filling	28	25,2
	Capacity:		
	- engine sump min level	12.5	11,2
	- engine sump max level	23	21
	 quantity in circulation that does not flow back to the en- gine sump 	5	4.5
	 quantity contained in the car- tridge filter (which has to be added to the cartridge filter re- fill) 	2.5	2.3
Urania FE 5W30 ⁽¹⁾ Urania LD5	Engine F3A		
Urania Turbo LD	Total capacity I st filling Capacity:	30	29.8
	- engine sump at minimum level	17	15.3
	- engine sump at maximum level	25	22.5
	 quantity in circulation that does not re- turn to sump 	7	6.3
	 quantity contained in cartridge filter (to add when changing the cartridge filter) 	2.5	2.3
	Gearbox		
Tutela Truck FE-Gear ¹	ZF 9 S 109 * ZF 16 S 151 * ZF 16 S 151 + Intarder * ZF 16 S 181 * ZF 16 S 181 + Intarder* ZF 16 S 221 *	8 11 18.5 13 21.5 13 21.5	7.2 10 16.65 12 19.35 12 19.35
Quantity I st filling	ZF 16 S 221 + Intarder* EuroTronic automated 12 AS 2301*	12	9.53
	EuroTronic automated 12 AS 2301* + intarder	23	21
Tutela GI/A	Allison MD 3060 P - MD 3066 P	18	16
Tutela Truck FE-Gear ¹ Tutela ZC 90	Power take off (Multipower)	2.5	
) IVECO recommends using these oils for reasons o lubricants. Also suited for cold climates (minimum hese quantities are not decisive. An exact check mus		y with thes	e types

REPLENISHING FLUIDS

IRP	ICANTS RECOMMENDED BY IVEC				ntity	
חסנ	ICAINTS RECOMMENDED BT IVEC	PARTS FOR	REPLENISHING	Litres	kg	
	Tutela Truck FE-Axle ⁽²⁾	Front hubs (single) FRONT AXLE FRONT AXLE ADDED AXLE::	5876/4-/5 (F8021) 5886/5	0.35 0.35	0.32 0.32	
	Tutela W140/M-DA Tutela W90/M-DA ³	Steering central Rigid rear Rigid rear Steering rear	5876/4 (F 8021) 55080/DI (N 8071) 56082/DI (N 9171) 57080/DI (N 8072)	0.35 0.35 0.35 0.35	0.32 0.32 0.32 0.32	
		Bridge Meritor MS 13-1	175/T - MS 13-175/D:			
	Tutela Truck FE-Axle ⁽²⁾ Tutela WI 40/M-DA	□ (mechanical □ (pneumatic s Bridge 451391 HR		8.5 7 6	16.5 15.5 14.5	
	Tutela W90/M-DA ³	Rear axle Meritor in tanc - middle - rear	dem RT 160E/1	8.5 6.5	6.0 4.8	
	Tutela GI/A	Power steering		2.7* 3.5**	2. 1	
		* Excluding vehicles with ** For vehicles with steer				
SOT SPECIAL	Tutela TRUCK DOT SPECIAL	Clutch circuit (excluding vehicles with Euro Tronic gearbox)		0.5	0.4	
)+	Water+Paraflu ¹¹	Cooling system Engine F2B Engine F2B with Intarder Engine F3A Engine F3A with Intarder	Total capacity* Total capacity* Total capacity* Total capacity*	~34 ~50 ~38 ~58	$\sim 23. \\ \sim 27 \\ \sim 39. \\ \sim 57. \end{cases}$	
		* = Protective anti-freeze (concentration 50% freez				
	Tutela LHM	Cab tilting system		0.6	0.5	

(2) Only on axles with disc brakes, IVECO recommends using these oils for reasons of fuel economy. IVECO provides new vehicles already with these types of lubricants. Also suited for cold climates (minimum temperature down to -30°C)
 (3) Specific for cold climates

These quantities are not decisive. An exact check must be made by verifying the levels

		FL products
ingine oil Neets the specifications: ACEA E4 with a totally synthetic base ACEA E5 with a mineral base ACEA E3 with a mineral base	SAE 50W 30 SAE 15W 40 SAE 15W 40	URANIA FE 5W30 Urania LD5 Urania Turbo LD
Oil for differential gear and wheel hubs Meets the specifications: API GL5, MT-I with a totally synthetic base API GL5 with a mineral base API GL5 with a mineral base	SAE 75W 90 SAE 85W 140 SAE 80W 90	Tutela Truck FE-Axle Tutela W140/M-DA Tutela W90/M-DA
Oil for mechanical gearboxes Containing non-EP anti-wear additives Meets the specifications: API GL4 with a totally synthetic base API GL3 with a mineral base	SAE 75W 85 SAE 80W 90	Tutela Truck FE-Gear Tutela ZC90
Oil for power steering and hydrostatic transmissions A.T.F. DEXRON II D	S	Tutela GI/A
Grease for general greasing based on lithium soaps, N.L.G.I. consistency no. 2		Tutela MR 2
pecific grease for bearings and wheel hubs based on lithium soaps, N.L.G.I. consistency no. 3		Tutela MR 3
Clutch drive fluid Conforming to N.H.T.S.A. standards 116, ISO 4925, St VECO STANDARD 18-1820	td. SAEJ 1703,	Tutela TRUCK DOT SPECIAL
1ineral oil for hydraulic circuits n compliance with IVE CO STANDARD 18-1823		Tutela LHM
Vindscreen washer fluid, mixture of spirit, water and UNA NC 956-11	l surfactants	Tutela PROFESSIONAL SC 35
rease for central lubrication systems based on lithic th synthetic base, N.L.G.I. no. 2. /orking temperatures: from -30°C to +140°C	um soaps,	Tutela COMAR 2
Concentrated protective fluid for radiators ased on ethylene glycol containing corrosion inhibito candard: IVECO-STANDARD 18-1830	ors, conforming to the	Paraflu ¹¹

I6 GENERAL

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F2B Engine

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	Mounting the piston rings
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CYI	LINDER HEAD
	Valve removal
	Checking the planarity of the head on the cylinder block
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	Checking cam lift and pin alignment
	Camshaft
	Bushes
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	Assembly
VA	LVE SPRINGS
	Fitting the valves and oil seal ring

ROCKER SHAFT

ASSEMBLING THE ENGINE ON THE BENCH .

DIAGRAM SHOWING THE UNDERBLOCK FIXING SCREWS TIGHTENING ORDER

Rocker

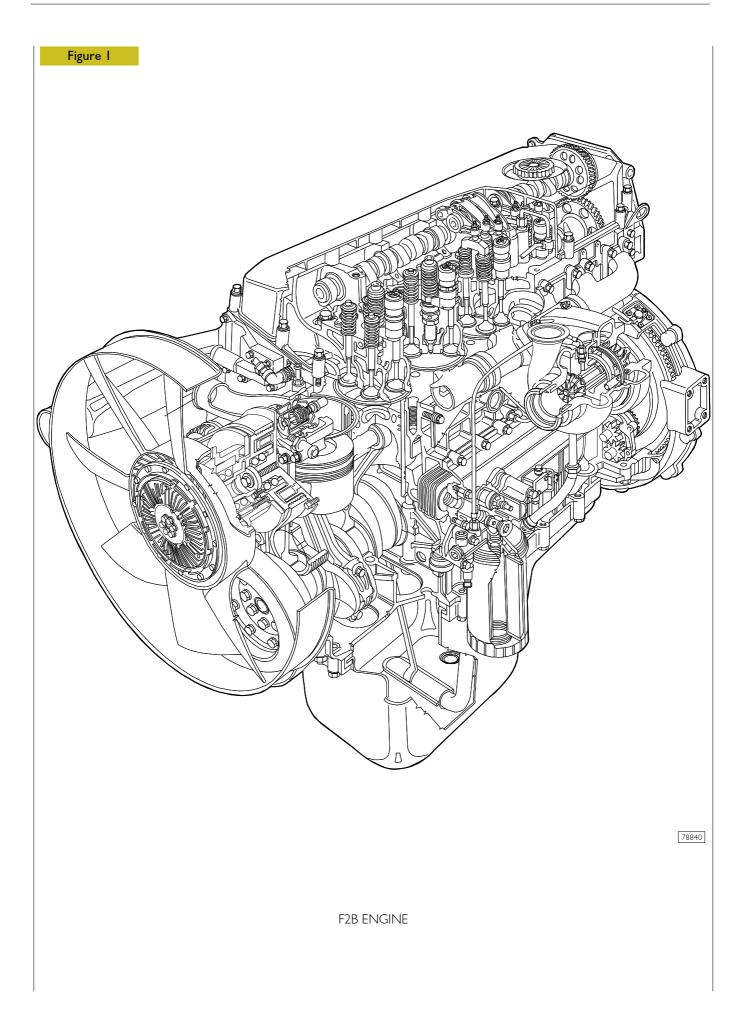
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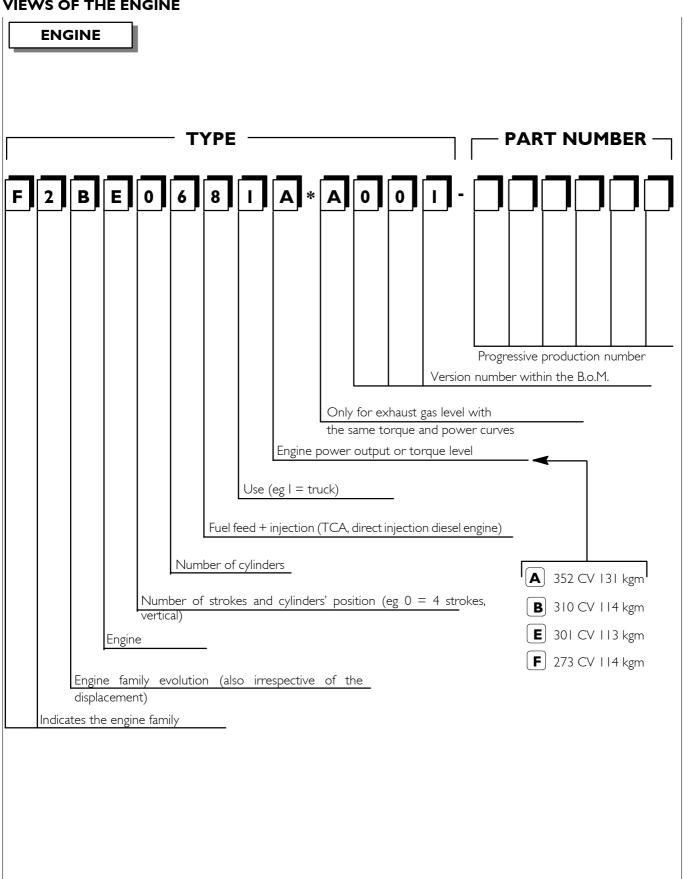
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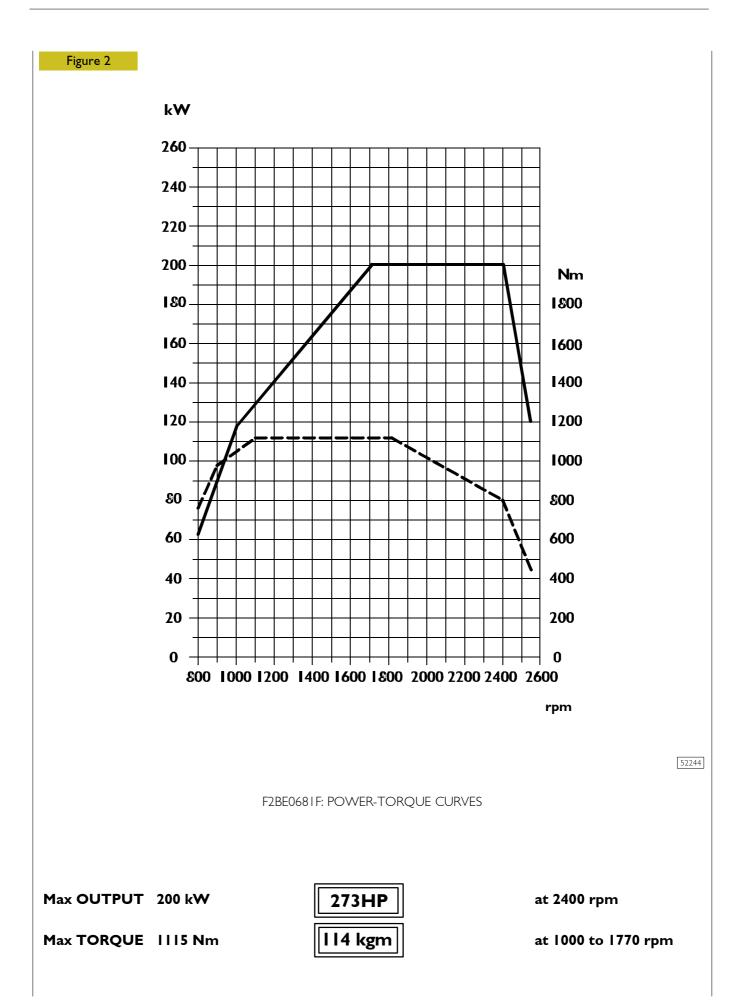
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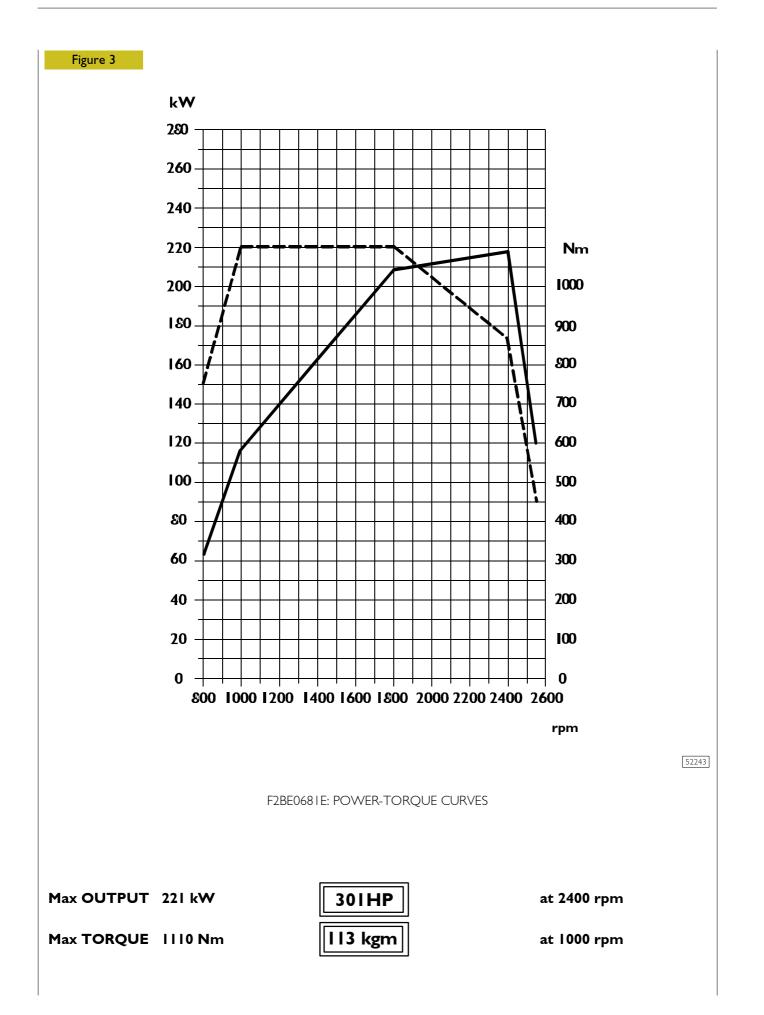
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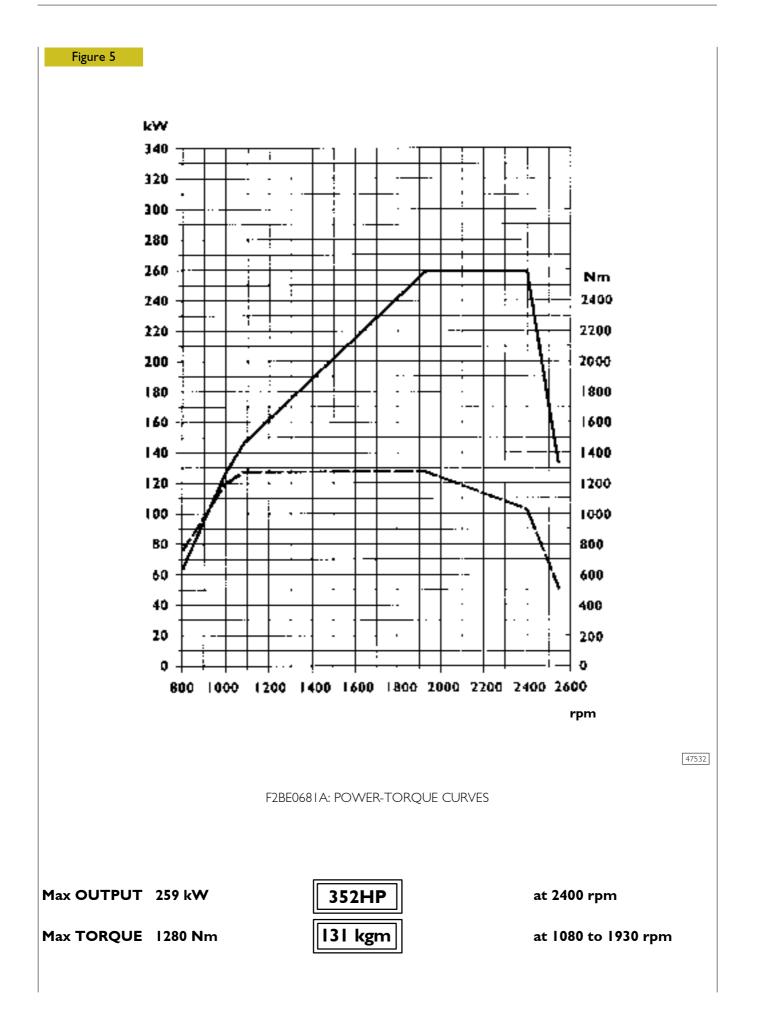






ENGINE F2B

Stralis AT/AD



GENERAL CHARACTERISTICS

	Туре		F2BE0681A	F2BE0681B	F2BE0681E	F2BE0681F
A	Cycle			Diesel 4	strokes	
	Feeding			Turbochargeo	with aftercooler	
	Injection			Din	ect	
	N. of cylinders			6 on	-line	
	Diameter	mm		11	5	
	Stroke	mm		12	25	
↓ ↓ ↓ ↓ ↓ ↓ ↓ ↓ ↓ ↓	Total displacement	cm ³	7790			
Q	Compression ratio		16 ± 0.8			
	Max. power	KW (HP)	259 (352)	228 (310)	221 (301)	200 (273)
		rpm	2400	2400	2400	2400
	Max. torque	Nm (Kgm)	280 (3)	5 (4)	0 (3)	5 (4)
		rpm	1080 to 1930	1000 to 1950	1000	1000 to 1770
	Engine idling speed, no load	, rpm		525 :	+ 25	
	Maximum engine speed, no load					
		rpm		2760	± 20	

VALVE TIMING opens before T.D.C. A 17° closes after B.D.C. B 31° opens before B.D.C. D 48° opens before B.D.C. C 9° For timing check - - Image: State of the sta	Туре	F2B
image: specific specif	opens before T.D.C. A	
X mm Running mm X mm Nozzle type Image: State of the sta		48°
Injection type Bosch With electronically regulated injectors PDE 30 pump injectors controlled by overhead camshaft Nozzle type	X { mr mr Running X { mr	0.35 to 0.45
Nozzle type	Injection	
Injection order Injection pressure bar Injection pressure bar Injection pressure bar	Nozzle type	_
Injector calibration bar	Injection order	I - 4 - 2 - 6 - 3 - 5
「		

	Туре	F2B
	SUPERCHARGING Turbocharger	Holset, with variable geometry
	type:	HY 40 V
	LUBRICATION	Forced by gear pump, pressure control valve, oil filter
bar	Oil pressure, engine hot (100 °C ± 5 °C): at idling speed bar at maximum speed bar	I.5 5
COOLING		By centrifugal pump, regulating thermostat, viscostatic fan, radiator and heat exchanger
	Water pump control	By belt
	Thermostat:	N. I
	starts to open:	~85 °C
	fully open:	_
	OIL FILLING	
	Total capacity at 1st filling liters	28
	kg	25.5
	Capacity: - engine sump min level liters	
Fiat lubricants	kg	12.5
Urania Turbo LD (according to E3-96	- engine sump max level liters	11.2
standard) Urania Turbo (according to E2-96 standard)	kg - quantity in circulation that does not flow back to the	23 21
	engine sump liters kg	5
	- quantity contained in the cartridge filter (which has to be added to the cartridge filter refill)	
	liters kg	2.5 2.3

ASSEMBLY CLEARANCE DATA

	Туре	F2B
CYLINDER BLOCK A MECHANISM COMPO		mm
	Cylinder sleeve bore upper Ø 1 lower	130.200 to 130.225 128.510 to 128.535
Z L	Cylinder liners: outer diameter: upper Ø 2 lower length L	30. 6 to 30. 86 28.475 to 28.500
	Cylinder sleeve - crankcase bore upper lower	0.014 to 0.064 0.010 to 0.060
	Cylinder sleeve	ð 2
	inside diameter G	A* II5.000 to II5.012 Ø 3 II5.010 to II5.022 K 0.035 to 0.065
* Available dia. class	Pistons: measuring dimension > outside diameter Ø outside diameter Ø outside diameter Ø 2	A• 14.888 to 4.900
 Class A pistons supp Class B pistons are f are not supplied as s 	olied as spares. fitted in production only a	114.898 to 114.910 nd
	Piston - cylinder sleeve	0.100 to 0.124
	Piston diameter 🖇	<u>_</u>
	Pistons protrusion >	< 0.32 to 0.99
Ø3	Gudgeon pin 🖇	Ø 3 45.994 to 46.000
	Gudgeon pin - pin hous	ing 0.010 to 0.024

	Туре		F2B
	туре		mm
∬ ¥ xi		X *	2.71 to 2.74
	Piston ring grooves	Х2	2.55 to 2.57
		X3	4.02 to 4.04
	*measured on \varnothing of []	2 mm	
	Piston rings: trapezoidal seal	SI*	2.575 to 2.595
¥ ∬ S I	lune seal	S2	2.470 to 2.490
	milled scraper ring with slits and internal spring	\$3	3.975 to 3.990
	*measured on \varnothing of []	2 mm	
			0.115 to 0.165
	Piston rings -	С	0.060 to 0.100
	grooves	2 3	0.030 to 0.065
	Piston rings		_
	Piston ring end gap in cylinder liners:	1	
► { X2	,	ХI	0.35 to 0.50
×3		X2	0.70 to 0.96
		X3	0.30 to 0.60
(), ↓ ØI	Small end bush housi	ng Ø I	49.975 to 50.000
	Big end bearing housing	Ø2	Rated value 77.000 to 77.030
∅ 2	Selection classes Ø2	 2 3	77.000 to 77.010 77.010 to 77.020 77.020 to 77.030
Ø 4	Small end bush diame	eter	
	outside	Ø4	50.055 to 50.080
	inside 🔟	Ø 3	46.015 to 46.030
S S	Big end bearing shell Red Green	S	2.000 to 2.010 2.011 to 2.020 2.021 to 2.020
	Yellow Small end bush - hou	sing	2.021 to 2.030 0.055 to 0.105
	Piston pin - bush	JIIIR	0.015 to 0.036
PRATS A	Big end bearing shells		0.127 - 0.254 - 0.508
\bigcirc	Connecting rod weig	ht A	g. 2890 to 2920
	Class	В	g. 2921 to 2950
		С	g. 2951 to 2980

		F2B
	Туре	mm
X	Measuring dimension X	125
	Max. connecting rod axis misalignment	0.08
│Ť	tolerance	
	Main journals Ø I Selection class { 2	Rated value 82.910 to 82.940 82.910 to 82.919 82.920 to 82.929
ØI <u>Ø</u> 2	Selection class23CrankpinsØ 2	82.930 to 82.940 Rated value 72.915 to 72.945
	Selection class	72.915 to 72.924 72.925 to 72.934 72.935 to 72.945
	Main bearing shells SI	
	Red Green Yellow ●	3.000 to 3.010 3.011 to 3.020 3.021 to 3.030
	Big end bearing shells S2 Red Green	2.000 to 2.010 2.011 to 2.020 2.021 to 2.030
	Yellow ● Main bearing housings Ø3	Rated value 89.000 to 89.030
Ø 3	Selection class $\begin{cases} 1\\ 2\\ 3 \end{cases}$	89.000 to 89.009 89.010 to 89.019 89.020 to 89.030
	Bearing shells - main journals	0.040 to 0.080
-++	Bearing shells - big ends	0.035 to 0.075
IVECO	Main bearing shells	0.127 - 0.254 - 0.508
PRATES A	Big end bearing shells	0.127 - 0.254 - 0.508
	Main journal, thrust bearing XI	39.96 to 40.00
X2	Main bearing housing, thrust bearing X2	32.94 to 32.99
× 3	Thrust washer halves X3	3.38 to 3.43
	Driving shaft shoulder	0.11 to 0.30
	Alignment	≤ 0.05
	Ovality I - 2	0.010
	Taper I - 2	0.010
• Fitted in production	n only and not supplied as spa	res

	Туре	F2B
CYLINDER HEAD	S - VALVE TRAIN	mm
	Valve guide housings in cylinder head Ø1	12.980 to 12.997
	Ø 2 Valve guide 🚣 Ø 3	8.023 to 8.038 3.0 2 to 3.025
- cS	Valve guides - housings in the cylinder heads	0.015 to 0.045
	Valve guide	_
	Valves: $\swarrow 1 \qquad $	7.985 to 8.000 60° 30′ ± 7′ 30″ 7.985 to 8.000 45° + 15′
	Valve stem and its guide	0.023 to 0.053
	Housing in head for valve seat \swarrow Ø1 \swarrow Ø1	41.985 to 42.020 40.985 to 41.020
Ø 2	Outside diameter of valve seat; angle of valve seat in cylinder head: $\swarrow 2$ α	42.060 to 42.075 60° - 30'
- α	$ \begin{array}{c} $	41.060 to 41.075 45° - 30′
×	Recessing of valve X	0.5 to 0.8 1.6 to 1.9
-\$ ⁻	Between valve seat and head	0.040 to 0.090

	Туре	F2B
		mm
	Valve outside spring	
	height: free height H	62.6
	under a load of: 2 N 454 ± 22 HI	48.5
	N 840 ± 42 H2	36.5
×	Injector protrusion X	0.7
	Camshaft bush housing fitted in the cylinder head: I \Rightarrow 7 Ø	80.000 to 80.030
	Camshaft journal diameter: I ⇒ 7 Ø	75.924 to 75.940
Ø	Camshaft bushing outer diameter: Ø	80.090 to 80.115
	Camshaft bushing inner diameter: Ø	75.990 to 76.045
	Bushings and housings in engine block	0.060 to 0.115
	Bushings and journals	0.050 to 0.121
	Cam lift: ⊏次	8.07
Н		7.63
		8.80 to 8.82
	Rocker shaft Ø I	37.984 to 38.000

	Туре	F2B
		mm
	Bushing housing in rocker arms	
		41.000 to 41.016
		53.000 to 53.019
Ø		42.000 to 42.016
	Bushing outer diameter for rocker arms:	
¥		41.097 to 41.135
Ø		53.105 to 53.156
*		42.066 to 42.091
	Bushing inner diameter for rocker arms:	
•		38.025 to 38.041
Ø		50.025 to 50.041
T		38.015 to 38.071
	Between bushings and housings	
		0.081 to 0.135
5		0.086 to 0.156
		0.050 to 0.091
	Between rocker arms and shaft	
		0.025 to 0.057
		0.225 to 0.057
		0.015 to 0.087
	DCHARGER	
Type End float		HOLSET, variable geometry
Radial play		_

TIGHTENING TORQUES

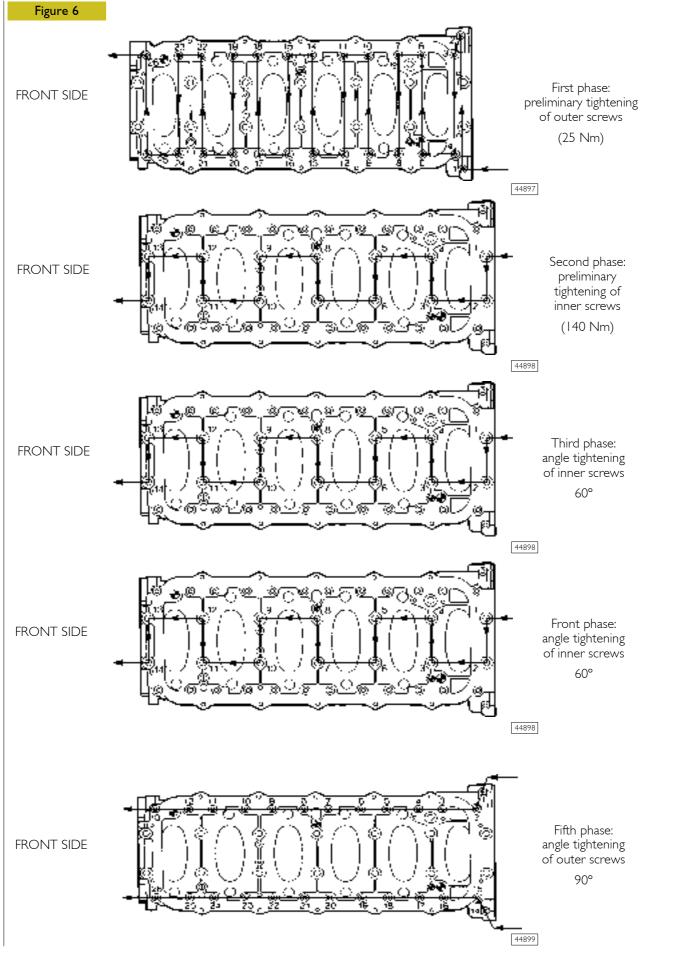
PART		TORQUE	
		Nm	kgm
Under-basement fastening screws to cylinder block (see Figure 6) $ig lpha$			
Duter screws First stage : pre-tightening	M10x1.25	25	2.5
nner screws Second stage : pre-tightening	MI6x2	140	14
nner screws Third stage : angle closing	MI6x2		0°
nner screws Fourth stage : angle closing	MI6x2		0°
Outer screws Fifth stage : angle closing	M10x1,5		°C
Pipe union for piston cooling nozzle	M12X1.5	35 ± 2	3.5 ± 0.2
ntercooler fastening screws to cylinder block \blacklozenge (see Figure 10)			
pre-tightening		11.5 ± 3.5	1.15 ± 0.35
tightening		19 ± 3	1.9 ± 0.3
Plug		125 ± 15	12.5 ± 1.5
Spacer and oil sump fastening screws (see Figure 11)		41.5 ± 3.5	4.1 ± 0.3
Gearcase fastening screws to cylinder block		41.5 ± 3.5	4.1 ± 0.3
6 /		63 ± 7	6.3 ± 0.7
		9 ± 3	1.9 ± 0.3
Cylinder head fastening screw: (see Figure 7) ♦		/	0.0
		50	5
First stage pre-tightening Second stage pre-tightening		100	5
Third stage angle closing			0°
Fourth stage angle closing			5°
Rocker arm shaft fastening screw \blacklozenge		1.	J
First stage pre-tightening		40	4
Second stage pre-tightening)°
Locknut for rocker arm adjusting screw ♦		39 ± 5	3.9 ± 5
Screws for injector fastening brackets \blacklozenge		36.5	3.65
Shoulder plate fastening screws to head \blacklozenge		23.5	2.35
Engine support bracket fastening screws to cylinder head		74 ± 8	7.4 ± 0.8
Gear fastening screws to camshaft: ♦			
First stage pre-tightening		50	5
Second stage pre-tightening			0°
Phonic wheel fastening screws to distribution gear		8.5 ± 1.5	0.8 ± 0.1
xhaust pipe fastening screws • (see Figure 8)			
pre-tightening		32.5 ± 7.5	3.2 ± 0.7
tightening		47 ± 2.5	4.7 ± 0.2
ngine brake actuator cylinder fastening screws		24.5 ± 2.5	2.4 ± 0.2
Connecting rod cap fastening screws: ♦			
irst stage pre-tightening		50	5
Second stage pre-tightening		4(0°
Engine flywheel fastening screws: ♦	M16x1.5x58		
-irst stage pre-tightening		100	10
Second stage pre-tightening		61	°C
Engine flywheel fastening screws: ♦	MI6x1.5x110		
First stage pre-tightening		100	10
Second stage pre-tightening		12	0°
lywheel pulley fastening screws to crankshaft : ♦			
First stage pre-tightening		70	7
		5	0°
Second stage pre-tightening		. 1	0

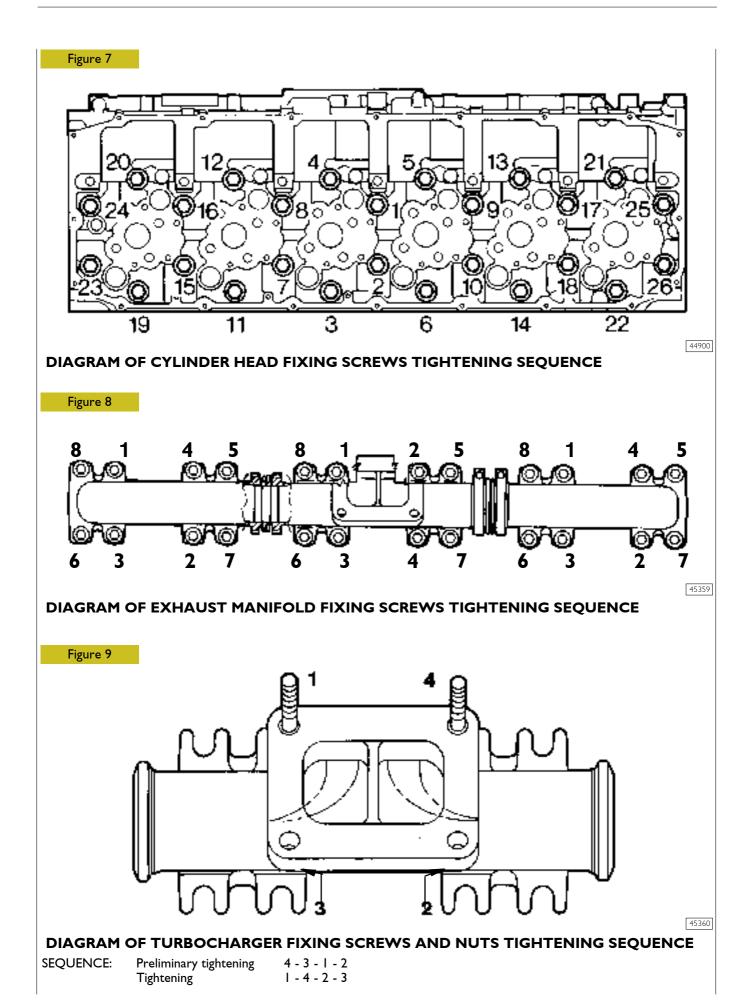
TIGHTENING TORQUES

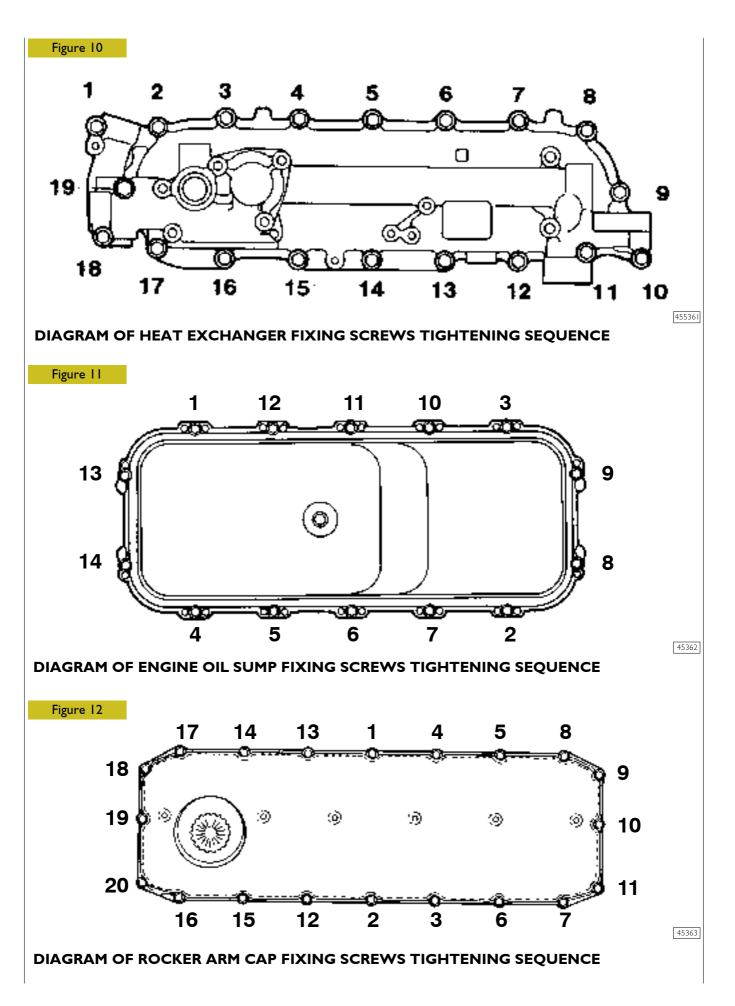
PART		TORQUE		
		Nm	kgm	
Damper flywheel fastening	screws: ♦	5 ± 5	.5 ± .5	
Idler gear pin fastening scre	ews: ♦			
First stage	pre-tightening	30	3	
Second stage	pre-tightening	9	0°	
Idle gear link rod fastening	screw	24.5 ± 2.5	2.4 ± 0.2	
Oil pump fastening screw		24.5 ± 2.5	2.4 ± 0.2	
Oil pump suction rose fast	ening screw	24.5 ± 2.5	2.4 ± 0.2	
Front cover fastening screv	v to cylinder block 🔶	19 ± 3	1.9 ± 0.3	
Control unit fastening scre	w to cylinder block ♦	19 ± 3	1.9 ± 0.3	
Supply pump fastening scre	ew to gearcase ♦	19 ± 3	1.9 ± 0.3	
Fuel filter support fastening	screw to cylinder head ♦	37 ± 3	3.7 ± 0.3	
Turbo-compressor fastenir pre-tightening tightening	ng screws and nuts • (see Figure 9)	32.5 ± 7.5 46 ± 2	3.2 ± 0.7 4.6 ± 0.2	
Water pump fastening scre	ew to cylinder block	24.5 ± 2.5	2.4 ± 0.2	
Pulley fastening screw to h	ub	55 ± 5	5.5 ± 0.5	
Rocker arm cover fastening	g screws (see Figure 12)	9	0.9	
Thermostat box fastening	screws to cylinder head	24.5 ± 2.5	2.4 ± 0.2	
Automatic tightener fasten	ing screws to cylinder block	45 ± 5	4.5 ± 0.5	
Fixed tightener fastening so	rews to cylinder block	105 ± 5	10.5 ± 0.5	
Fan support fastening screv	vs to cylinder block	24.5 ± 2.5	2.4 ± 0.2	
Starter fastening screws		44 ± 4	4 ± 0.4	
Air heater on cylinder hea	d	30 ± 3	3 ± 0.3	
Air compressor fastening s	crew to cylinder head	74 ± 8	7.4 ± 0.8	
Air compressor control ge	ar fastening nut	170	7 ±	
Hydraulic power steering p	oump gear fastening nut	46.5 ± 4.5	4.6 ± 0.4	
Air conditioner compresso	r fastening screw to support	24.5 ± 2.5	2.4 ± 2.5	
Air conditioner compresso	r support fastening screw to cylinder block	44 ± 4	4.4 ± 0.4	
Alternator support fastenir	ng screw to cylinder block	44 ± 4	4.4 ± 0.4	
Alternator bracket fastenir	g screw to cylinder block	24.5 ± 2.5	2.4 ± 0.2	
Water pipe unions		35	3.5	
Water temperature sensor		32.5 ± 2.5	3.2 ± 0.2	

TIGHTENING TORQUES

PART	TORQUE		
	Nm	kgm	
Engine brake solenoid valve fastening screws	32.5 ± 2.5	3.2 ± 0.2	
Flywheel rev sensor fastening screw	8 ± 4	0.8 ± 0.2	
Camshaft rev sensor fastening screw	8 ± 2	0.8 ± 0.2	
P.D.E solenoid connector fastening screw	1.62 ± 0.3	0.1 ± 0.3	
Overboost pressure sensor fastening screw	8 ± 2	0.8 ± 0.2	
Absolute pressure sensor fastening screw	22.5 ± 2.5	2.2 ± 0.2	
P.W.M. control valve fastening screw/nut	8 ± 2	0.8 ± 0.2	
Fuel/coolant temperature sensor	35	3.5	
Coolant temperature indicator	23.5 ± 2.5	2.3 ± 0.2	
Filter clogging sensor	10	ļ	
Oil temperature switch	25 ± 1	2.5 ± 0.1	
Oil pressure sensor	25 ± 1	2.5 ± 0.1	
Oil clogging sensor	55 ± 5	5.5 ± 0.5	
Electric wire fastening screw	8 ± 2	0.8 ± 0.2	
Heater fastening screw	12.5 ± 2.5	1.2 ± 0.2	



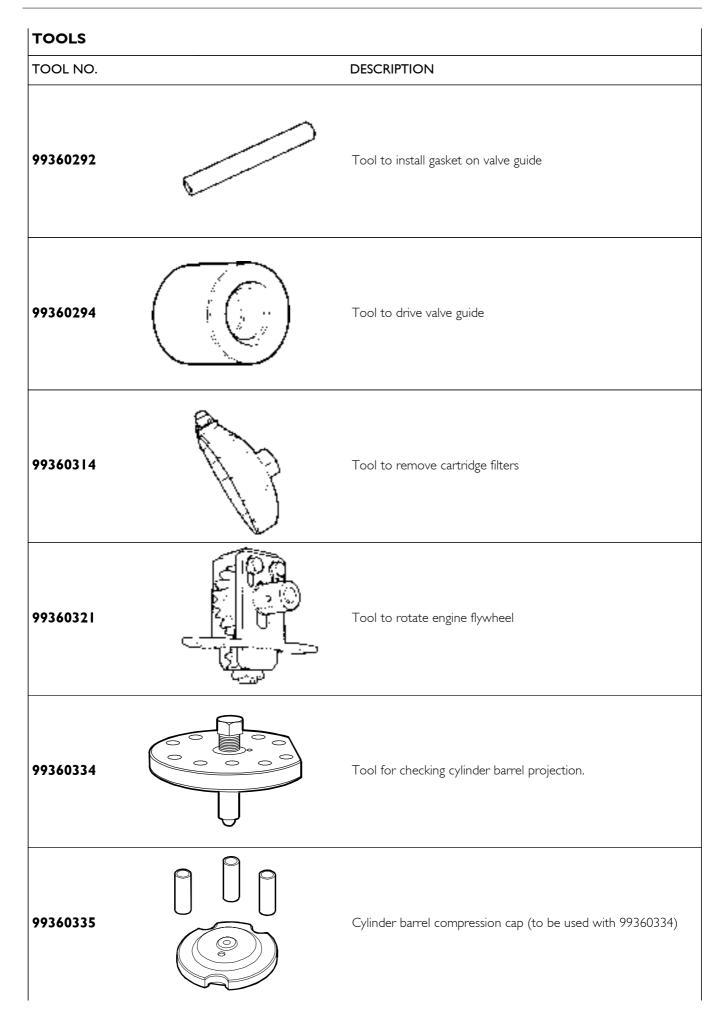




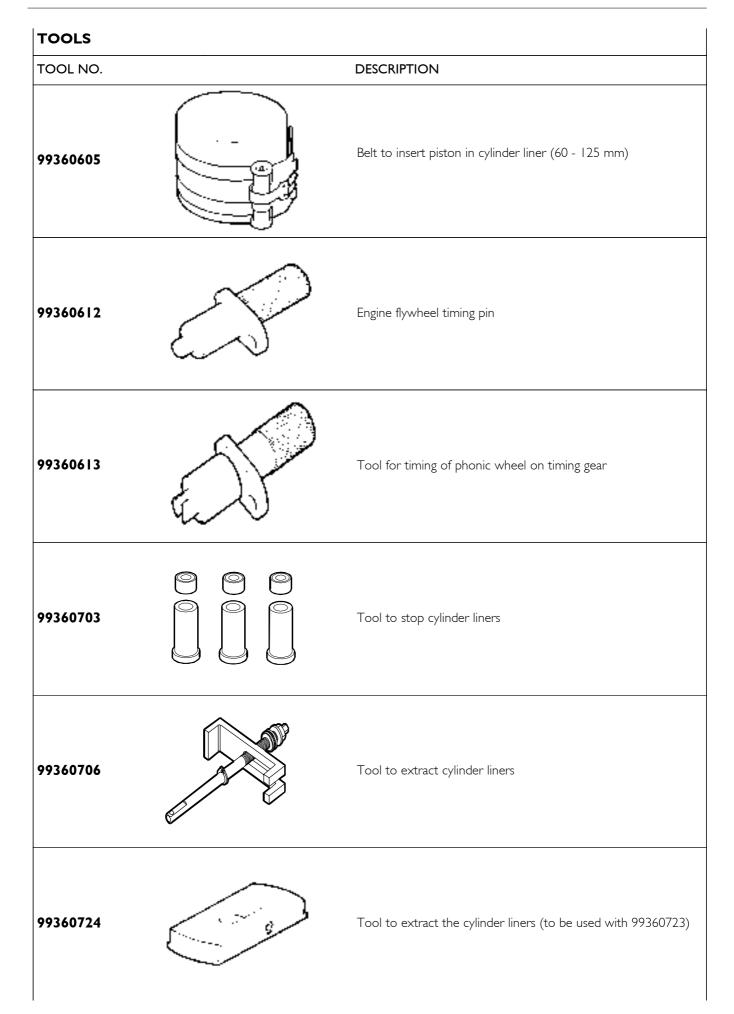
TOOLS TOOL NO. DESCRIPTION Full-optional tool-kit to rectify valve seat 99305019 99305047 Equipment for spring load check 99322230 Rotary telescopic stand 9934005 I Extractor for crankshaft front gasket 99340052 Extractor for crankshaft rear gasket 5 99340205 Percussion extractor

TOOLS		
TOOL NO.		DESCRIPTION
99342148		Injector extractor
99342149		Extractor for injector-holder
99346245	Co P	Tool to install the crankshaft front gasket
99346246		Tool to install the crankshaft rear gasket
99348004		Universal extractor for 5 to 70 mm internal components
99350072		Box wrench for transmission gear support fixing screws

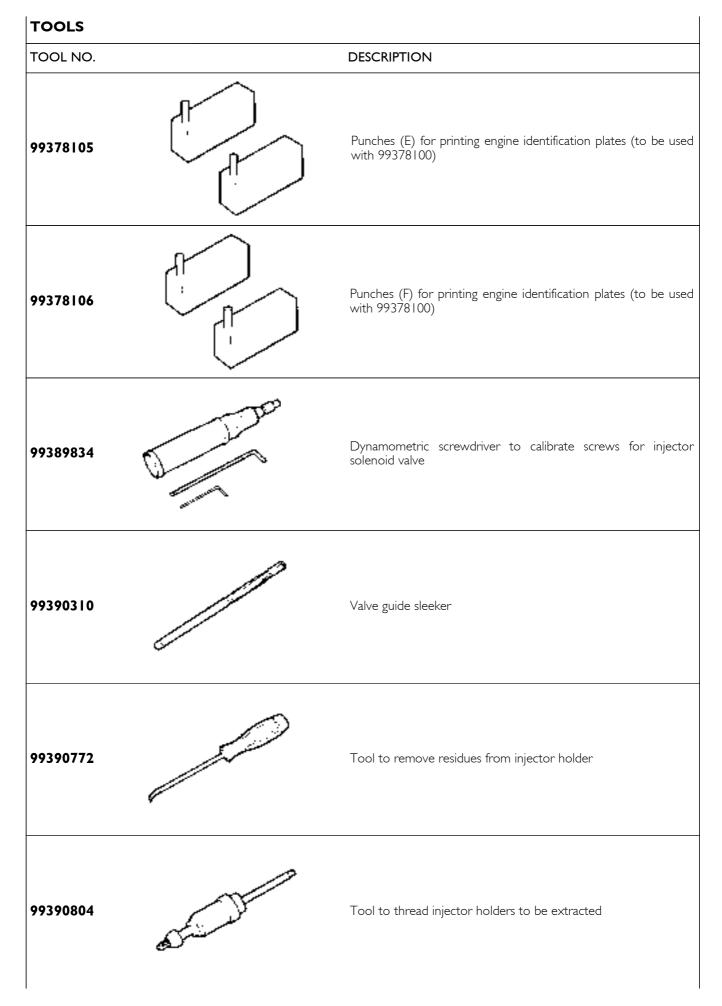
TOOLS TOOL NO. DESCRIPTION 99350074 Box wrench for block junction bolts to the underblock A Skid retaining tools (12+6) for rocker arm adjusting screws during 99360144 rocker arm shaft removal/ refitting 99360177 Injector housing plug Pincers for removing and refitting circlips and pistons 99360184 (105-160 mm) 99360264 Tool to take down-fit engine valves 99360288 Tool to remove valve guide



TOOLS TOOL NO. DESCRIPTION 99360351 Tool to stop engine flywheel Tool to take down and fit back camshaft bushes 99360487 ŝ 99360500 Tool to lift crankshaft 99360551 Bracket to take down and fit engine flywheel 99360558 Tool to lift and transport rocker shaft 99360585 Balance for lifting and handling engine

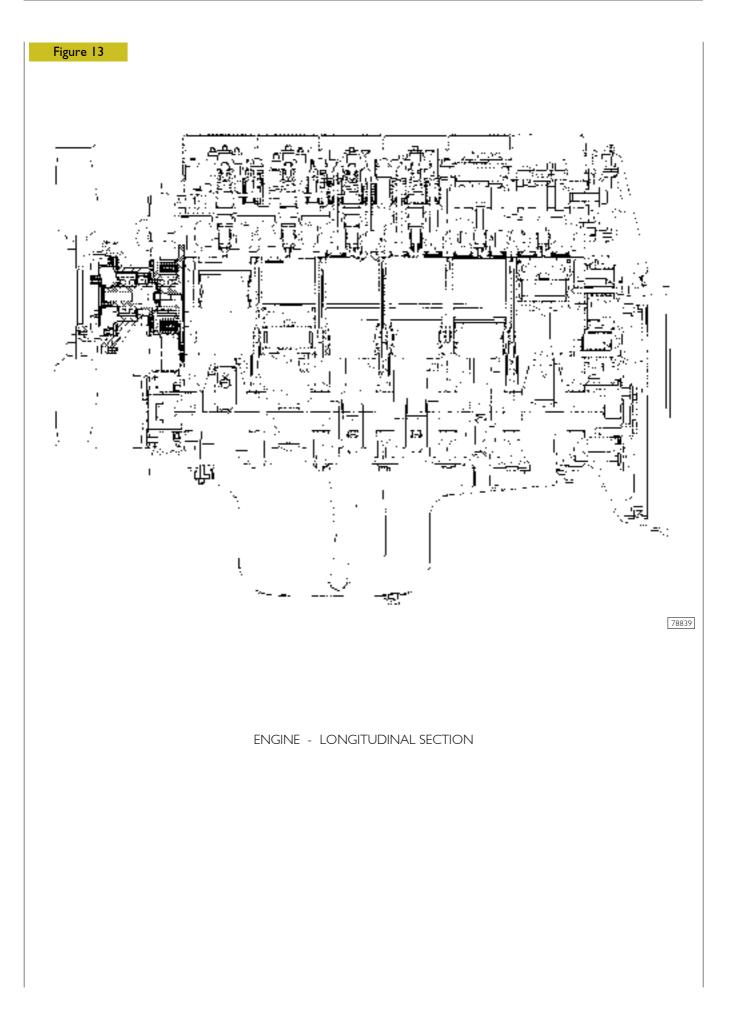


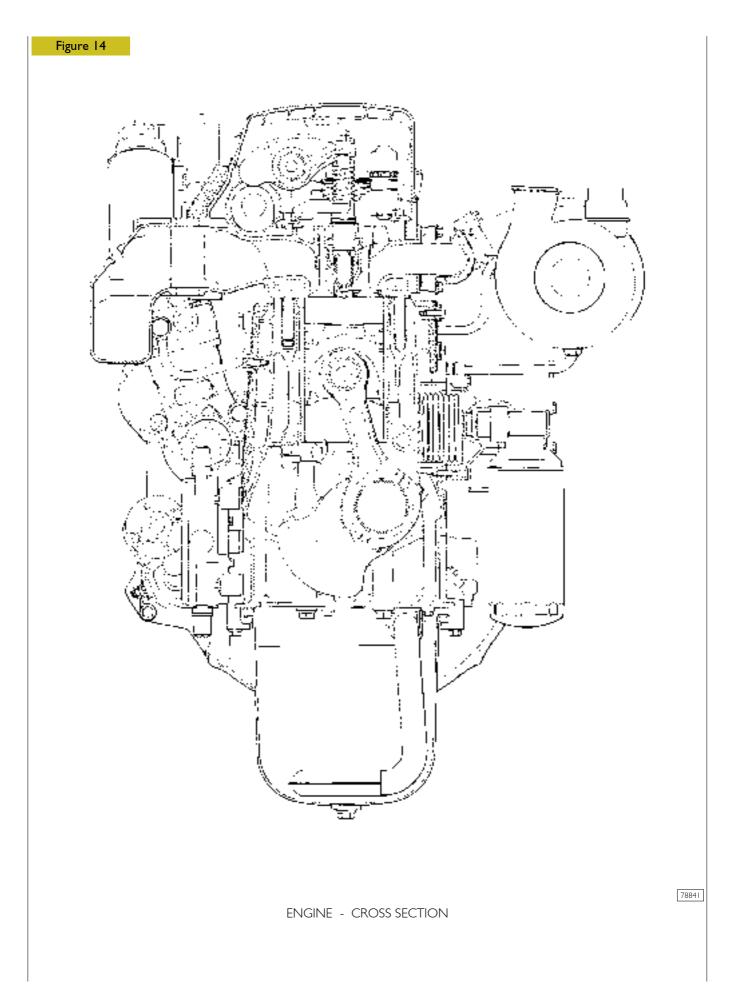
TOOLS TOOL NO. DESCRIPTION 99361035 Brackets fixing the engine to rotary stand 99322230 and the second 99365054 Tool for injector holder heading Tool to detect cylinder liner projections (use with 99395603) 99370415 Tool for printing engine identification plates (to be used with 99378100 special punches) Punches (A) for printing engine identification plates (to be used 99378101 with 99378100) Punches (B) for printing engine identification plates (to be used 99378102 with 99378100)

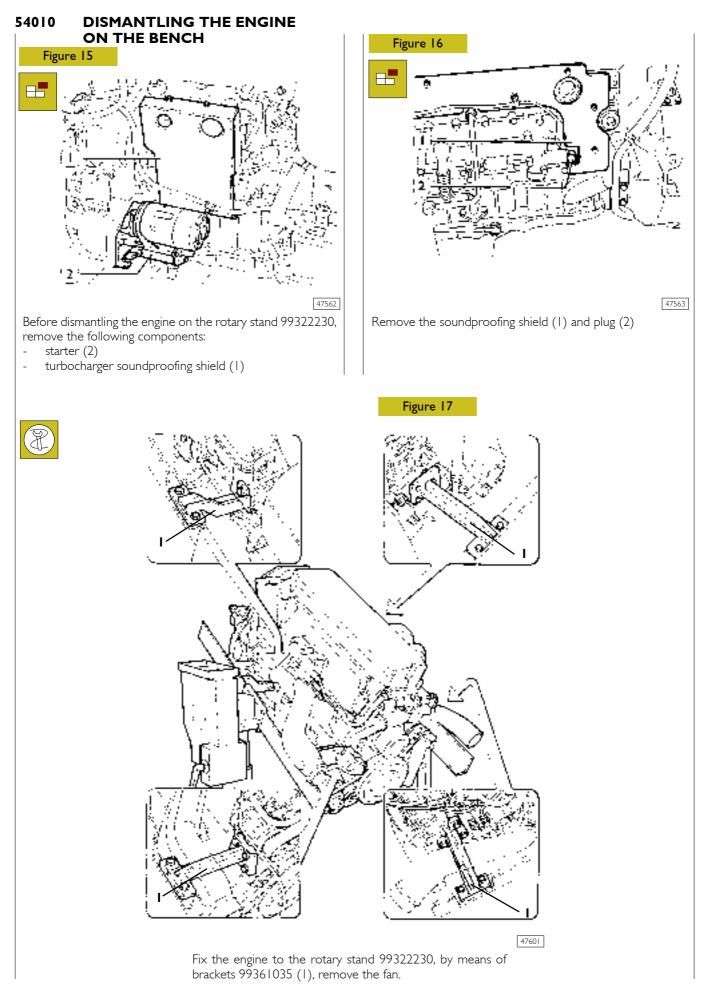


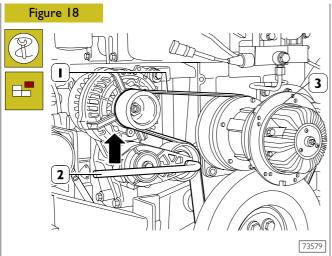
TOOLS		
TOOL NO.		DESCRIPTION
99394014		Guide bush (to be used with 99394041 or 99394043)
99394041		Cutter to rectify injector holder housing (to be used with 99394015)
99394043	en S	Reamer to rectify injector holder lower side (to be used with 99394015)
99395215	0	Gauge for centre distance check between camshaft and idle gear
99395216	90	Measuring pair for angular tightening with 1/2" and 3/4" square couplings
99395363		Complete square to check connecting rod squaring

TOOLS	
TOOL NO.	DESCRIPTION
99395603	Dial gauge (0 - 5 mm)
99395687	Reaming gauge (50-178 mm)
99396033	Centering ring of crankshaft front cap



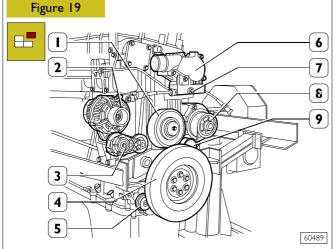






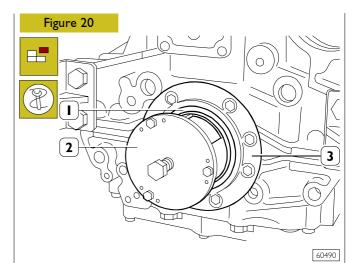
Using an appropriate tool (2), operate in the direction of the arrow, and remove the belt (1) driving the water pump, alternator and fan.

Take out the screws and remove the electromagnetic coupling (3).

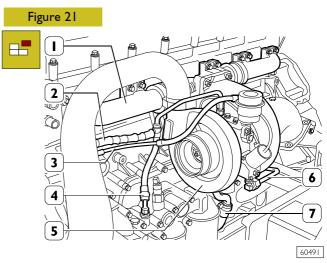


Remove the following components:

- thermostat unit (6) fitted with turbine actuator pressure sensor (7);
- alternator (2);
- pulley support (1);
- water pump (8) and piping;
- automatic belt tightener support (3);
- fixed belt tightener (9);
- damping flywheel (4) and pulley underneath it;
- automatic belt tightener (5);
- disconnect all the electric connections and the sensors.

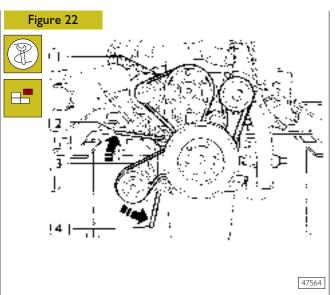


Fit the extractor 99340053 (2) and remove the engine crankshaft seal gasket (1), remove the cover (3).

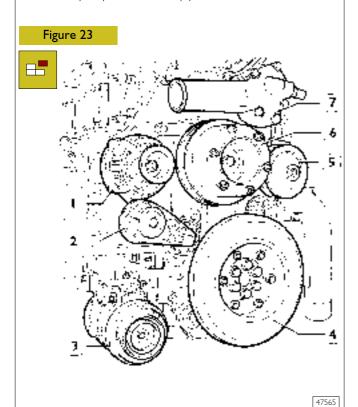


Remove the following components:

- water outlet line (2);
- oil delivery line (4);
- actuator air line (3);
- water delivery line (6);
- oil return line (7);
- turbocharger (5);
- exhaust manifold (1).

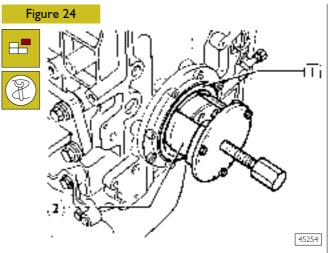


Load the belt tensioner spring by tool (4), acting in the direction shown by the arrow, on the head of the screw fixing the roller. The screw cannot be untightened as the thread is counterclock-wise. Remove the belt (3). By tool (2), act in the direction shown by the arrow and remove the fan, alternator and water pump control belt (1).

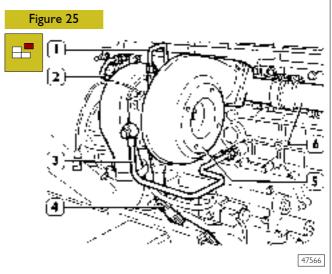


Remove the following components:

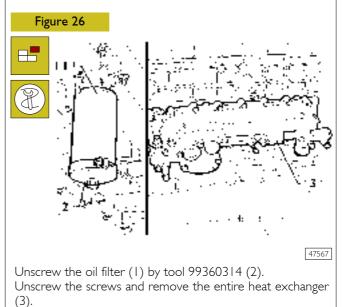
- alternator (1);
- belt tightener support (2);
- air conditioner compressor (3);
- flywheell (4);
- water pump and piping (5);
- fan pulley spacer (6);
- thermostat unit (7).

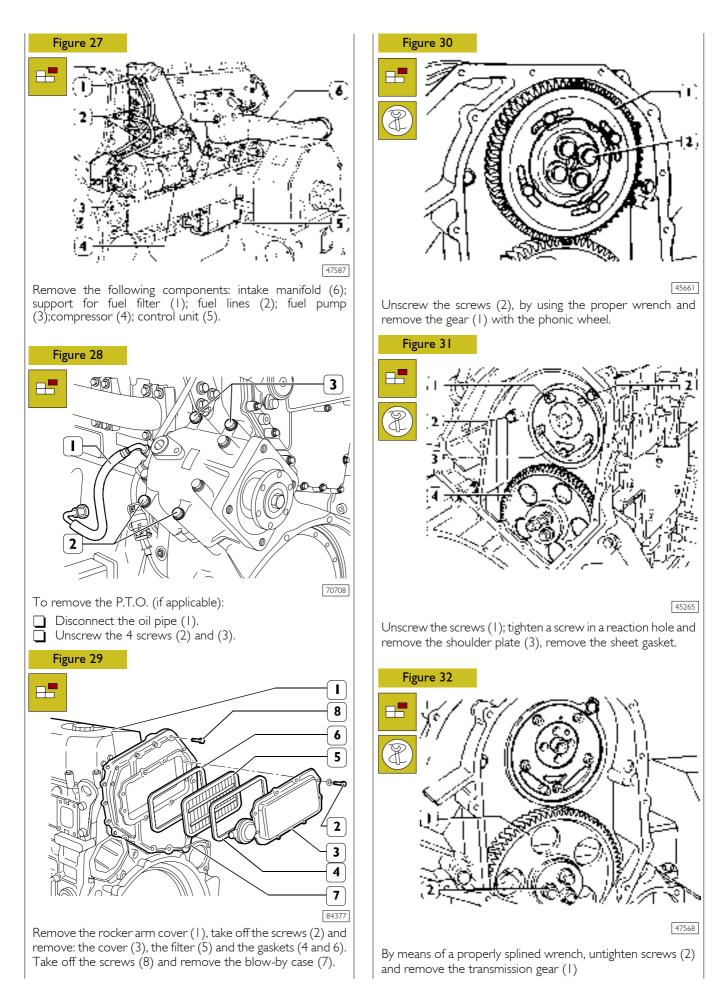


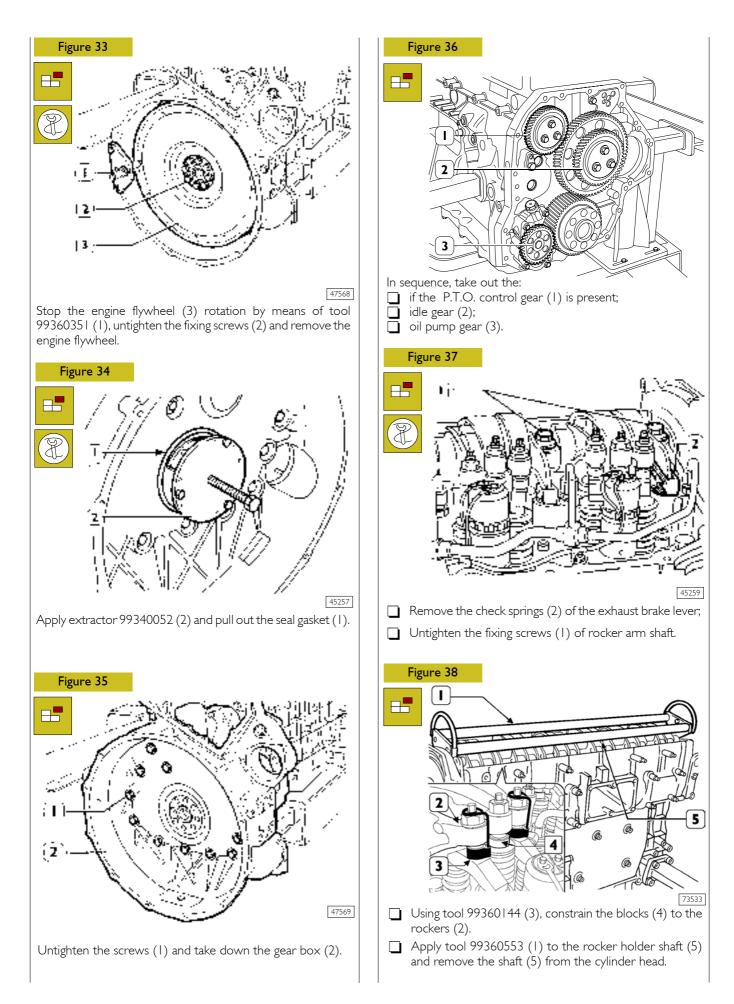
Install extractor 99340051 (2) and remove the seal gaskets (1). Unscrew the screws and remove the cover. Disconnect all electric connections and sensors.

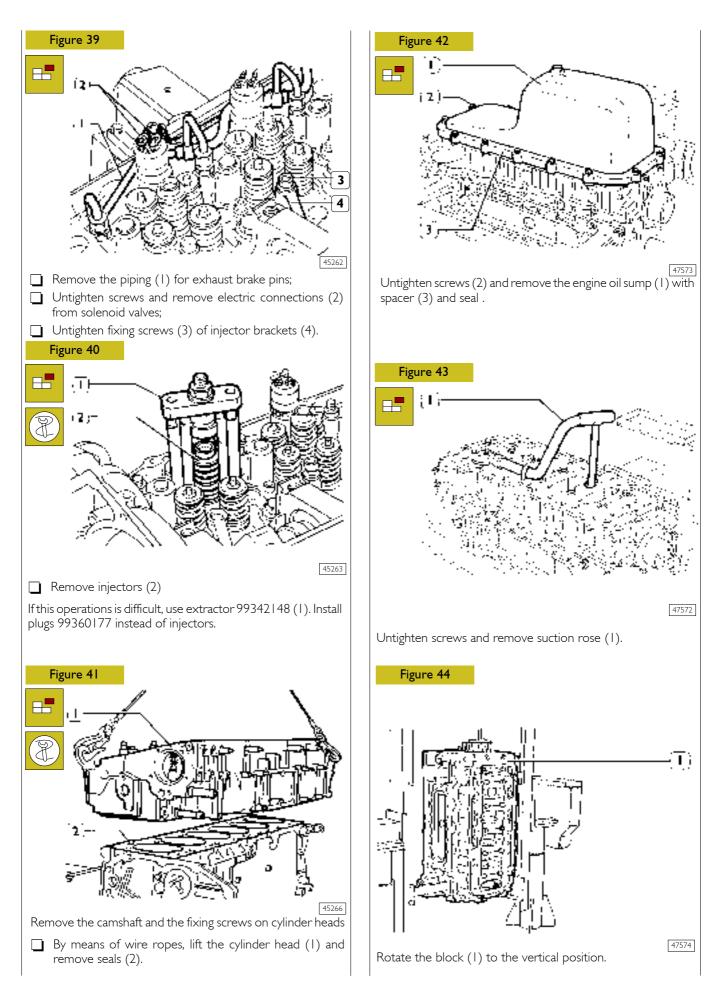


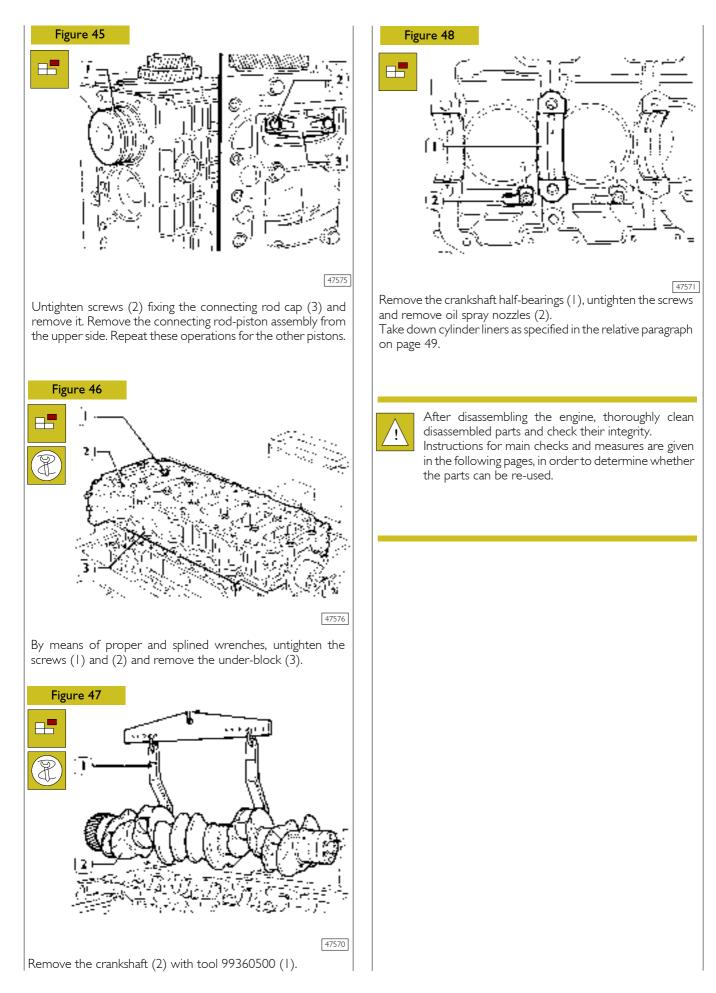
Remove the following components: oil supply lines (1); water cooling supply lines (3); water discharge lines (2); oil return lines (4); turbocharger (5); exhaust manifold (6).



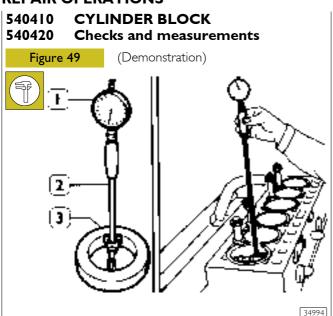








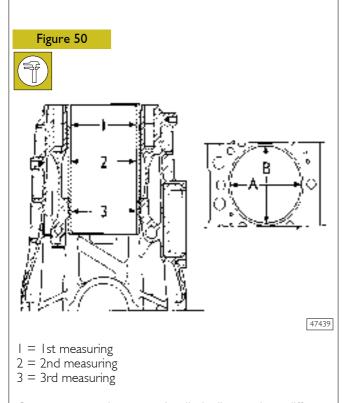
REPAIR OPERATIONS



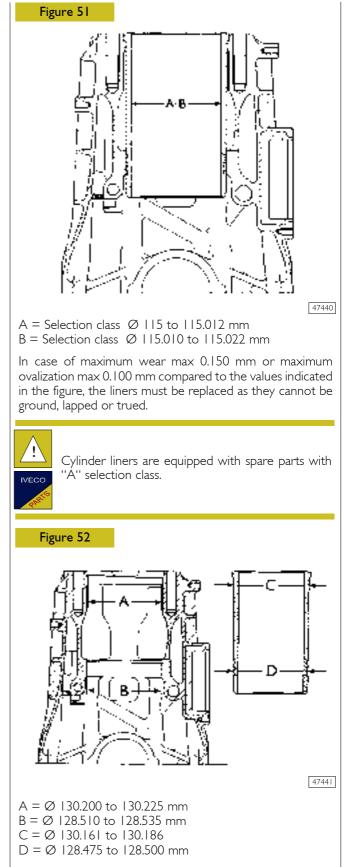
34994

Internal diameter of the cylinder liners is checked for ovalization, taper and wear, using a bore dial (1) centesimal gauge 99395687 (2) previously reset to ring gauge (3), diameter 115 mm.

If a 115 ring gauge is not available use a micrometer caliper.

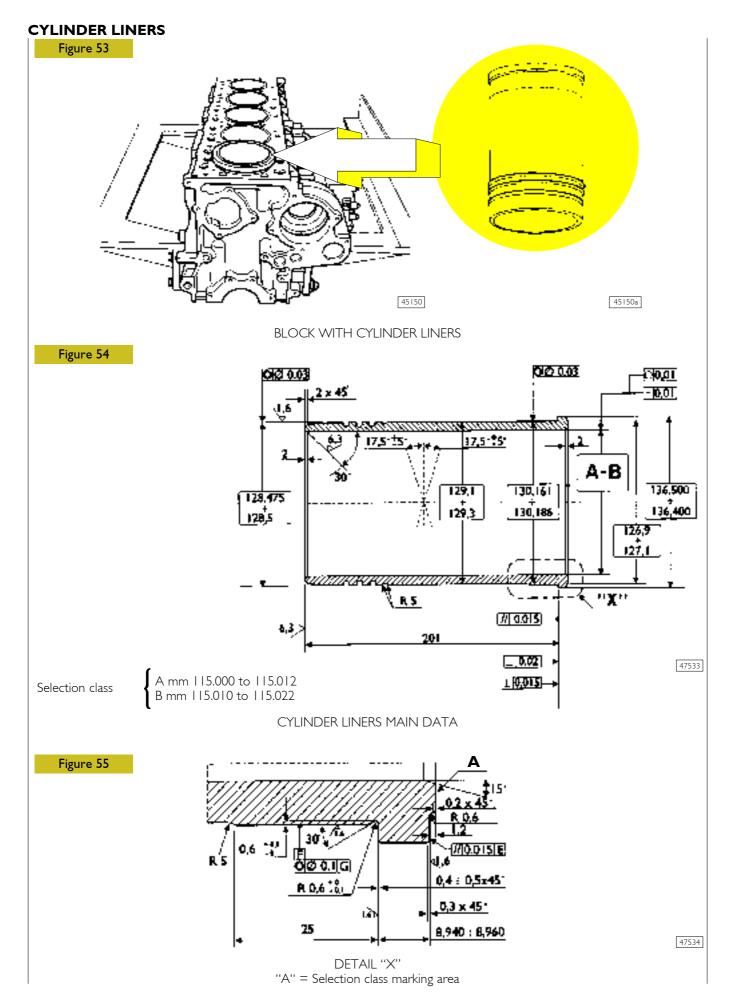


Carry out measurings on each cylinder liner at three different levels and on two (A-B) surfaces, to one another perpendicular, as shown in Figure 50.

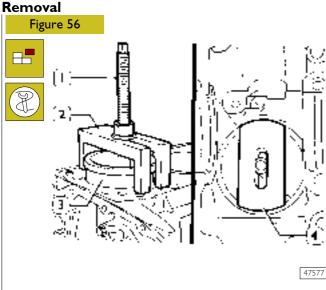


The figure shows the outer diameters of the cylinder liners and the relative seat inner diameters.

The cylinder liners can be extracted and installed several times in different seats, if necessary.



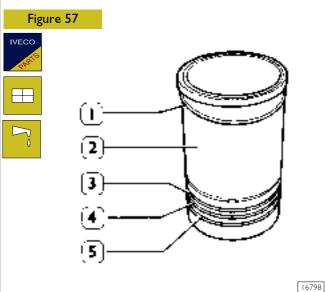
540420 Replacing cylinder liners



Place details 99360706 (I and 2) and plate 99360724 (4) as shown in the figure, by making sure that the plate (4) is properly placed on the cylinder liners.

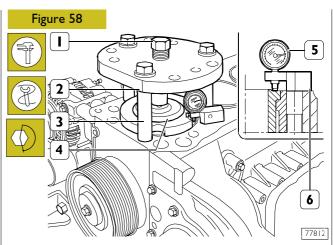
Tighten the screw nut (1) and remove the cylinder liner (3) from the block.

Fitting and checking protrusion



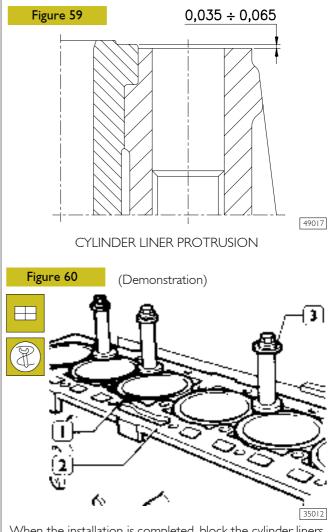
Always replace water sealing rings (3, 4 and 5). Install the adjustment ring (1) on the cylinder liner (2); Iubricate lower part of liner and install it in the cylinder unit using the proper tool.

The adjustment ring (1) is supplied as spare parts in the following thicknesses: 0.08 mm - 0.10 mm - 0.12 mm.



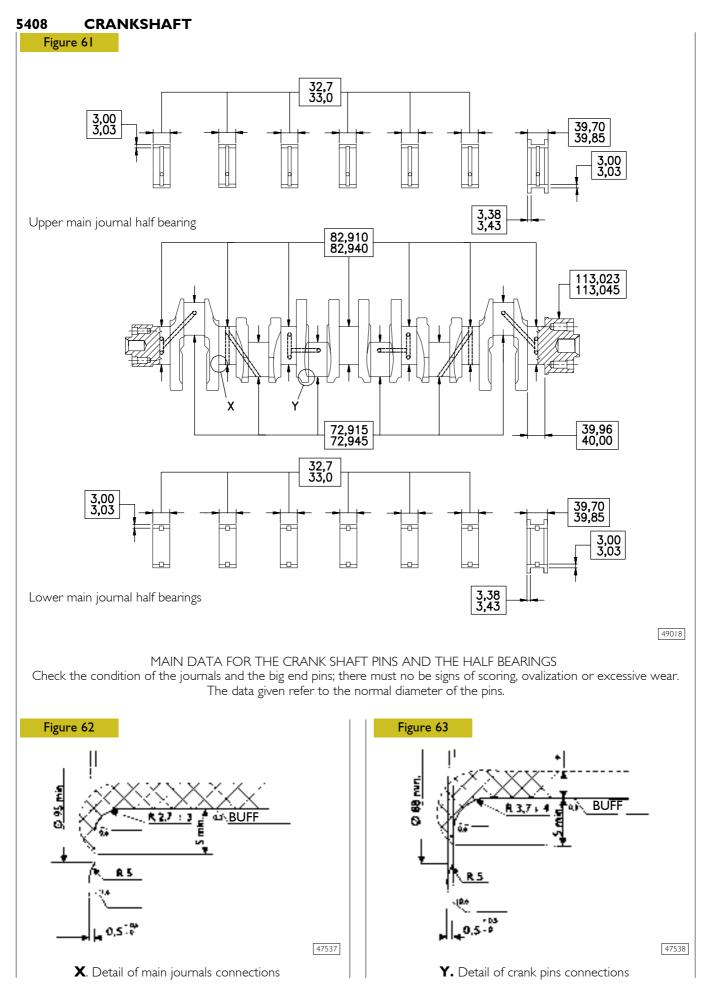
Check cylinder barrel protrusion with tool 99360334 (1-2-3-4) and tighten screw (1) to 170 Nm.

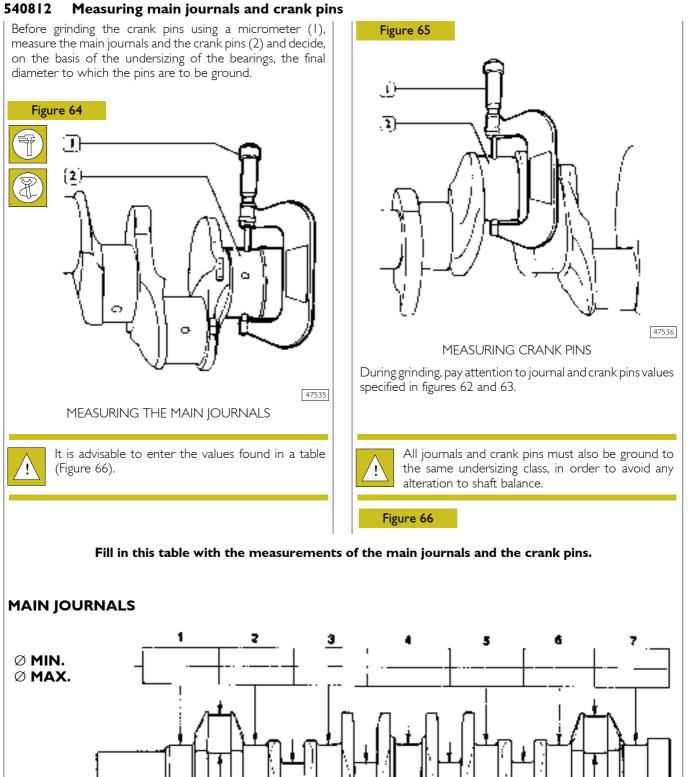
With dial gauge 99395603 (5) placed on base 99370415 (6). Measure the cylinder barrel protrusion compared to the cylinder head supporting plane, it must be 0,035 to 0,065 mm (Figure 59); otherwise replace the adjusting ring (1, Figure 57) fitted with spare parts having different thickness.

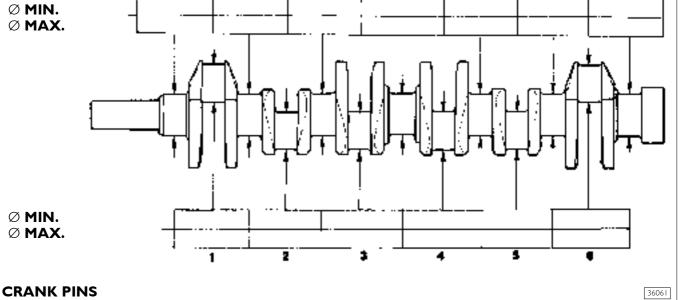


When the installation is completed, block the cylinder liners (1) to the block (2) with studs 99360703 (3).

IVECO







PRELIMINARY MEASUREMENT OF MAIN AND BIG END BEARING SHELL SELECTION DATA

CRANKPINS:

connecting rod.

Determine the class of diameter of the seat in the

Determine the class of diameter of the crankpin.

Select the class of the bearing shells to mount.

For each of the journals of the crankshaft, it is necessary to carry out the following operations:

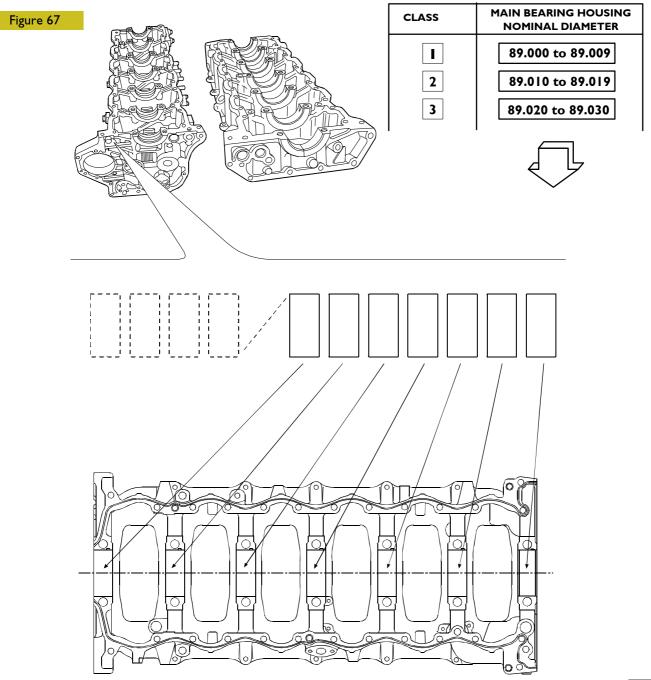
MAIN JOURNALS:

- Determine the class of diameter of the seat in the crankcase.
- Determine the class of diameter of the main journal.
- Select the class of the bearing shells to mount.

DEFINING THE CLASS OF DIAMETER OF THE SEATS FOR BEARING SHELLS ON THE CRANKCASE

On the front of the crankcase, two sets of numbers are marked in the position shown (Figure 67 at top).

- The first set of digits (four) is the coupling number of the crankcase with its base.
- The following seven digits, taken singly, are the class of diameter of each of the seats referred to (Figure 67 at bottom).
- Each of these digits may be 1, 2 or 3.



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Selecting the main and big end bearing shells

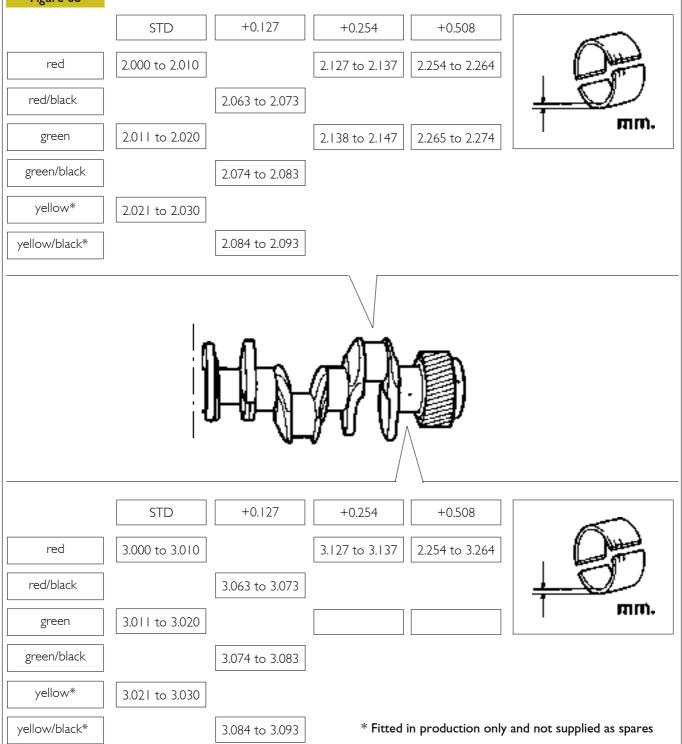


To obtain the required assembly clearances, the main and big end bearing shells need to be selected as described hereunder.

This operation makes it possible to identify the most suitable bearing shells for each of the journals (the bearing shells, if necessary, can have different classes from one journal to another).

Depending on the thickness, the bearing shells are selected in classes of tolerance marked by a coloured sign (red-green – red/black – green/black). The following tables give the specifications of the main and big end bearing shells available as spares in the standard sizes (STD) and in the permissible oversizes (+0.127, +0.254, +0.508).

Figure 68



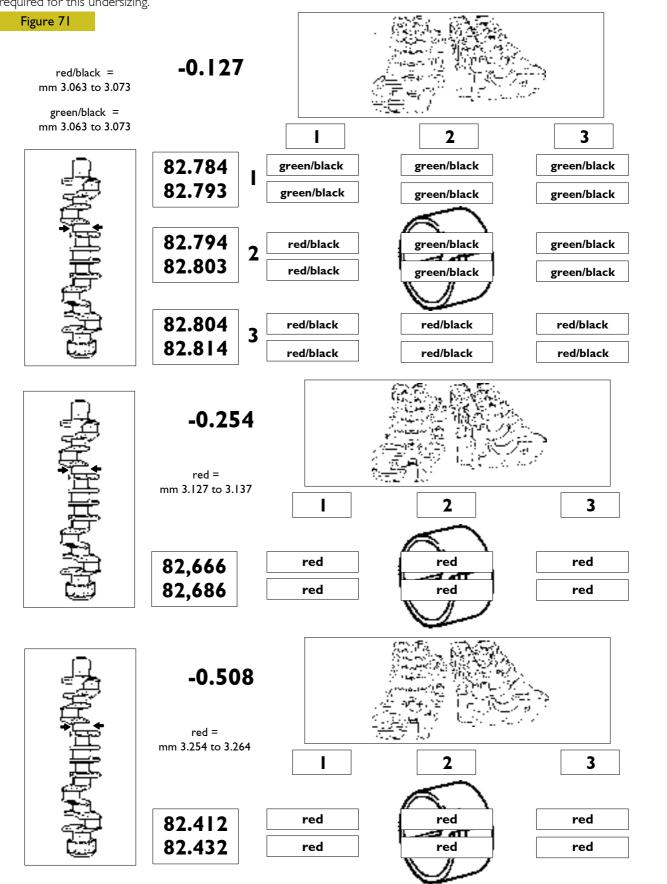
DEFINING THE CLASS OF DIAMETER OF THE MAIN JOURNALS AND CRANKPINS (Journals with nominal diameter) Main journals and crankpins: determining the class of diameter of the journals. Three sets of numbers are marked on the crankshaft in the position shown by the arrow (Figure 69 at top): The first number, of five digits, is the part number of the shaft. Under this number, on the left, a set of six digits refers to the crankpins and is preceded by a single digit showing the status of the journals (I = STD, 2 = -0.127), the other six digits, taken singly, give the class of diameter of each of the crankpins they refer to (Figure 69 at top). The set of seven digits, on the right, refers to the main journals and is preceded by a single digit: the single digit shows the status of the journals (I = STD, 2 = -0.127), the other seven digits, taken singly, give the class of diameter of each of the main journals they refer to (Figure 69 at bottom). Figure 69 CRANKPIN CLASS NOMINAL DIAMETER L 72.915 to 72.924 2 72.925 to 72.934 3 72.935 to 72.945 MAIN JOURNALS CLASS NOMINAL DIAMETER 82.910 to 82.919 I 2 82.920 to 82.929 3 82.930 to 82.940

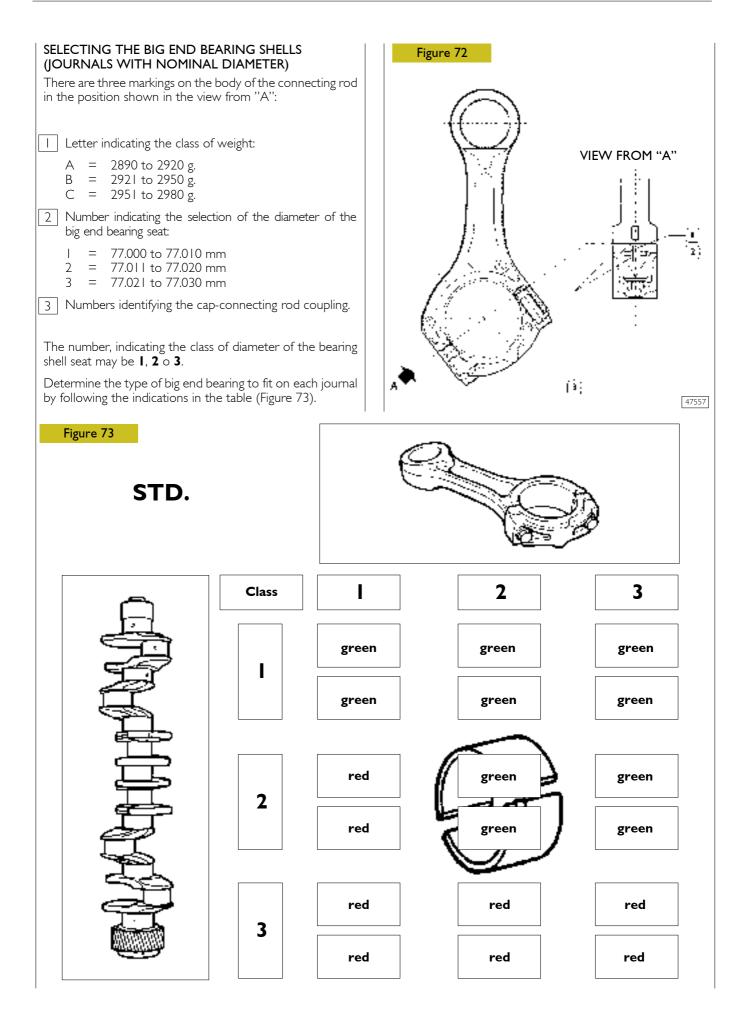
Selection of main half-bearings (nominal diameter pins) After detecting, for each journal, the necessary data on block and crankshaft, select the type of half-bearings to be used, in compliance with the following table: Figure 70 STD. I 2 3 green green green I green green green green green red 2 red green green red red red 3 red red red

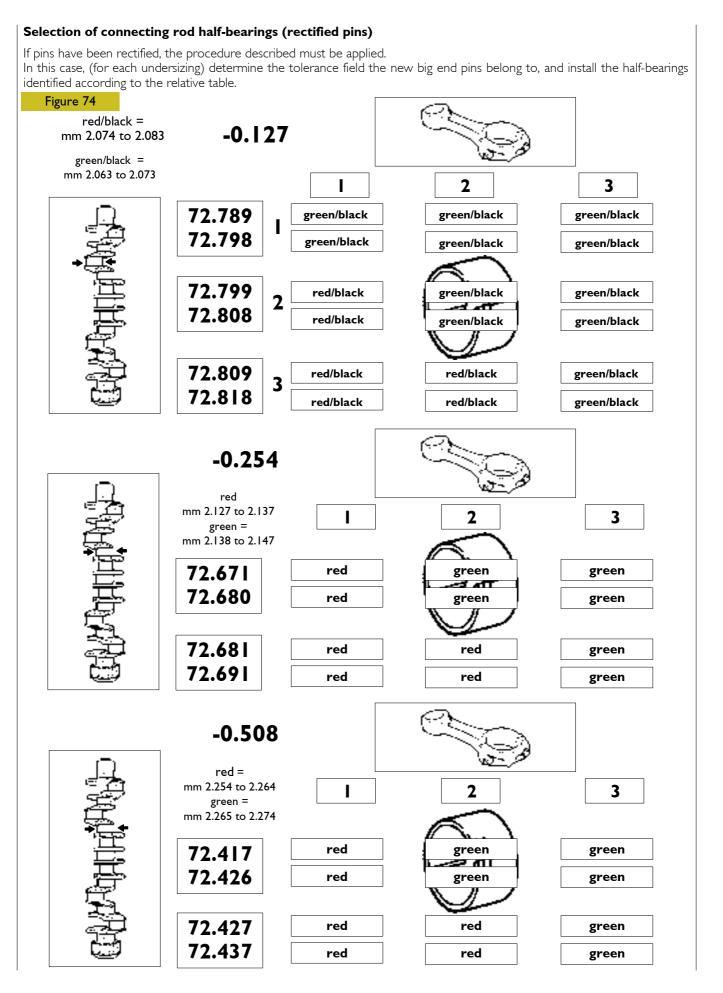
Selection of main half-bearings (rectified pins)

If the journals have been rectified, the procedure described cannot be applied.

In this case, make sure that the new diameter of the journals is as specified on the table and install the only half-bearing type required for this undersizing.







540815 Replacing the timing control gear and the oil pump

Check that the teeth of the gears are not damaged or worn, otherwise remove them using the appropriate extractor.

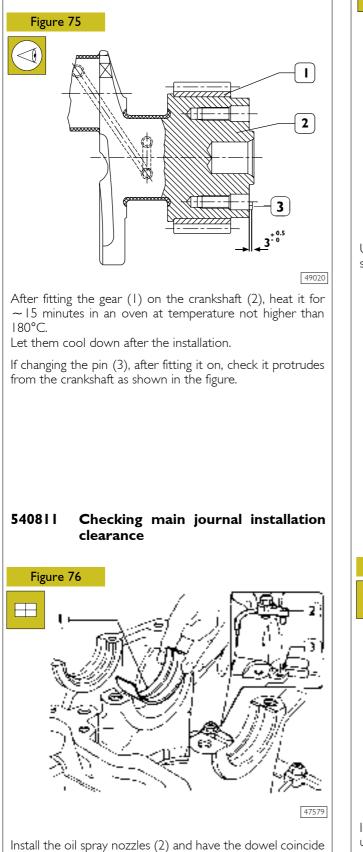
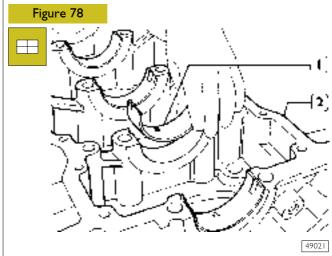


Figure 77

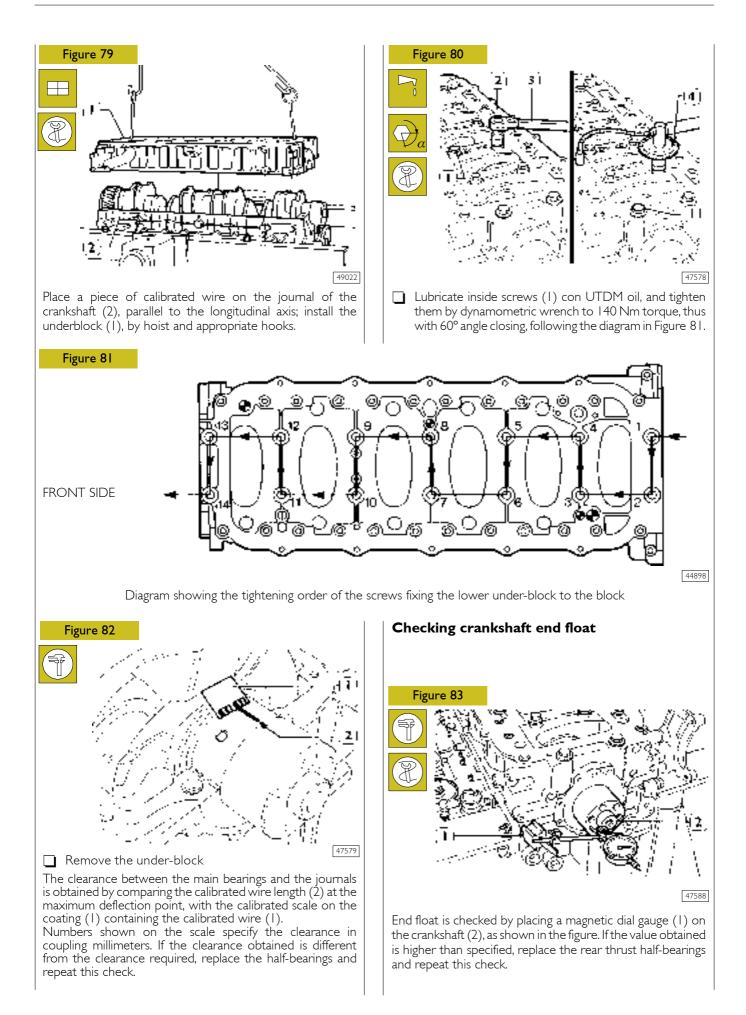
Using the hoist and hook 99360500 (1) mount the driving shaft (2).

Install the oil spray nozzles (2) and have the dowel coincide with the block hole (3). Install the half-bearings (1) on the main bearings.

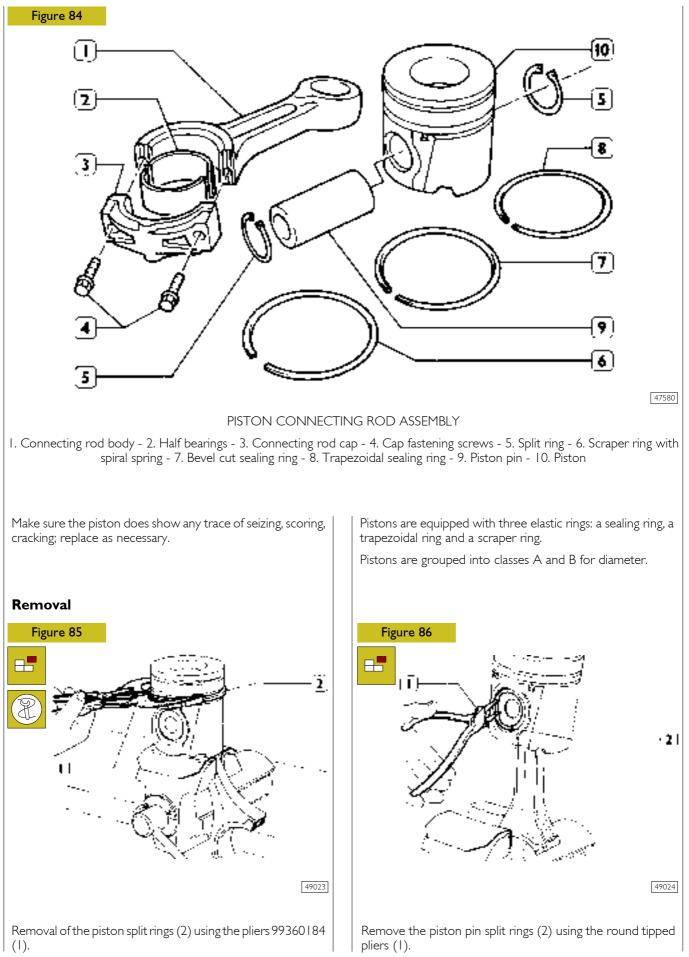


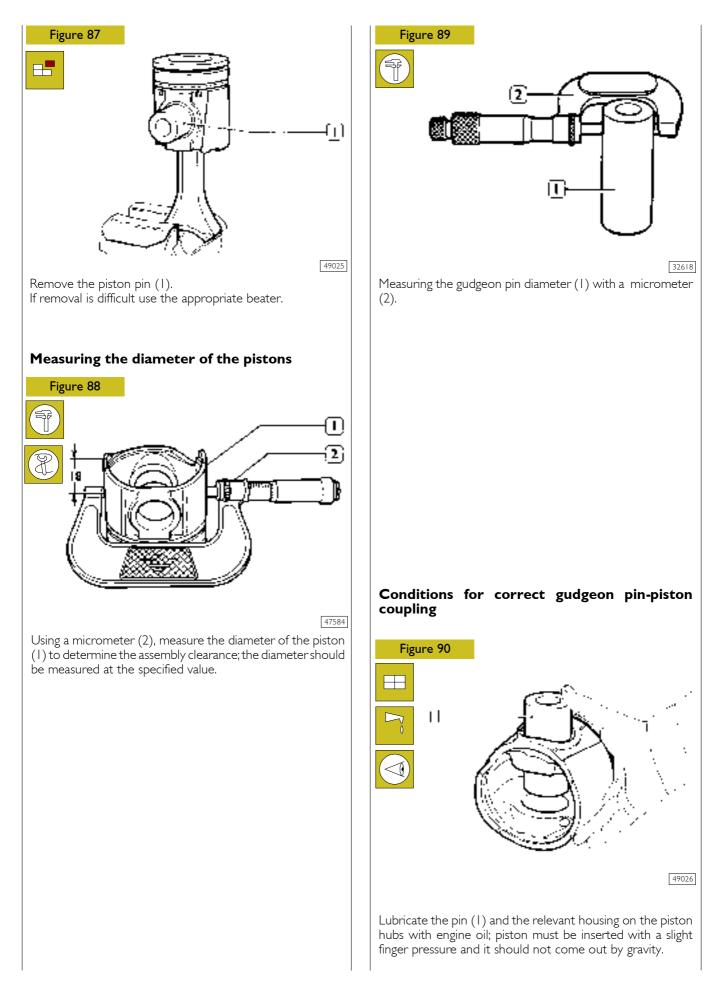
Install the half-bearings (1) on the main bearings in the underblock (2).

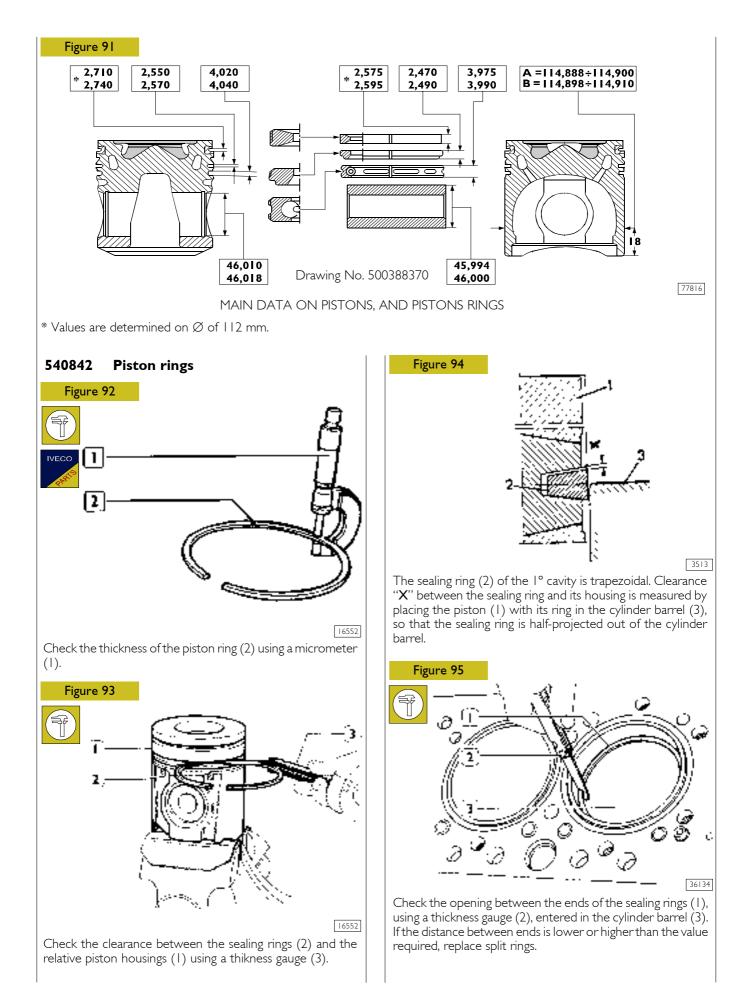
Check the installation clearance between the main journals and the relative bearings as follows:

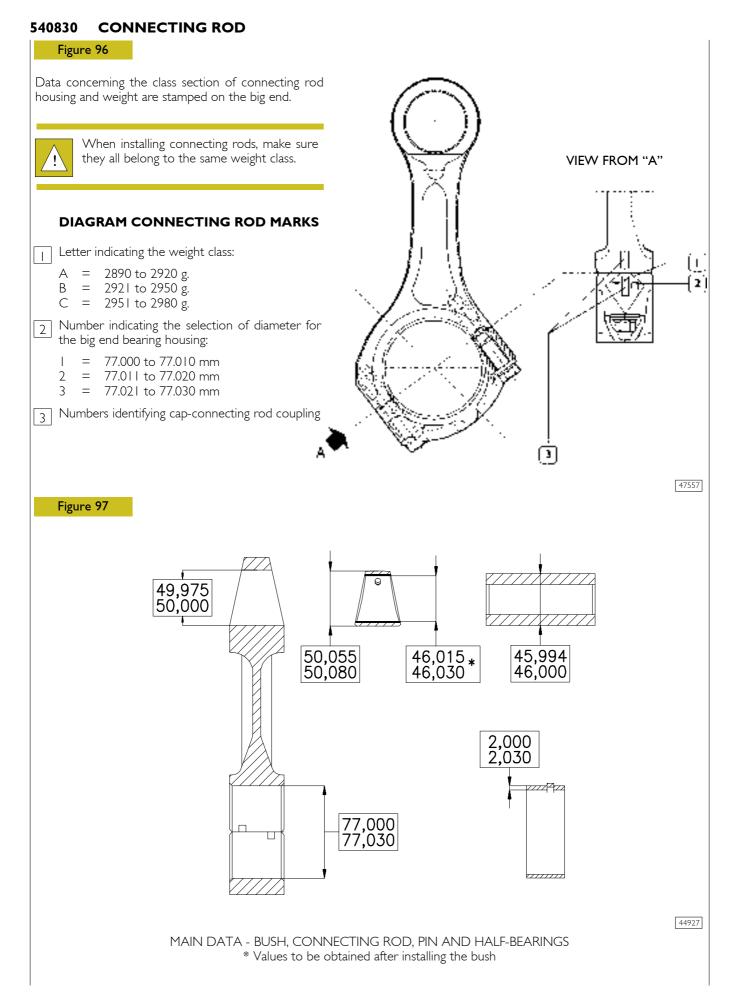




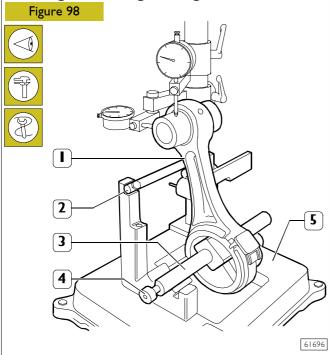










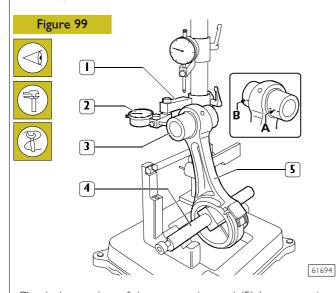


Checking axis alignment

Check the alignment of the axes of the connecting rods (1) with device 99395363 (5), proceeding as follows:

- Fit the connecting rod (1) on the spindle of the tool 99395363 (5) and lock it with the screw (4).
- Set the spindle (3) on the V-prisms, resting the connecting rod (1) on the stop bar (2).

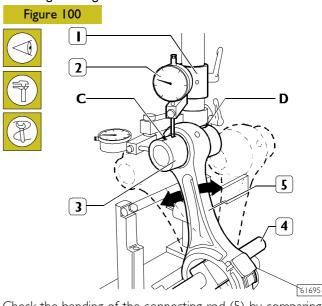
Checking torsion



Check the torsion of the connecting rod (5) by comparing two points (**A** and **B**) of the pin (3) on the horizontal plane of the axis of the connecting rod.

Position the mount (1) of the dial gauge (2) so that this pre-loads by approx. 0.5 mm on the pin (3) at point **A** and zero the dial gauge (2). Shift the spindle (4) with the connecting rod (5) and compare any deviation on the opposite side **B** of the pin (3): the difference between **A** and **B** must be no greater than 0.08 mm.

Checking bending



Check the bending of the connecting rod (5) by comparing two points C and D of the pin (3) on the vertical plane of the axis of the connecting rod.

Position the vertical mount (1) of the dial gauge (2) so that this rests on the pin (3) at point C.

Swing the connecting rod backwards and forwards seeking the highest position of the pin and in this condition zero the dial gauge (2).

Shift the spindle (4) with the connecting rod (5) and repeat the check on the highest point on the opposite side **D** of the pin (3). The difference between point **C** and point **D** must be no greater than 0.08 mm.

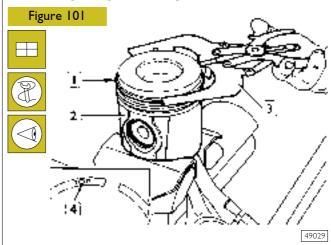
Mounting the connecting rod - piston assembly

Carry out the steps for removal described on page 61 in reverse order.



The connecting rod screws can be reused as long as the diameter of the thread is not less than 13.4 mm.

Mounting the piston rings

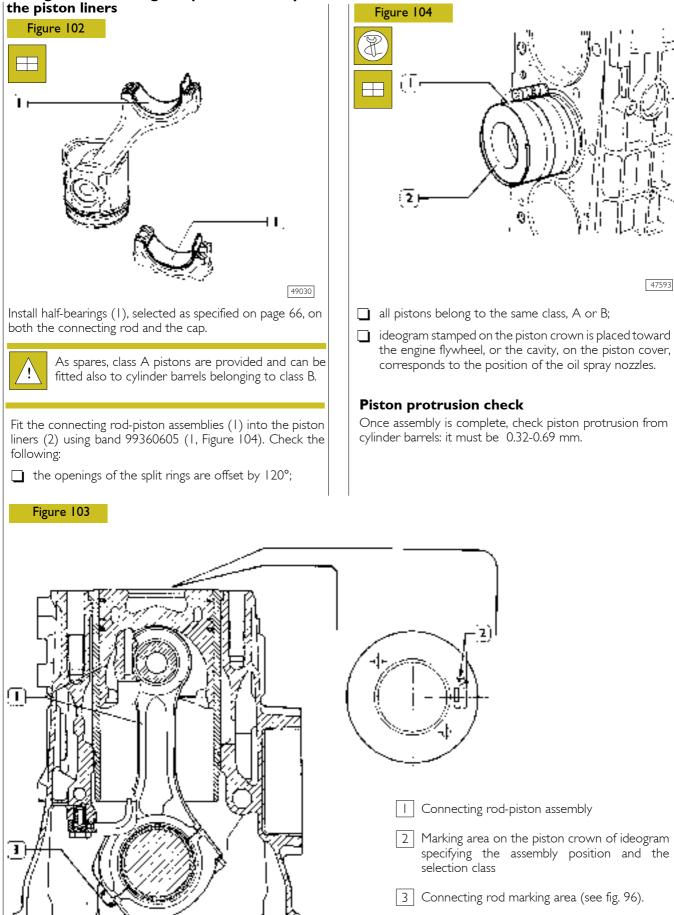


To fit the piston rings (1) on the piston (2) use the pliers 99360184 (3).

The rings need to be mounted with the word "TOP" (4) facing upwards. Direct the ring openings so they are staggered 120° apart.

47593

Fitting the connecting rod-piston assembly into

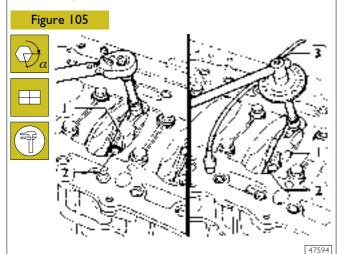


61831

540831 Checking assembly clearance of big end pins

To check the clearance proceed as follows:

Connect the connecting rods to the relative main journals, place a length of calibrated wire on the latter.



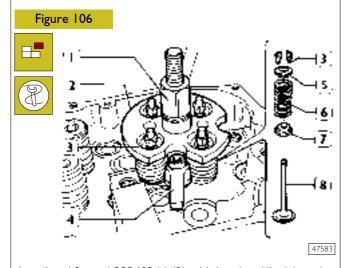
Install the connecting rod caps (1) with half-bearings; tighten the connecting rod cap fixing screws (2) to 50 Nm (5 kgm) torque. By tool 99395216 (3), tighten the screws further at 40° angle.

Remove the caps and check the clearance by comparing the width of the calibrated wire with the scale calibration on the envelope containing the wire.

540610 CYLINDER HEAD

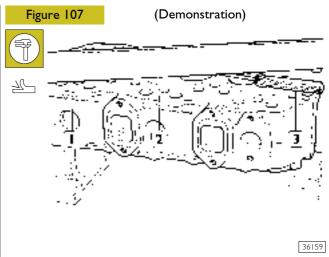
Before taking down the cylinder head, check the seal using the appropriate tool; in case of leakage replace the cylinder head.

Valve removal



Install and fix tool 99360264 (2) with bracket (4); tighten by lever (1) until cotters are removed (3); remove the tool (2) and the upper plate (5), the spring (6) and the lower plate (7). Repeat the operation on all the valves. Turn the cylinder head upside down and remove the valves (8).

Checking the planarity of the head on the cylinder block



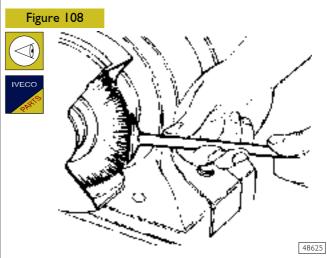
The planarity (1) is checked using a ruler (2) and a thikness gauge (3). If deformations exist, surface the head using proper surface grinder; the maximum amount of material to be removed is 0.2 mm.



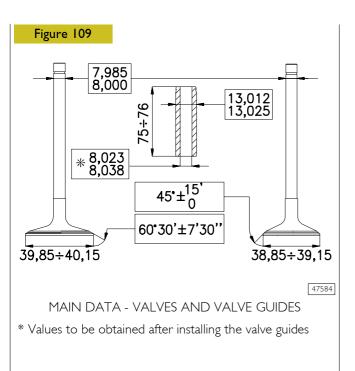
After leveling, make sure that valve sinking and injector protrusion are as described in the relative paragraph.

540622 VALVE

Removing deposits and checking the valves

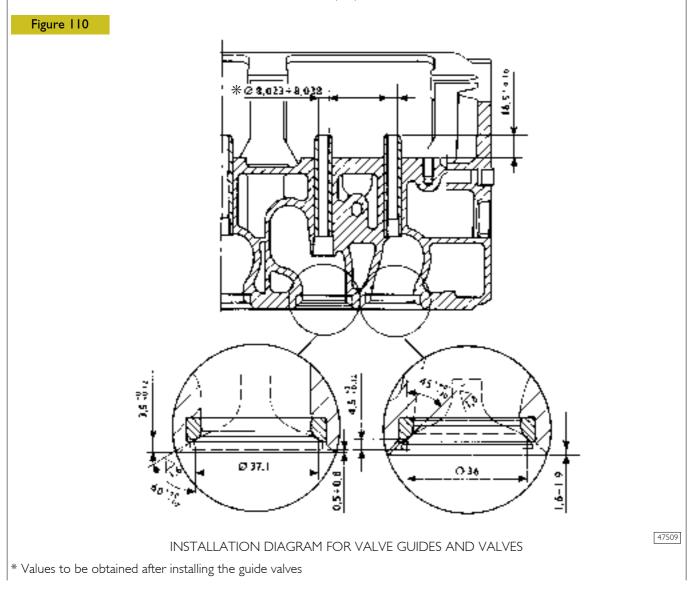


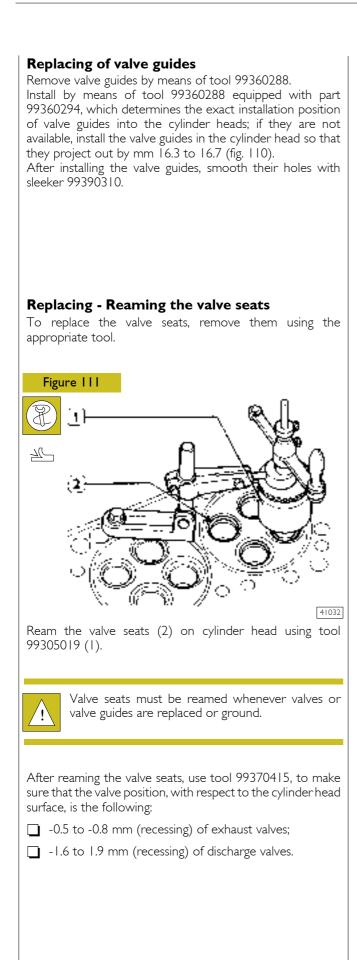
Remove carbon deposits using the metal brush supplied. Check that the valves show no signs of seizure or cracking. Check the diameter of the valve stem using a micrometer (see fig. 109) and replace if necessary.



540667 VALVE GUIDES

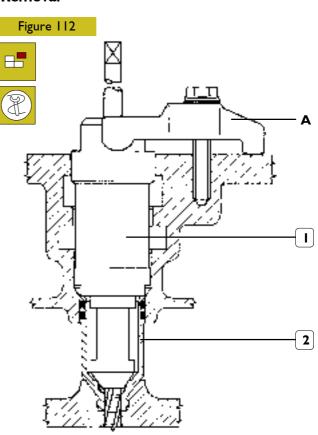
Check, by means of a micrometer, that valve stem diameters are as specified; if necessary, grind the valves seat with a grinder, removing the minimum quantity of material.





540613 REPLACING INJECTOR HOLDER CASES

Removal

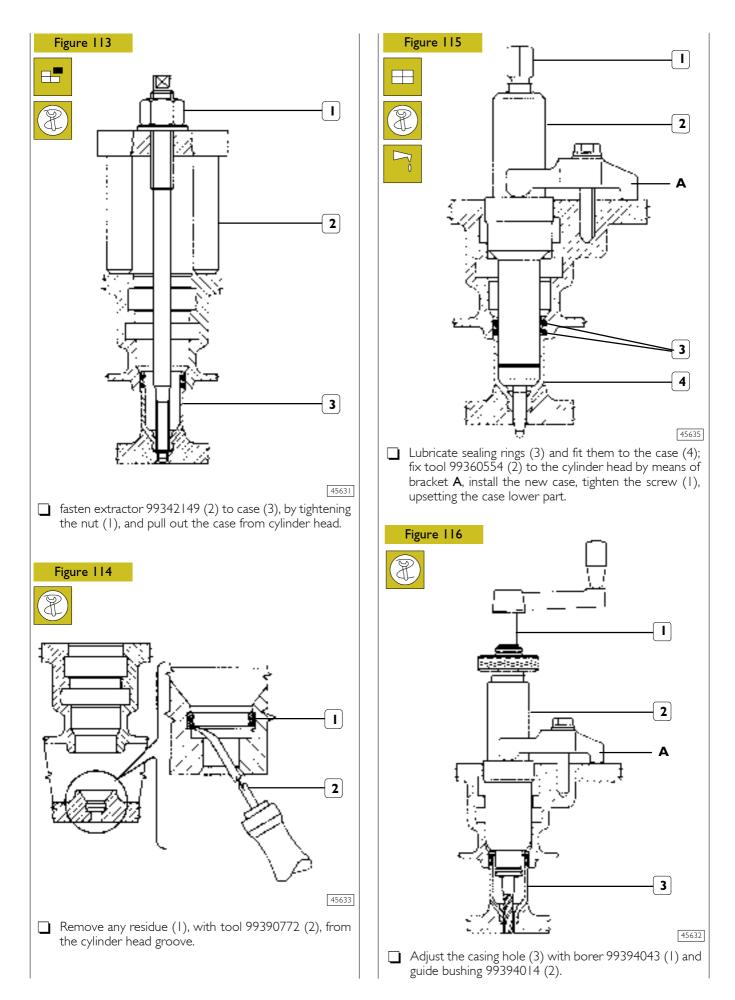


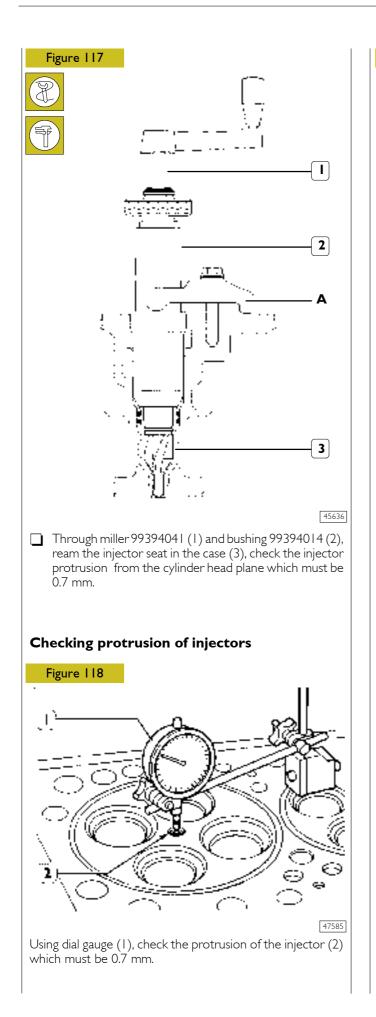
45634

To replace the injector case (2), act as follows:

 \Box thread the case (2) with tool 99390804 (1).

Carry out operations described in figs. 112-115-116-117 by fixing tools to the cylinder head by means of braket A.





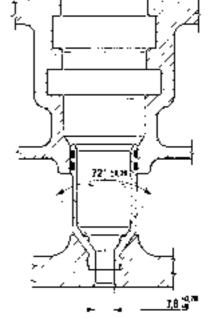
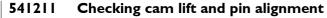


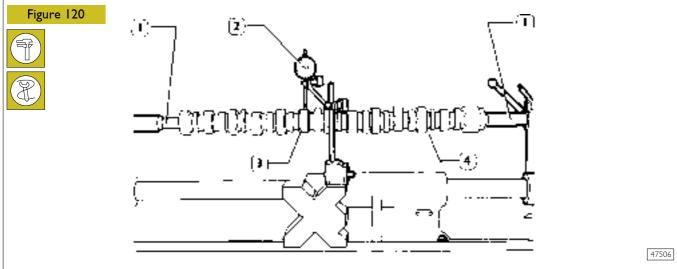
Figure 119

44909

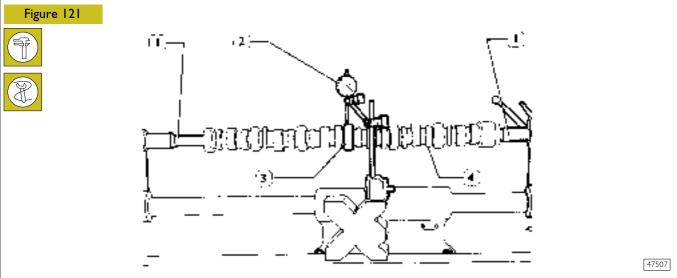
INSTALLATION DIAGRAM FOR INJECTOR CASE

5412 TIMING GEAR

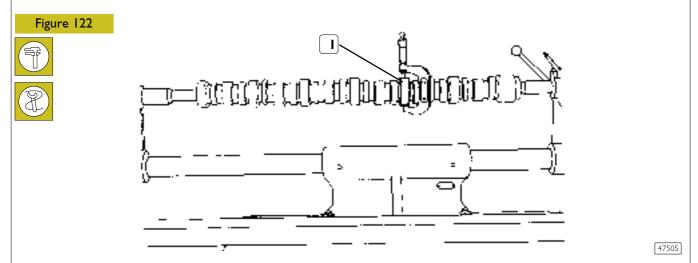




Place the camshaft (4) on the tailstock (1) and check cam lift (3) using a centesimal gauge (2); values are shown in table on page 20.

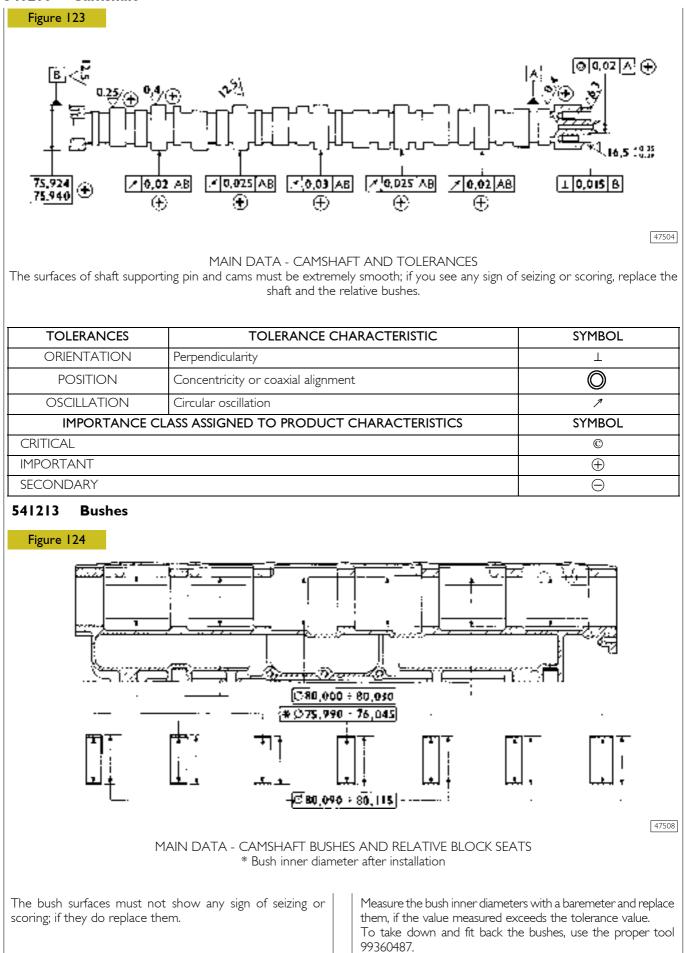


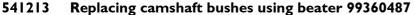
When the camshaft (4) is on the tailstock (1), check alignment of supporting pin (3) using a centesimal gauge (2); it must not exceed 0.030 mm. If misalignment exceeds this value, replace the shaft.

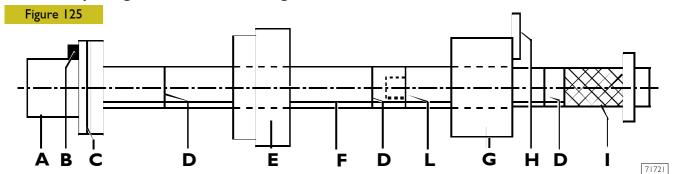


In order to check installation clearance, measure bush inner diameter and camshaft pin (1) diameter; the real clearance is obtained by their difference.

If clearance exceeds 0.150 mm, replace bushes and, if necessary, the camshaft.

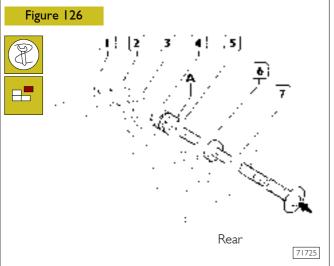






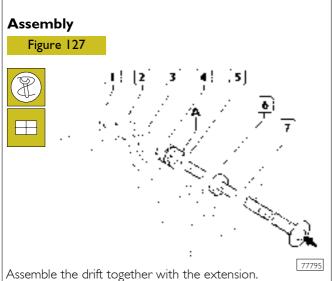
A. Drift with seat for bushings to insert/extract. - B. Grub screw for positioning bushings. - C. Reference mark to insert seventh bushing correctly. - D. Reference mark to insert bushings I, 2, 3, 4, 5, 6 correctly (red marks). - E. Guide bushing. - F. Guide line. - G. Guide bushing to secure to the seventh bushing mount. - H. Plate fixing yellow bushing to cylinder head. - I. Grip. - L. Extension coupling.

Removal



The sequence for removing the bushings is 7, 6, 5, 4, 3, 2, 1. The bushings are extracted from the front of the single seats. Removal does not require the drift extension for bushings 5, 6 and 7 and it is not necessary to use the guide bushing. For bushings 1, 2, 3 and 4 it is necessary to use the extension and the guide bushings.

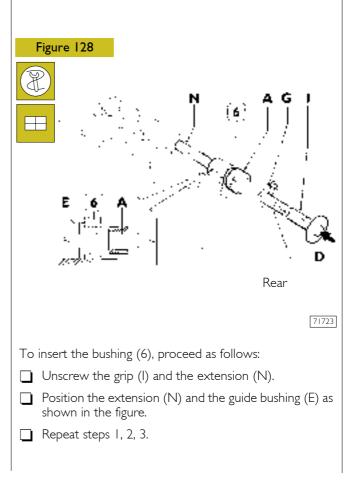
Position the drift accurately during the phase of removal.

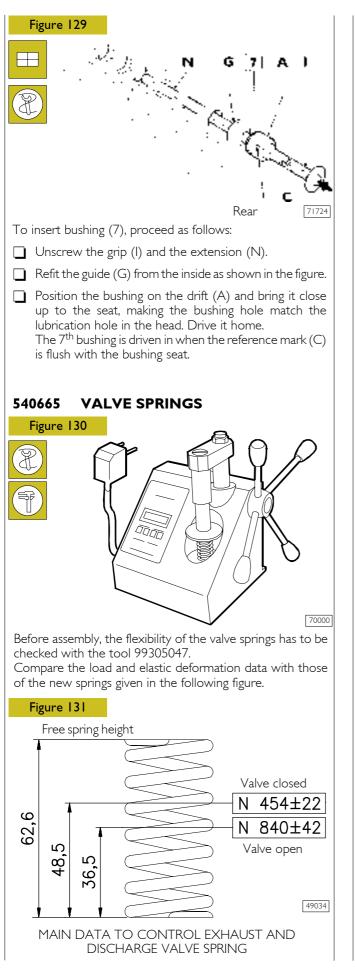


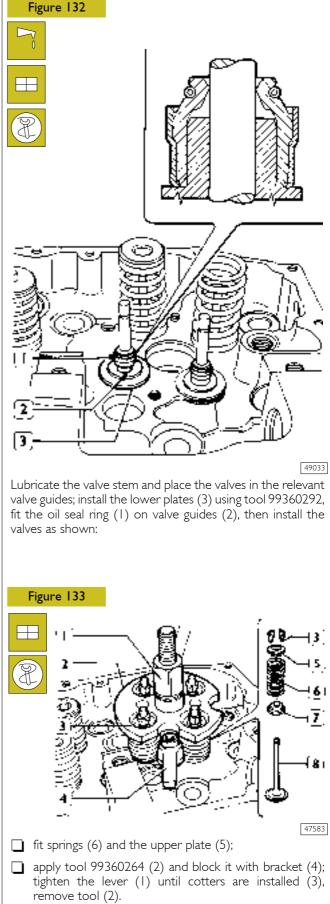
To insert bushings 1, 2, 3, 4 and 5, proceed as follows:

- Position the bushing to insert on the drift (A) making the grub screw on it coincide with the seat (B) (Figure 125) on the bushing.
- 2 Position the guide bushing (E) and secure the guide bushing (G) (Figure 125) on the seat of the 7th bushing with the plate (H).
- 3 While driving in the bushing, make the reference mark (F) match the mark (M). In this way, when it is driven home, the lubrication hole on the bushing will coincide with the oil pipe in its seat.

The bushing is driven home when the 1st red reference mark (D) is flush with the guide bushing (G).

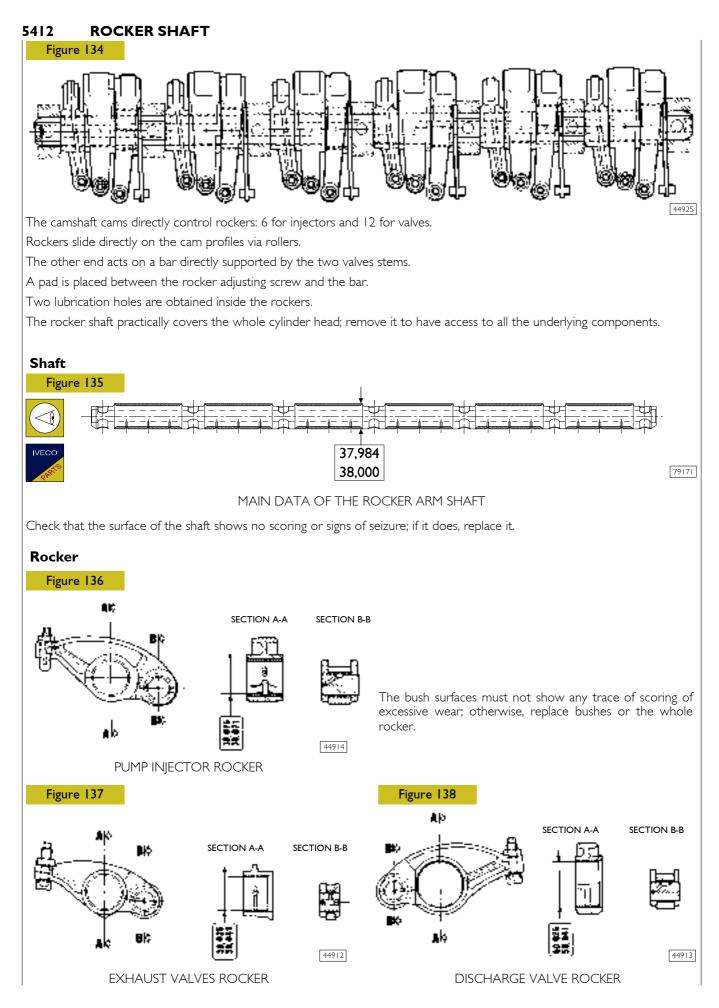




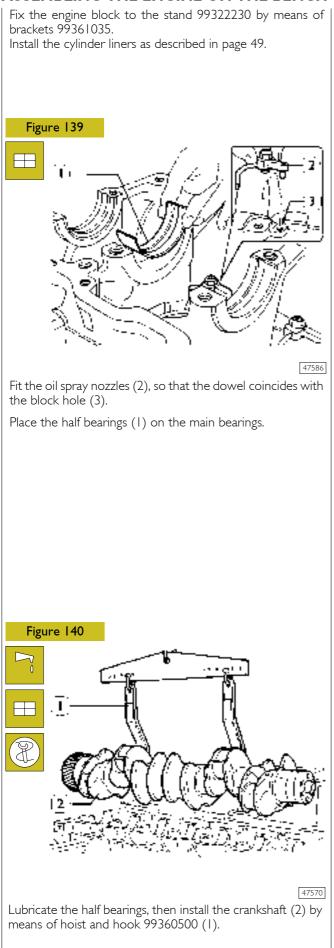


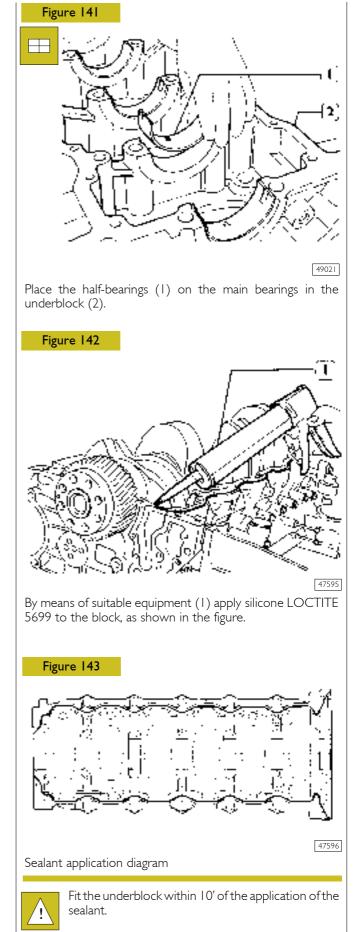
Fitting the valves and oil seal ring

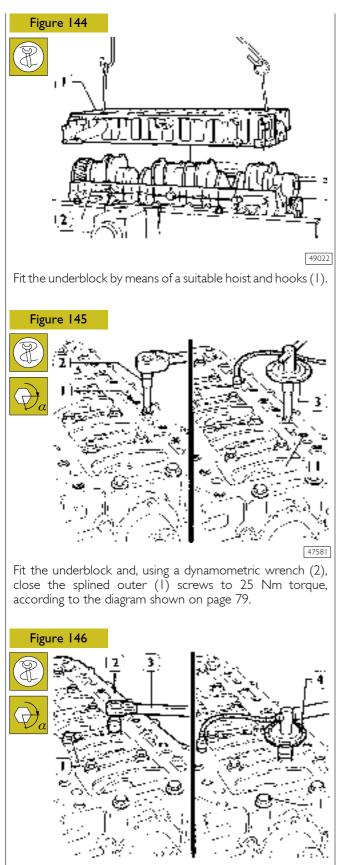
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ASSEMBLING THE ENGINE ON THE BENCH





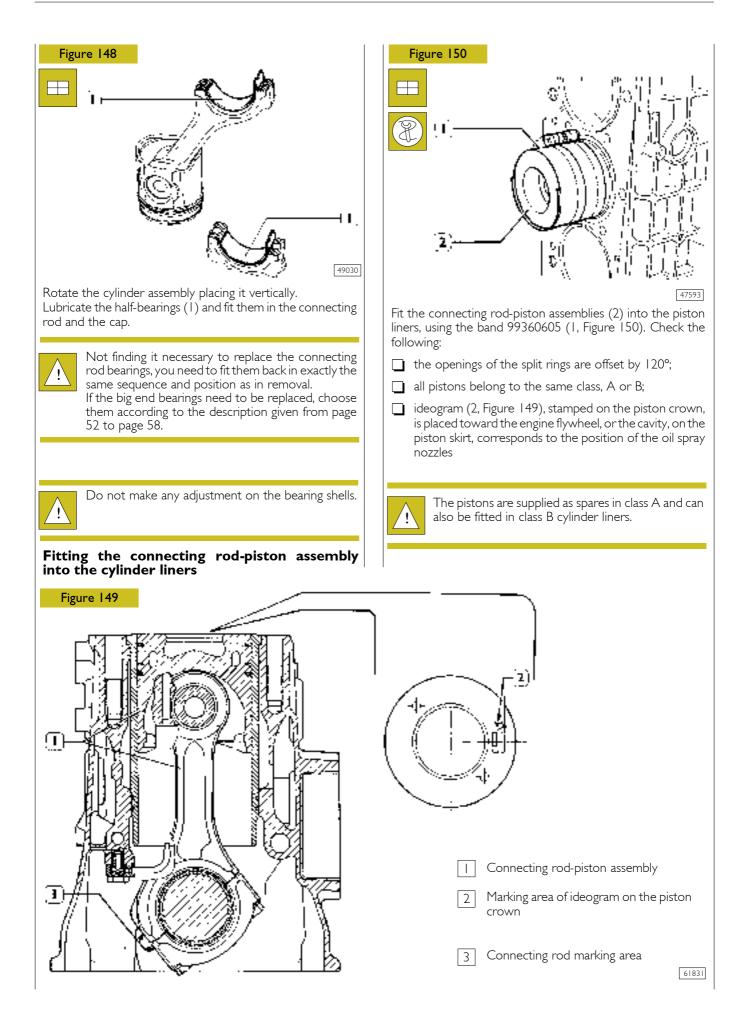


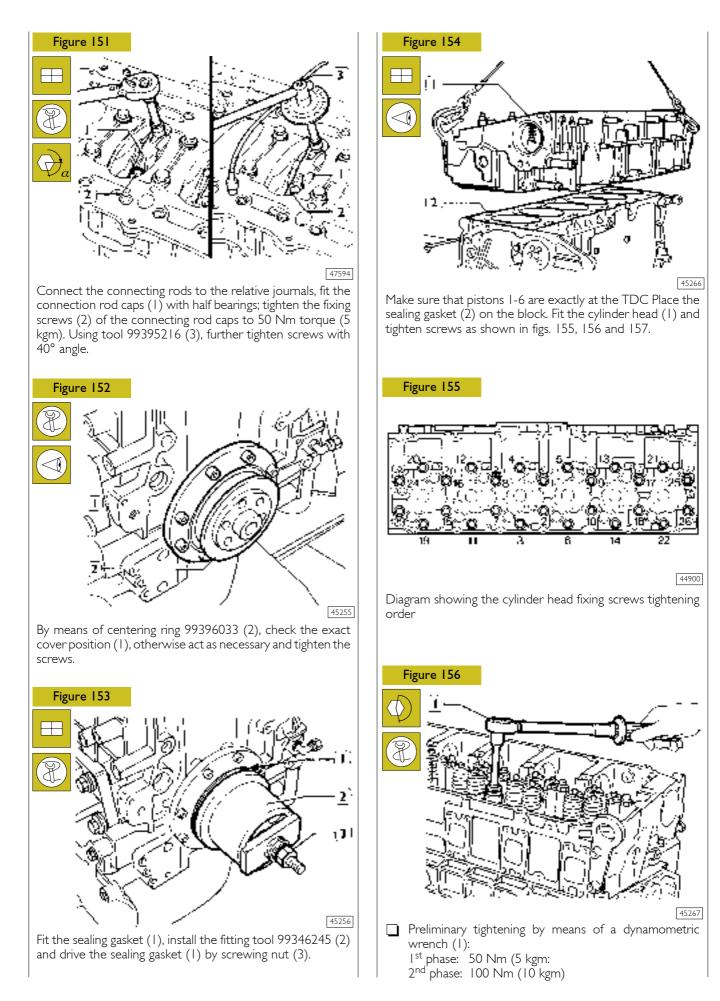
47579

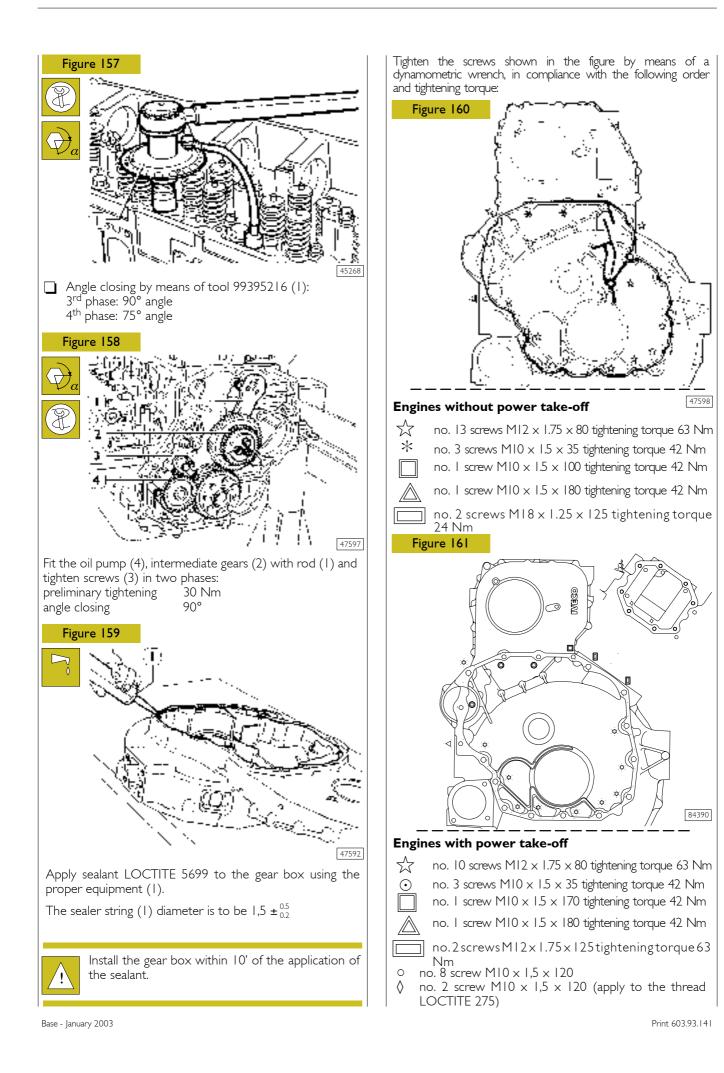
Close the inner screws (1) to 140 Nm torque by means of a dynamometric wrench (3), then with two further angular phases $60^{\circ} + 60^{\circ}$, using tool 99395216 (4). Tighten again the outer screws (1, Figure 145) with 90° angular closing, using tool 99395215 (3, Figure 145).

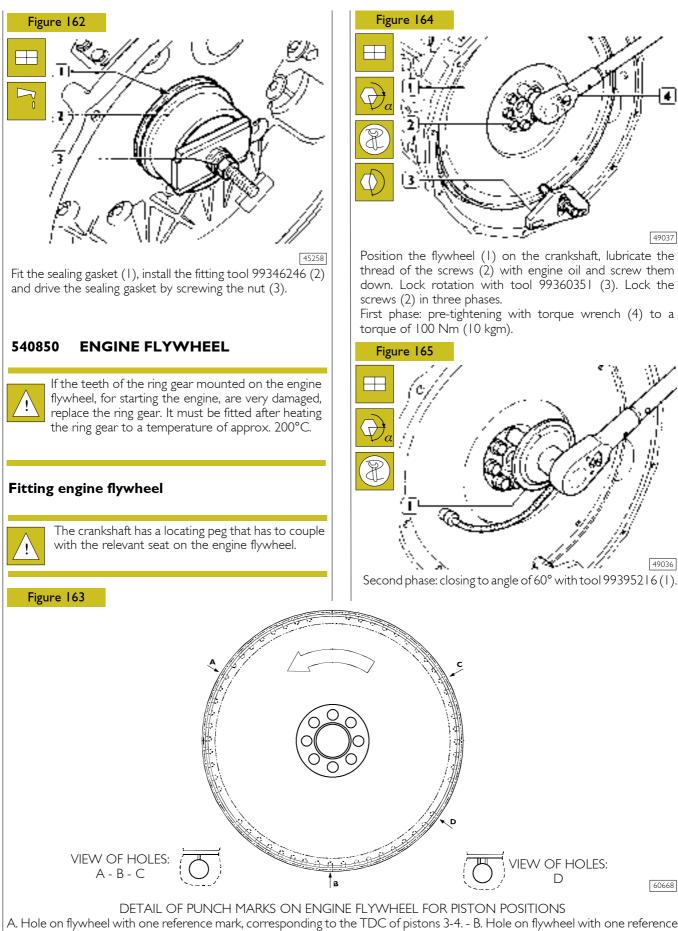
Figure 147 First phase: outer FRONT SIDE screws preliminary tightening (25 Nm) 44897 削 Ú¢ 'att à ŵ \otimes ω £Ο 13 12 Second phase: FRONT SIDE inner screws preliminary tightening ತುಲಹ (140 Nm) ୍ର୍ବ R $\overline{\langle x \rangle}$ iΟ 65 44898 ω <u>(</u> à 0 ŵ Third phase: inner screws FRONT SIDE angle closing (60°) මු () ୍କୃତ Ħ R ið 44898 ١Ì ω 69 É. Fourth phase: FRONT SIDE inner screws angle closing (60°) <u>ා</u>රල <u>80</u>8 രിപ് ത 86 ic? 44898 Fifth phase: FRONT SIDE outer screws angle closing (90°) 44899

DIAGRAM SHOWING THE UNDERBLOCK FIXING SCREWS TIGHTENING ORDER

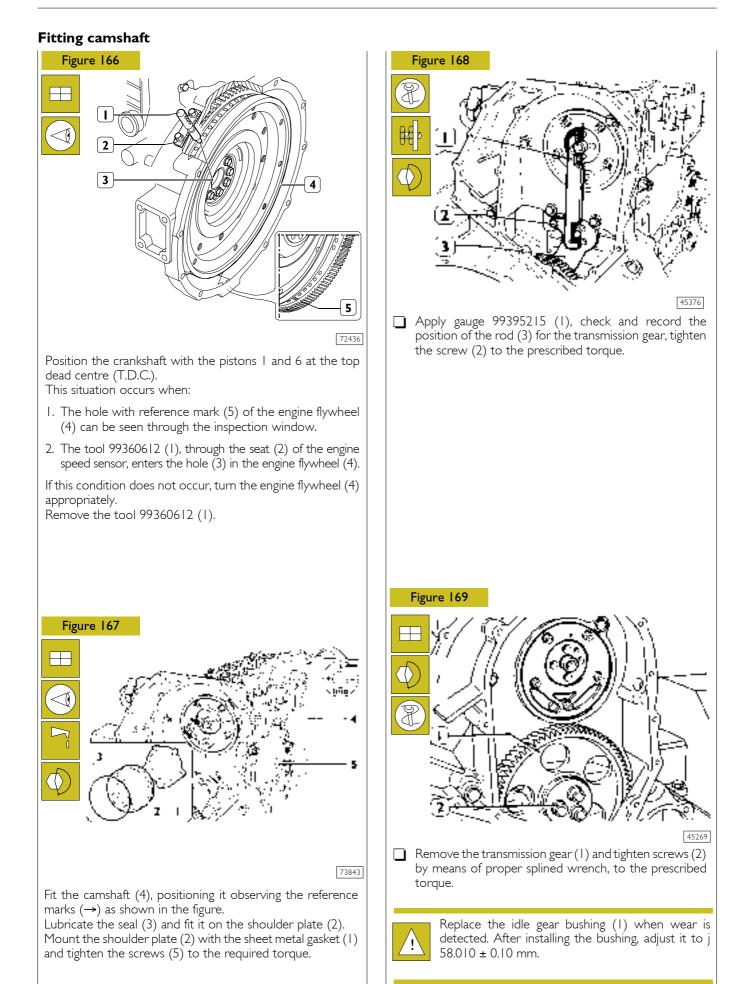


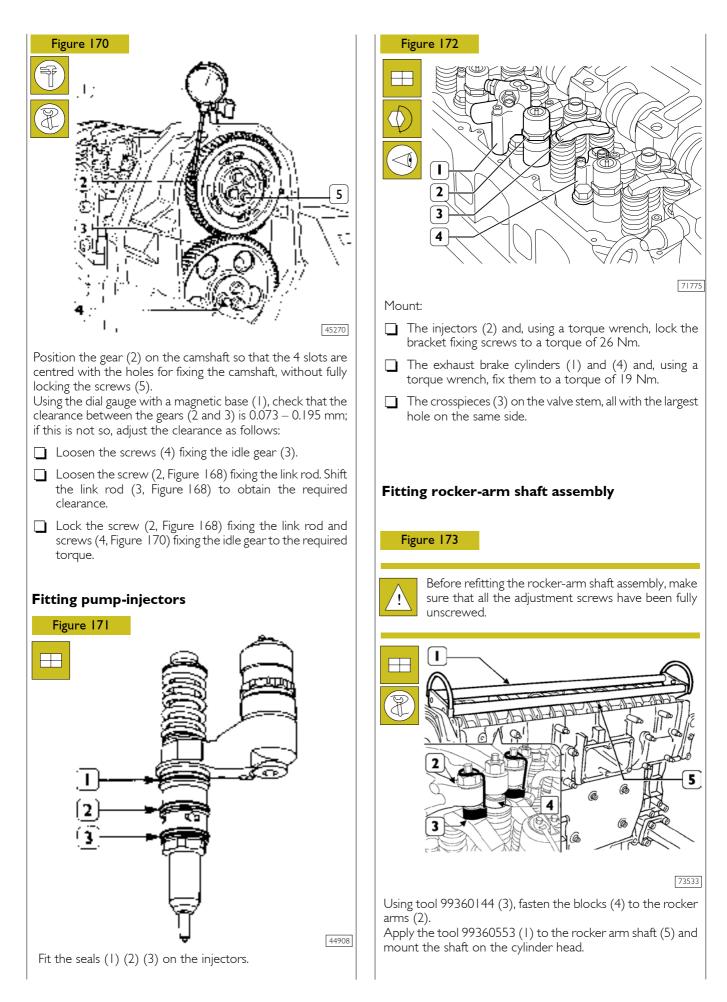


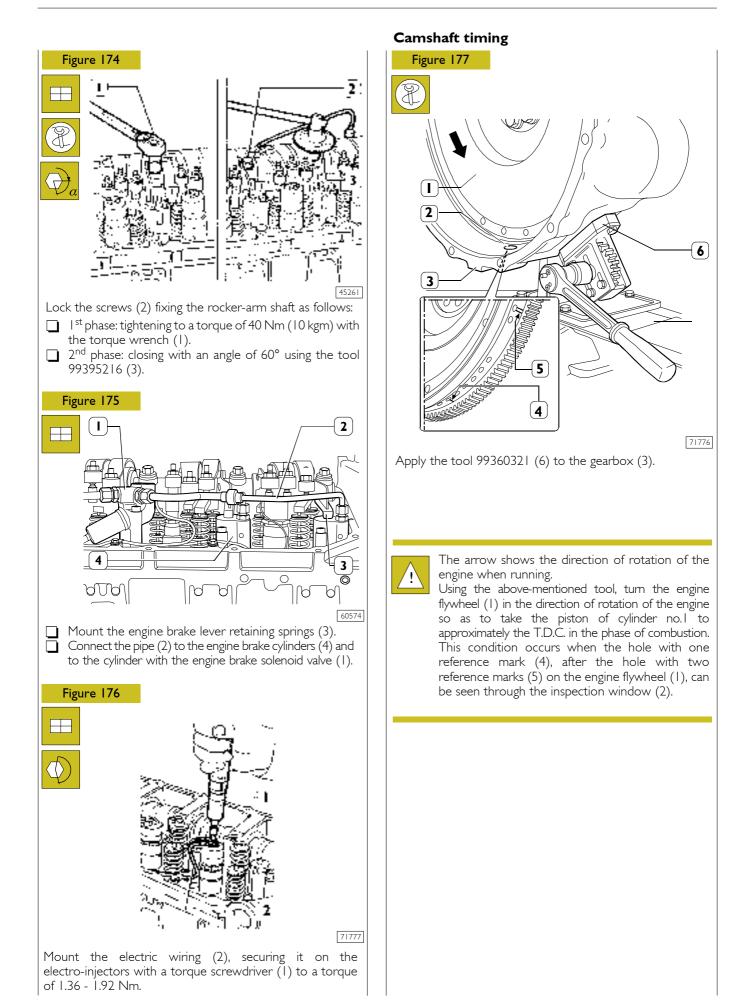


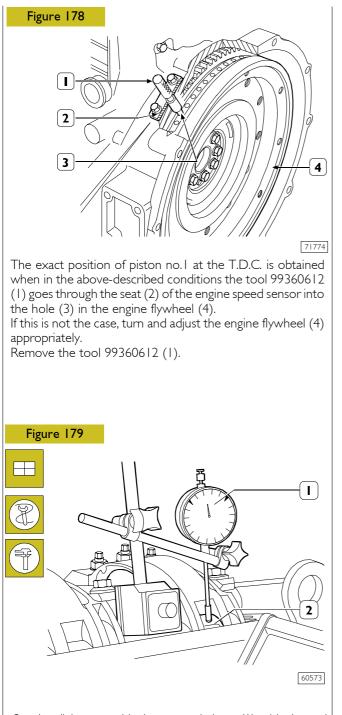


A. Hole on flywheel with one reference mark, corresponding to the TDC of pistons 3-4. - B. Hole on flywheel with one reference mark, corresponding to the TDC of pistons 1-6. - C. Hole on flywheel with one reference mark, corresponding to the TDC of pistons 2-5. - D. Hole on flywheel with two reference marks, position corresponding to 54°.







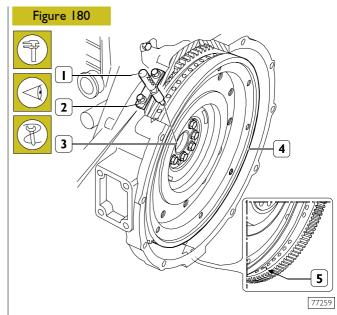


Set the dial gauge with the magnetic base (1) with the rod on the roller (2) of the rocker arm that governs the injector of cylinder no.1 and pre-load it by 6 mm.

With tool 99360321 (6, Figure 177), turn the crankshaft clockwise until the pointer of the dial gauge reaches the minimum value beyond which it can no longer fall.

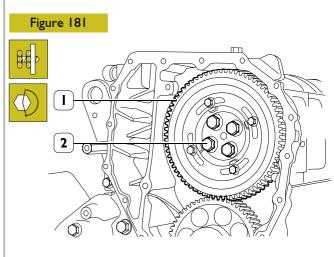
Reset the dial gauge.

Turn the engine flywheel anticlockwise until the dial gauge gives a reading for the lift of the cam of the camshaft of 4.90 \pm 0.05 mm.



The camshaft is in step if at the cam lift values of 4.90 ± 0.05 mm there are the following conditions:

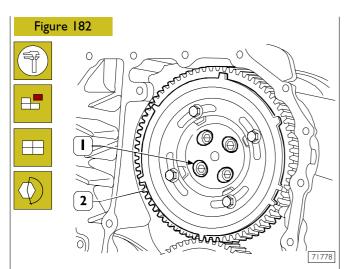
- 1) The hole marked with a notch (5) can be seen through the inspection window
- 2) The tool 99360612 (1) through the seat (2) of the engine speed sensor goes into the hole (3) in the engine flywheel (4).



60575

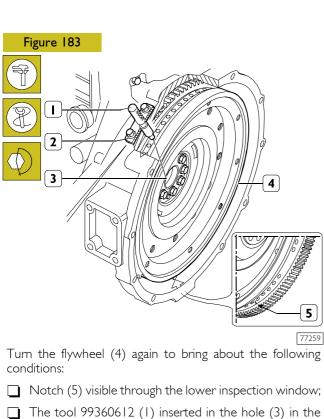
If you do not obtain the conditions illustrated in Figure 180 and described in points 1 and 2, proceed as follows:

- Loosen the screws (2) securing the gear (1) to the camshaft and utilize the slots (1, Figure 182) on the gear (2, Figure 182).
- Turn the engine flywheel appropriately so as to bring about the conditions described in points I and 2 Figure 180, it being understood that the cam lift must not change at all.
- 3) Lock the screws (2) and repeat the check as described above.
- 4) Tighten the screws (2) to the required torque.



When it is not possible to adjust advance through the slots (1) and the camshaft turns because integral with the gear (2); thus the cam lift reference value varies and it is necessary to proceed in the following way:

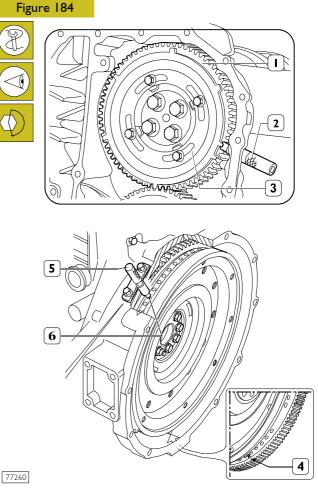
- Fasten the screws (2, Figure 181) and rotate the engine flywheel clockwise by ~1/2 turn;
- 2) Turn the engine flywheel anticlockwise until the dial gauge gives a reading of the lift of the cam of the camshaft of 4.90 ± 0.05 mm
- Take out the screws (2, Figure 181) and remove the gear
 (2) from the camshaft.



The tool 99360612 (1) inserted in the hole (3) in the engine flywheel (4) through the seat (2) of the engine speed sensor. Mount the gear (2, Figure 182) with the 4 slots (1, Figure 182) centred with the fixing holes of the camshaft, locking the relevant screws to the required tightening torque. Check the timing of the shaft by first turning the flywheel clockwise to discharge the cam completely and then turn the flywheel anticlockwise until the dial gauge gives a reading of: $4.90 \pm 0.05 \text{ mm}$

Check the timing conditions described in Figure 180.

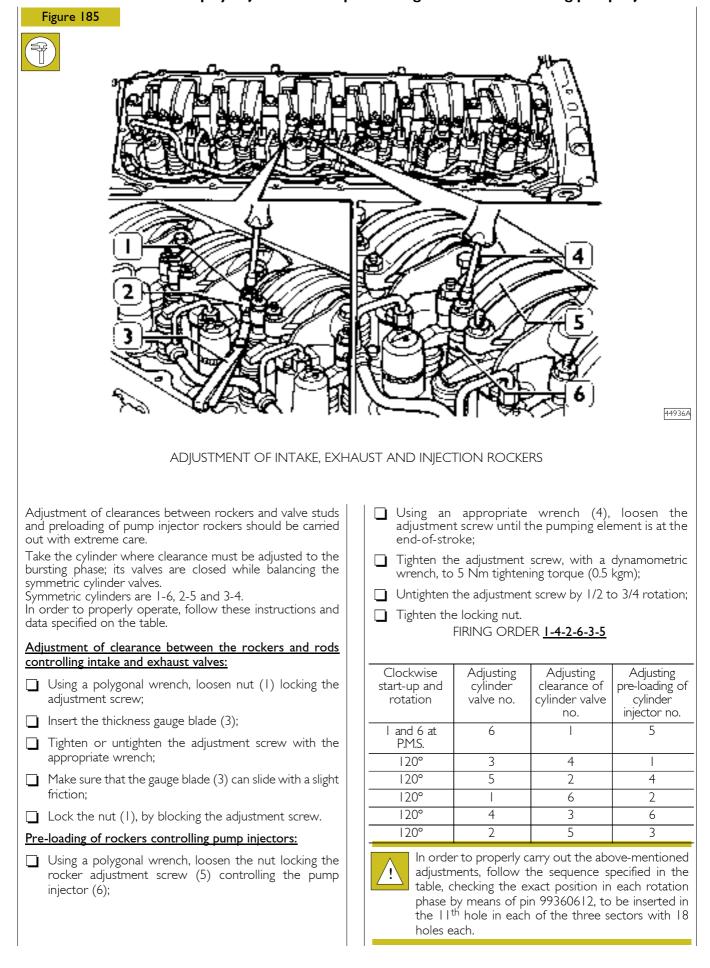
Phonic wheel timing



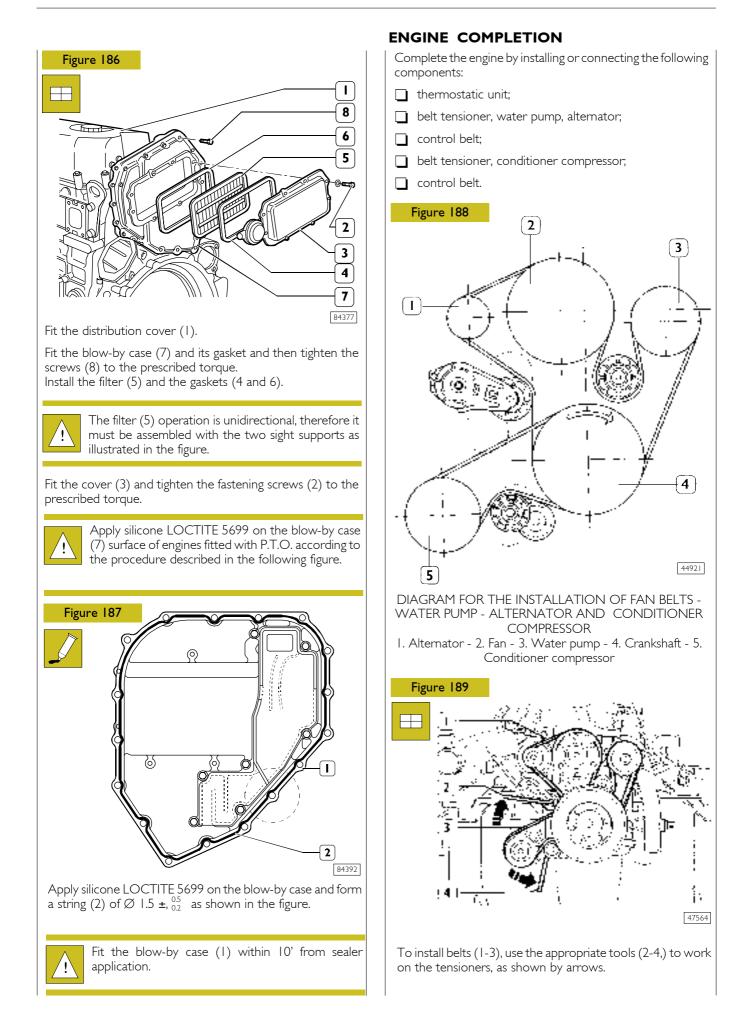
Turn the output shaft bringing cylinder piston I at compression stage to TDC.; turn the flywheel by about I/4 turn in opposite direction than normal direction of rotation. Turn the flywheel again according to normal direction of rotation until the hole marked with the double notch (4) can be seen through the inspection hole set under the flywheel housing. Fit tool 99360612 (5) into the flywheel sensor seat (6).

Fit tool 99360613 (2), through the timing sensor seat, on the tooth obtained on the phonic wheel.

Should tool (2) fitting be difficult, slacken screws (3) and direct the phonic wheel (1) properly to position the tool (2) on the tooth. Tighten the screws (3).



Intake and exhaust rocker play adjustment and pre-loading of rockers controlling pump injectors.



Automatic tensioners do not require further	Figure 190
adjustments after the installation.	⊞ (i) _
 damping flywheel; fan; compressor; fuel pump; fuel filter and piping; pre-heating resistance; intake manifold; soundproof shields; 	
 heat exchanger; oil filter, lubricating the gasket; rockers caps; exhaust manifold; turbocharger and its oil and water piping; power take-off (P.T.O.) (if any) and related pipes; oil level stick and oil vapor vent; 	
rotate the engine and install the oil rose pipe.	 place the gasket (4) on the oil sump (1), position the spacer (3) and fit the sump to the engine block by tightening screws (2) to the prescribed torque; electric connections and sensors; remove the engine from the stand and fit the starter; fill the engine with the oil quantity required.

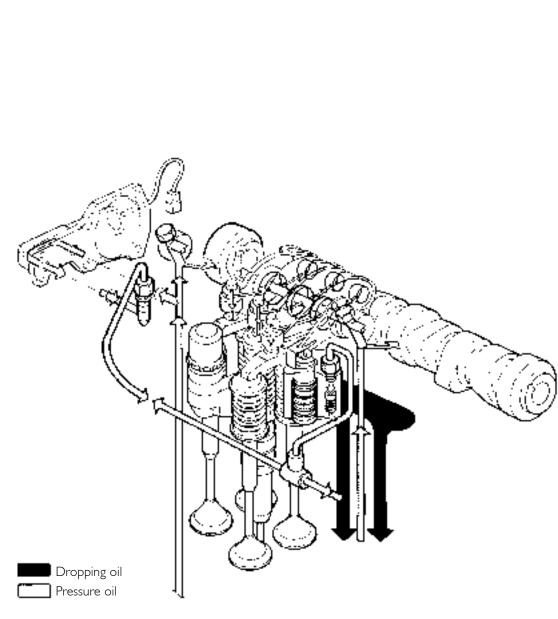
Figure 191

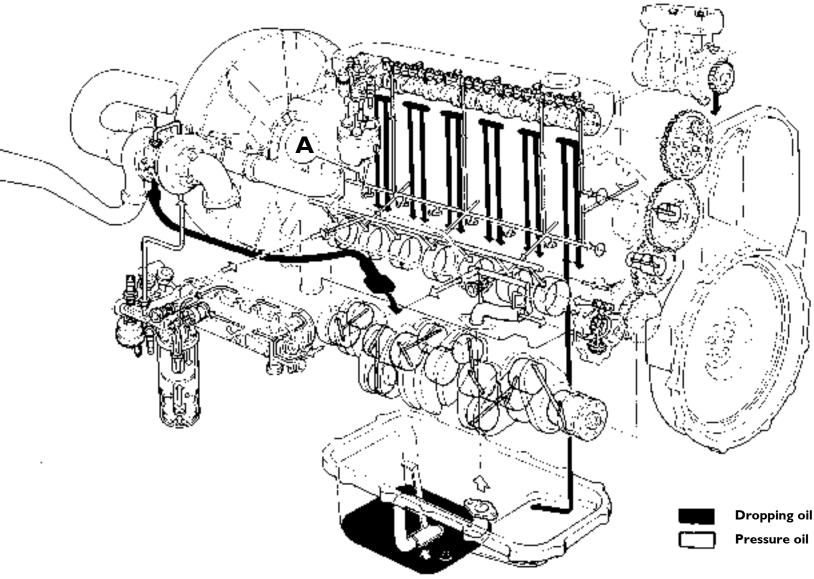
5430 LUBRICATION

Engine lubrication is obtained with a gear pump driven by the crankshaft via gears.

A heat exchanger governs the temperature of the lubricating oil.

The oil filter, signalling sensors and safety valves are installed in the intercooler.



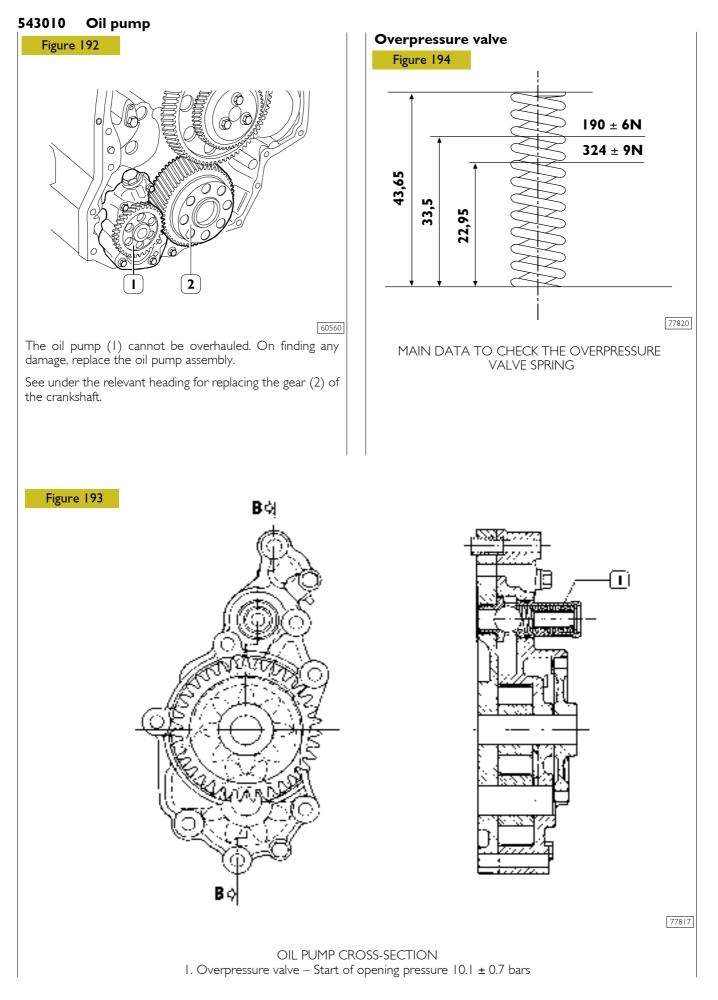


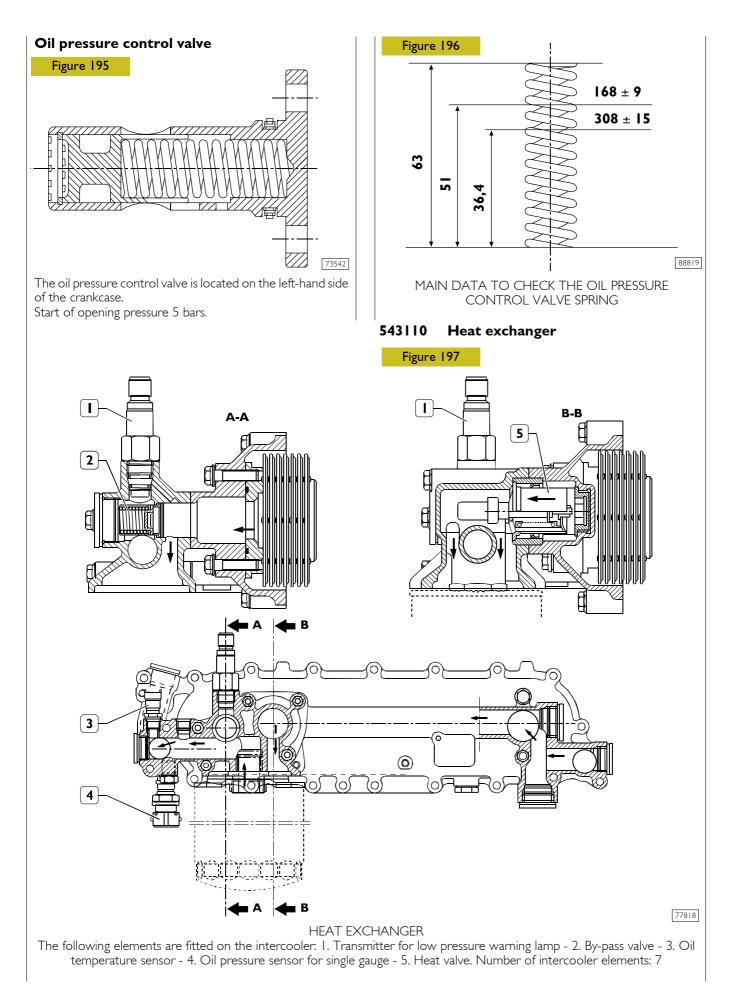
Lubrication circuit

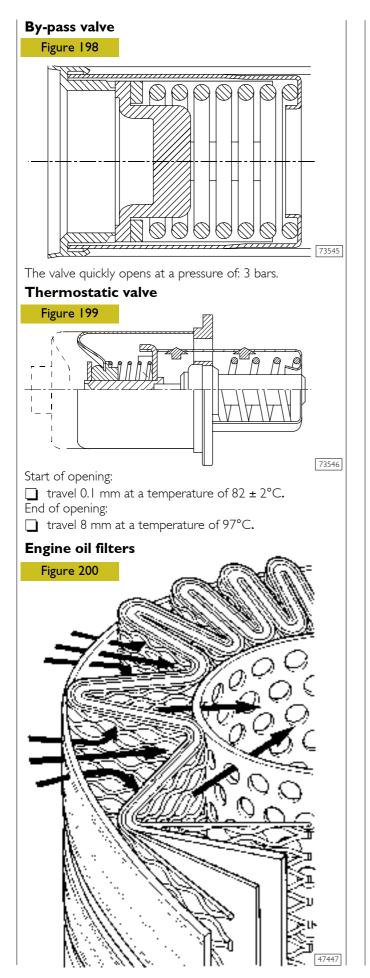
44918

94 ENGINE F2B

Stralis AT/AD







This is a new generation of filters that permit much more thorough filtration as they are able to holder back a greater amount of particles of smaller dimensions than those held back by conventional filters with a paper filtering element.

These high-filtration devices, to date used only in industrial processes, make it possible to:

reduce the wear of engine components over time;

maintain the performance/specifications of the oil and thereby lengthen the time intervals between changes.

External spiral winding

The filtering elements are closely wound by a spiral so that each fold is firmly anchored to the spiral with respect to the others. This produces a uniform use of the element even in the worst conditions such as cold starting with fluids with a high viscosity and peaks of flow. In addition, it ensures uniform distribution of the flow over the entire length of the filtering element, with consequent optimization of the loss of load and of its working life.

Mount upstream

To optimize flow distribution and the rigidity of the filtering element, this has an exclusive mount composed of a strong mesh made of nylon and an extremely strong synthetic material.

Filtering element

Composed of inert inorganic fibres bound with an exclusive resin to a structure with graded holes, the element is manufactured exclusively to precise procedures and strict quality control.

Mount downstream

A mount for the filtering element and a strong nylon mesh make it even stronger, which is especially helpful during cold starts and long periods of use. The performance of the filter remains constant and reliable throughout its working life and from one element to another, irrespective of the changes in working conditions.

Structural parts

The o-rings equipping the filtering element ensure a perfect seal between it and the container, eliminating by-pass risks and keeping filter performance constant. Strong corrosion-proof bottoms and a sturdy internal metal core complete the structure of the filtering element.

When mounting the filters, keep to the following rules:

- Oil and fit new seals.
- Screw down the filters to bring the seals into contact with the supporting bases.
- Tighten the filter to a torque of 35÷40 Nm.

5432 COOLING

Description

The engine cooling system works with forced circulation inside closed circuit and can be connected to an additional heater (if any) and to the intarder intercooler. It consists mainly of the following components:

- an expansion reservoir whose plug (1) incorporates two valves – discharge and charge – controlling the system pressure.
- a coolant level sensor placed at the bottom of the expansion reservoir with two coupling points:
 - coupling point for sensor S1 6 litres
 - 3.7 litres • coupling point for sensor S2
- an engine cooling unit to dissipate the heat taken by the coolant from the engine through the intercooler.
- a heat exchanger to cool down lubrication oil;
- a water pump with centrifugal system incorporated in the cylinder block;
- an electric fan consisting of a 2-speed electro-magnetic joint equipped with a neutral wheel shaft hub fitted with a metal plate moving along the axis and where the fan is installed. It is controlled electronically by the vehicle Multiplex system.
- a 3-way thermostat controlling the coolant circulation. **O**peration

The water pump is actuated by the crankshaft through a poli-V belt and sends coolant to the cylinder block, especially to the cylinder head (bigger quantity). When the coolant temperature reaches and overcomes the operating temperature, the thermostat is opened and from here the coolant flows into the radiator and is cooled down by the fan. The pressure inside the system depending on the temperature variation is controlled by the discharge and charge valves incorporated in the expansion reservoir filling plug (1).

The discharge valve has a double function:

- keep the system under light pressure in order to raise the coolant boiling point;
- discharge the pressure surplus in the atmosphere as a result of the coolant high temperature.

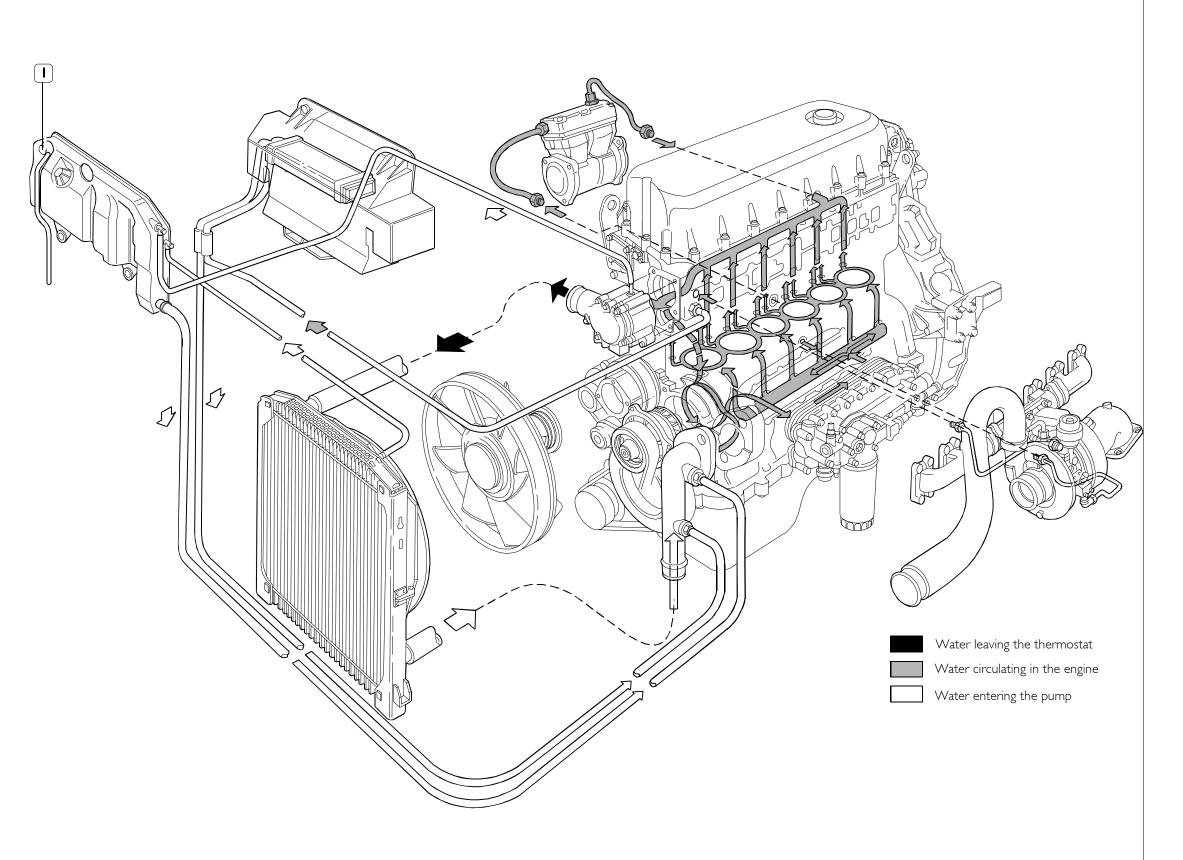
The charge valve makes it possible to transfer the coolant from the expansion reservoir to the radiator when a depression is generated inside the system as a result of the coolant volume reduction depending on the fall in the coolant temperature. Discharge valve opening:

bar

bar

+ 0.2 0.9 - 0.1 • Ist breather + 0.2 • 2nd breather 1.2 l.2 _{- 0.1} bar -0.03 _{- 0.02} bar Charge valve opening

Figure 201

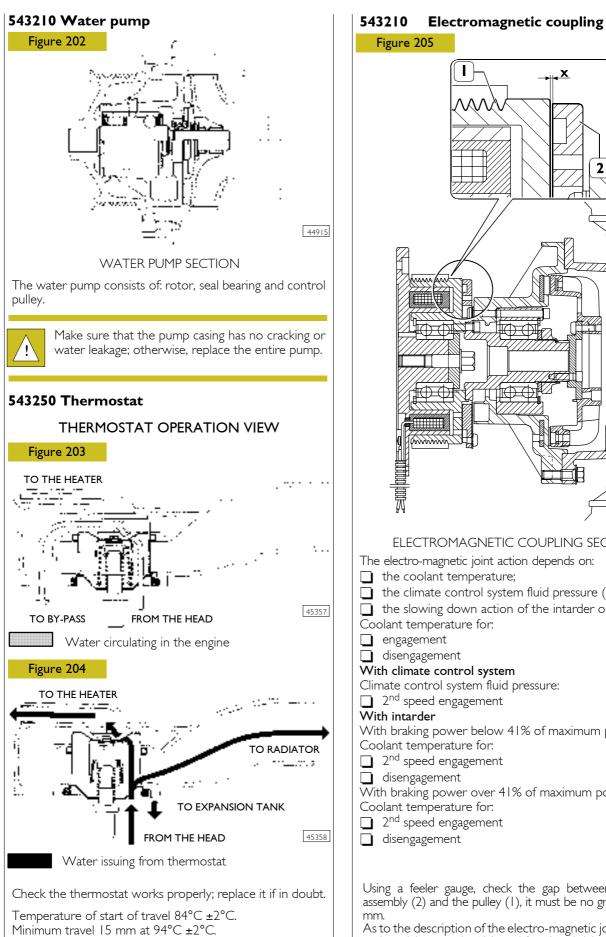


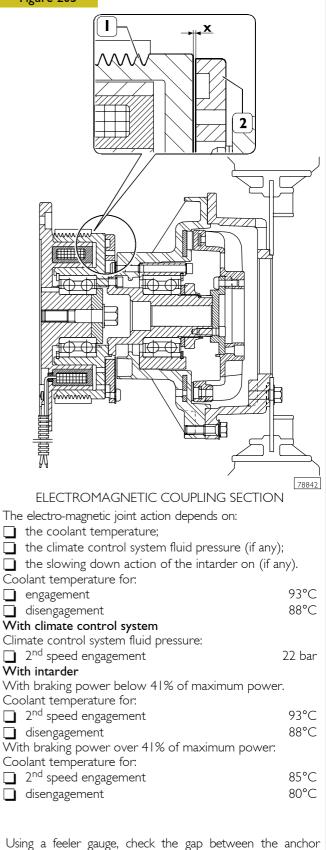
ENGINE F2B **99**

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100 ENGINE F2B

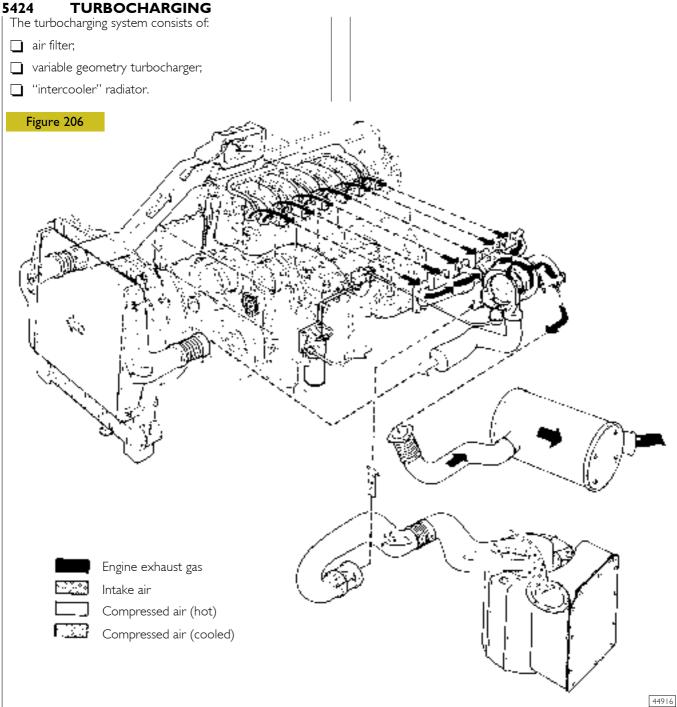
Stralis AT/AD





assembly (2) and the pulley (1), it must be no greater than 2.5

As to the description of the electro-magnetic joint operation and servicing, see the "Manual for electric/electronic system repairing'' St. 603.93.191.



TURBOCHARGING DIAGRAM

TURBOCHARGER

Operating principle

The variable geometry turbocharger (VGT) consists of a centrifugal compressor and a turbine, equipped with a mobile device which adjusts the speed by changing the area of the passing section of exhaust gases to the turbine.

Thanks to this solution, gas velocity and turbine speed can be high even when the engine is idling.

If the gas is made to go through a narrow passage, in fact, it flows faster, so that the turbine rotates more quickly.

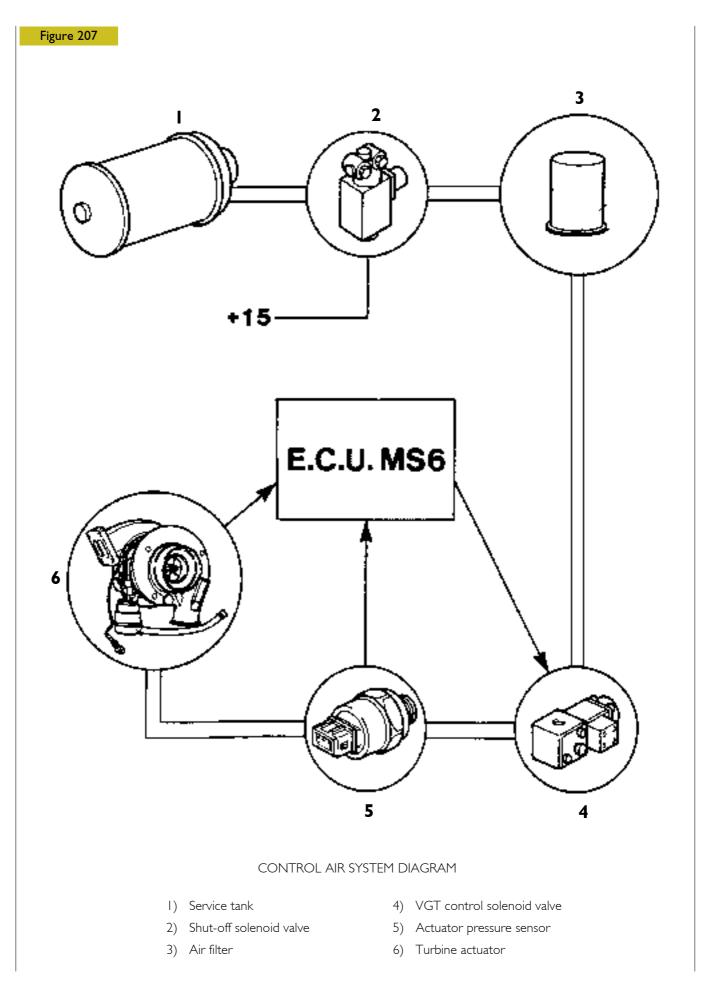
The movement of the device, choking the exhaust gas flowing section, is carried out by a mechanism, activated by a pneumatic actuator.

This actuator is directly controlled by the electronic control unit by a proportional solenoid valve.

The device is in maximum closing condition at idle speed.

At high engine operating speed, the electronic control system is activated and increases the passing section, in order to allow the in-coming gases to flow without increasing their speed.

A toroidal chamber is obtained during the casting process in the central body for the passage of the coolant.



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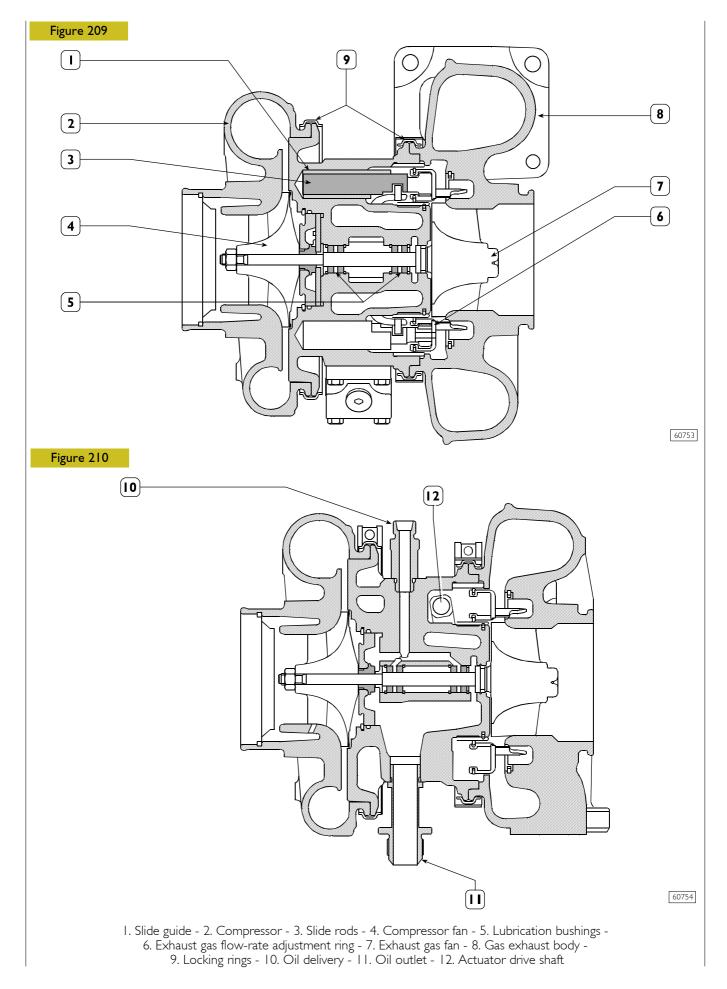
TURBO COMPRESSOR HOLSET HX 40V Figure 208 I 6 2 3 4 Ø 7 -V Ø 5 8 I. Air delivery to the intake manifold - 2. Compressor - 3. Air inlet - 4. Actuator -5. Exhaust gas speed governor - 6. Exhaust gas inlet - 7. Exhaust gas outlet - 8. Turbine 4 L 6 C Î 2 2 7

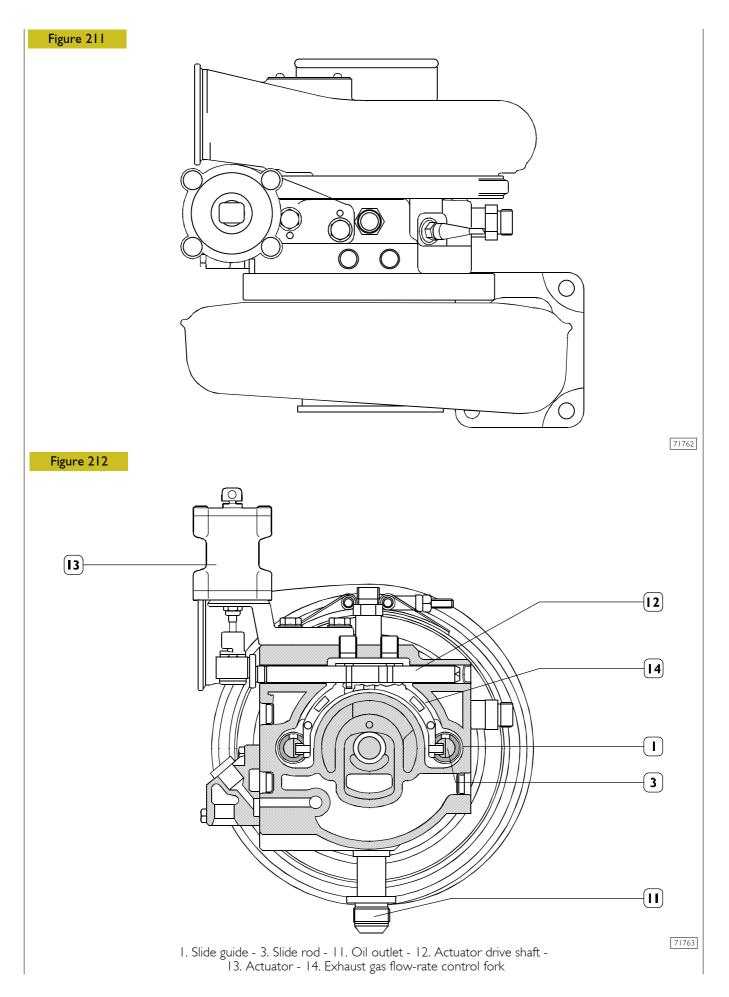
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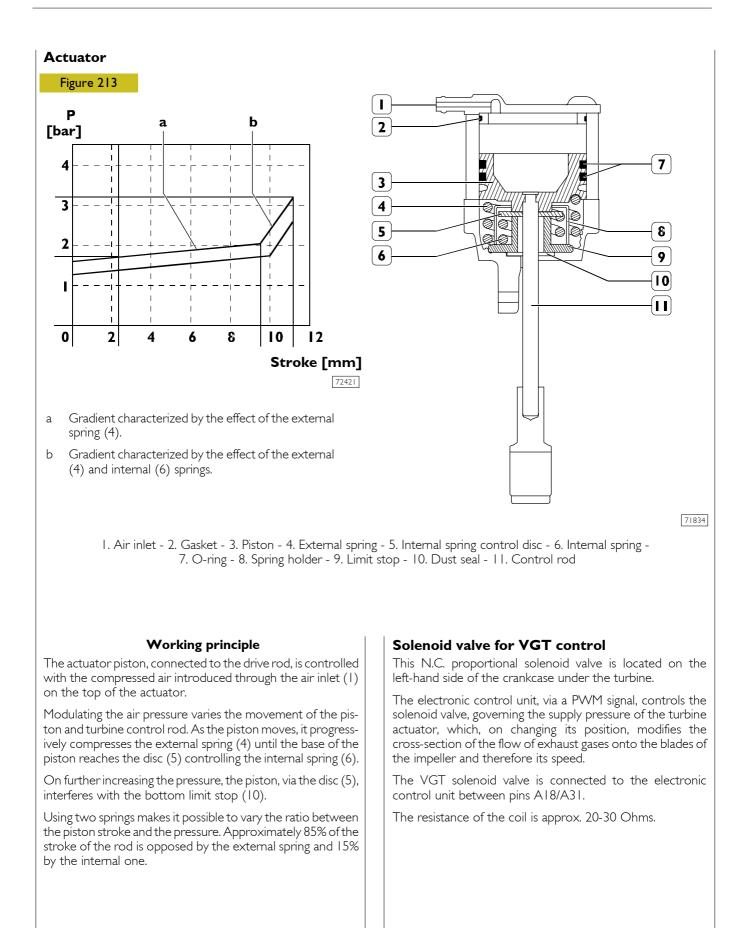
1. Air delivery to the intake manifold - 2. Compressor - 3. Air inlet - 4. Actuator - 5. Exhaust gas flow-rate adjustment ring - 6. Exhaust gas inlet - 7. Exhaust gas outlet - 8. Turbine - 9. Exhaust gas flow-rate control fork

3

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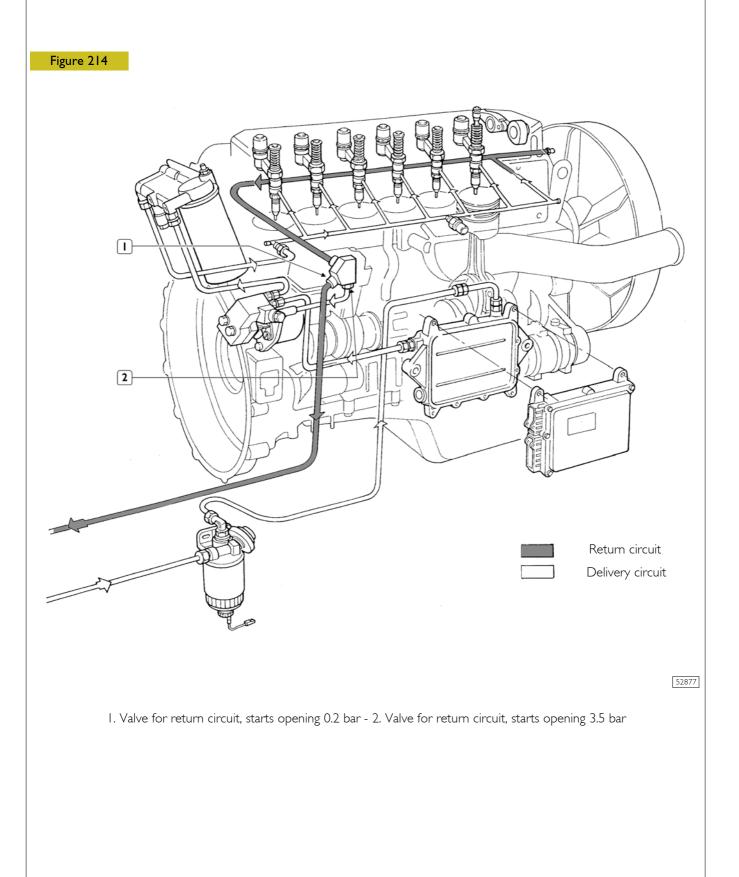


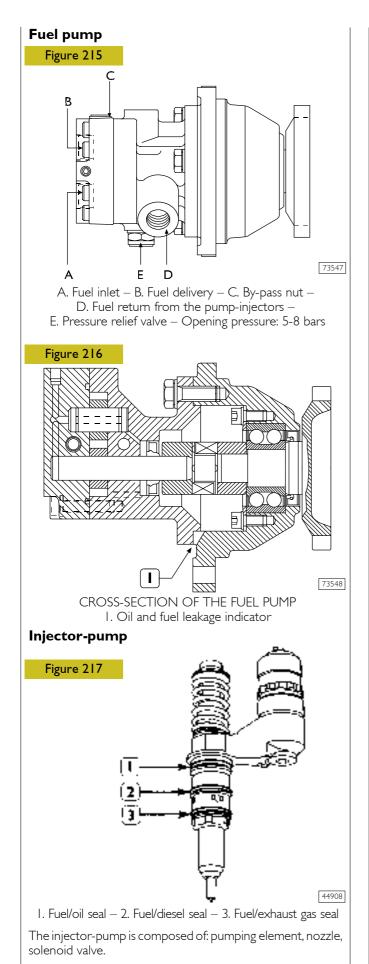




FUEL FEED

Fuel feed is obtained by means of a pump, fuel filter and pre-filter, 6 pump-injectors controlled by the camshaft by means of rockers and by the electronic control unit.





Pumping element

The pumping element is operated by a rocker arm governed directly by the cam of the camshaft.

The pumping element is able to ensure a high delivery pressure. The return stroke is made by means of a return spring.

Nozzle

Garages are authorized to perform fault diagnosis solely on the entire injection system and may not work inside the injector-pump, which must only be replaced.

A specific fault-diagnosis program, included in the control unit, is able to check the operation of each injector (it deactivates one at a time and checks the delivery of the other five). Fault diagnosis makes it possible to distinguish errors of an electrical origin from ones of a mechanical/hydraulic origin. It indicates broken pump-injectors.

It is therefore necessary to interpret all the control unit error messages correctly.

Any defects in the injectors are to be resolved by replacing them.

Solenoid valve

The solenoid, which is energized at each active phase of the cycle, via a signal from the control unit, controls a slide valve that shuts off the pumping element delivery pipe.

When the solenoid is not energized, the valve is open, the fuel is pumped but it flows back into the return pipe with the normal transfer pressure of approximately 5 bars.

When the solenoid is energized, the valve shuts and the fuel, not being able to flow back into the return pipe, is pumped into the nozzle at high pressure, causing the needle to lift.

The amount of fuel injected depends on the length of time the slide valve is closed and therefore on the time for which the solenoid is energized.

The solenoid valve is joined to the injector body and cannot be removed.

On the top there are two screws securing the electrical wiring from the control unit.

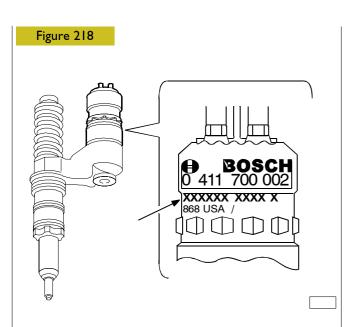
To ensure signal transmission, tighten the screws with a torque wrench to a torque of 1.36 - 1.92 Nm (0.136 - 0.192 kgm).

775010 Replacing injectors-pump

Injectors have to be replaced with great care (for their removal see the description on pages 44 and 45, for fitting them see the description on pages 85 and 86).

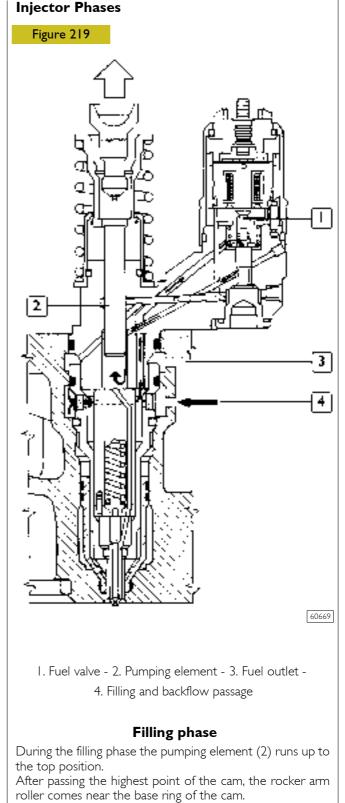


If this job is done with the engine on the vehicle, before removing the injectors-pump drain off the fuel contained in the pipes in the cylinder head by unscrewing the delivery and return fittings on the cylinder head.



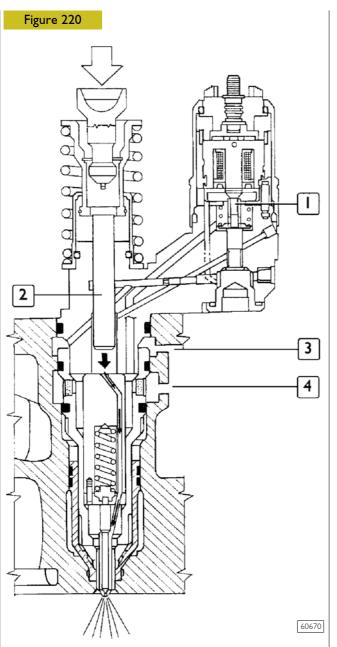
For each injector replaced, hook up to the MODUS station and, when asked by the program, enter the code punched on the injector (\rightarrow) to reprogram the control unit.

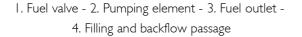
When checking the clearance of the rocker arms, it is important to check the injector-pump pre-load.



The fuel valve (1) is open and fuel can flow into the injector via the bottom passage (4) of the cylinder head.

Filling continues until the pumping element reaches its top limit.



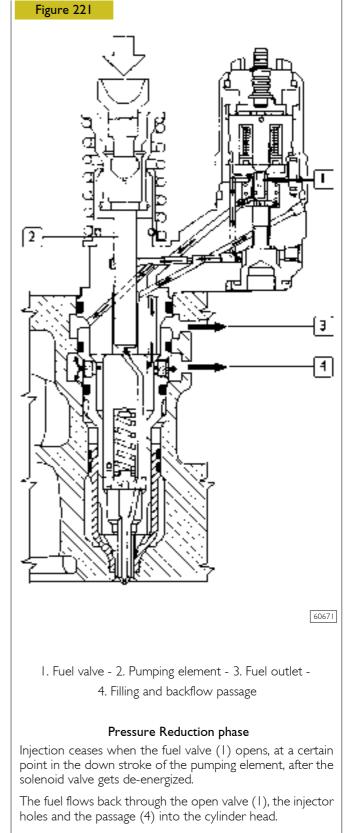


Injection phase

The injection phase begins when, at a certain point in the down phase of the pumping element, the solenoid valve gets energized and the fuel valve (1) shuts.

The moment delivery begins, appropriately calculated by the electronic control unit, depends on the working conditions of the engine.

The cam continues with the rocker arm to push the pumping element (2) and the injection phase continues as long as the fuel valve (1) stays shut.



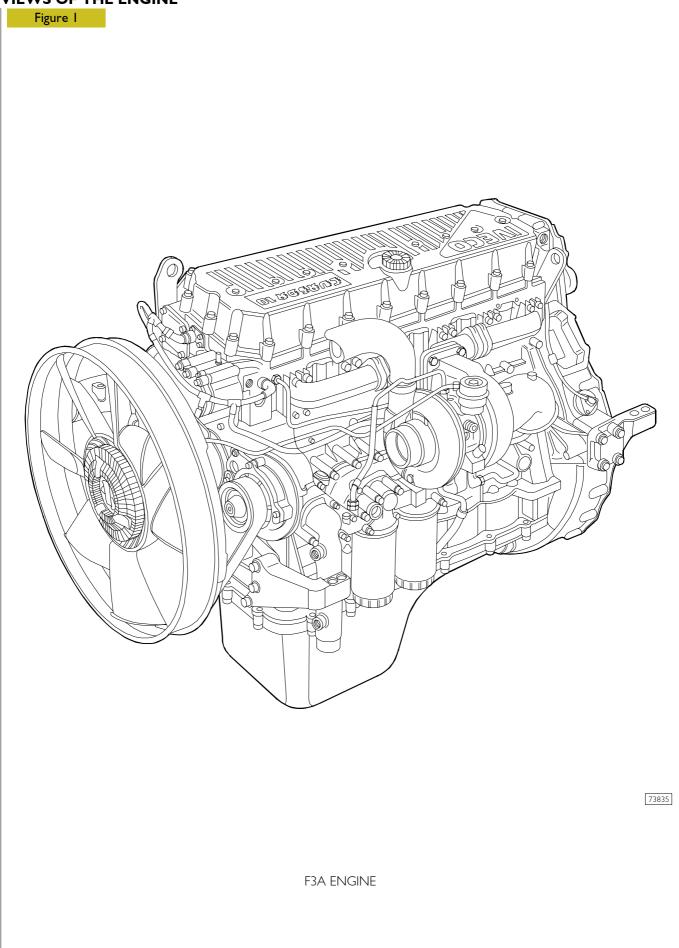
The time for which the solenoid valve stays energized, appropriately calculated by the electronic control unit, is the duration of injection (delivery) and it depends on the working conditions of the engine.

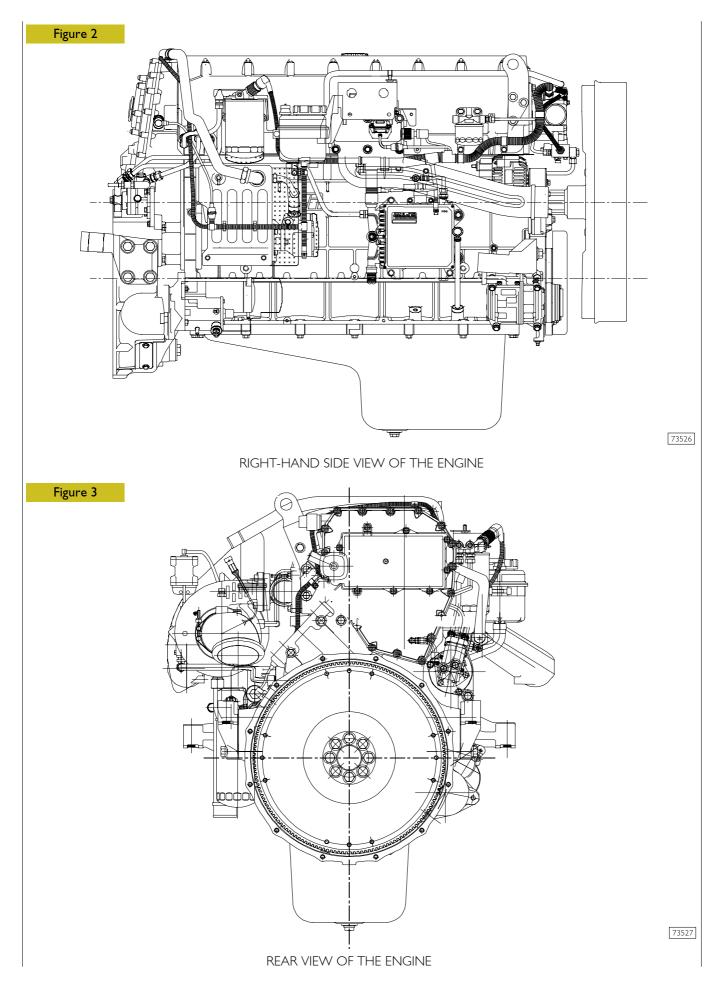
F3A Engine

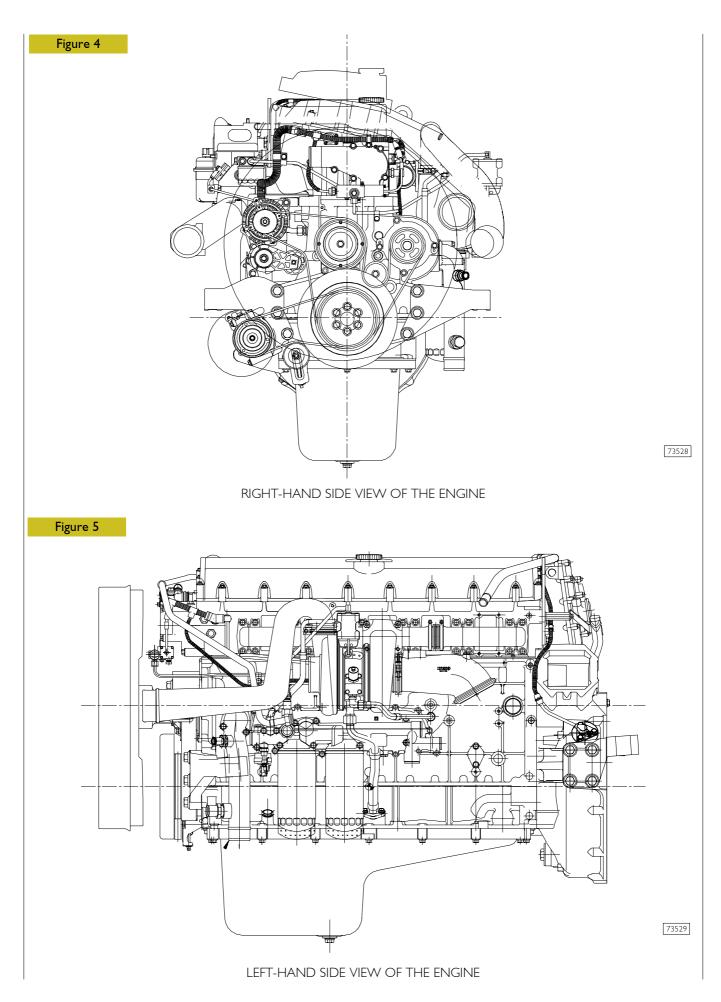
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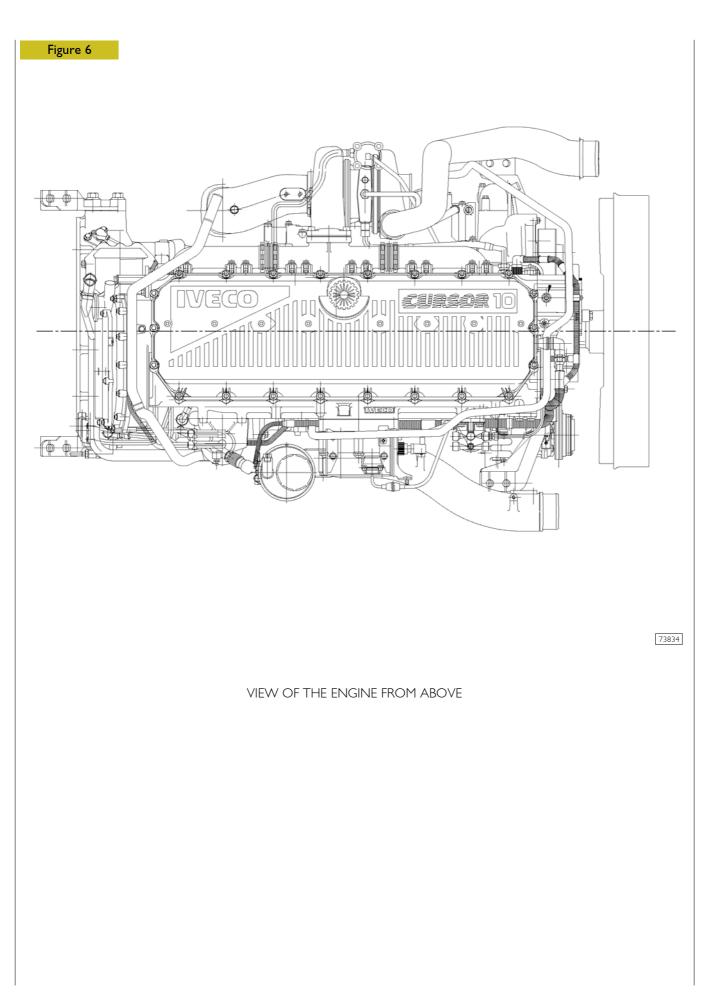
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VIEWS OF THE ENGINE



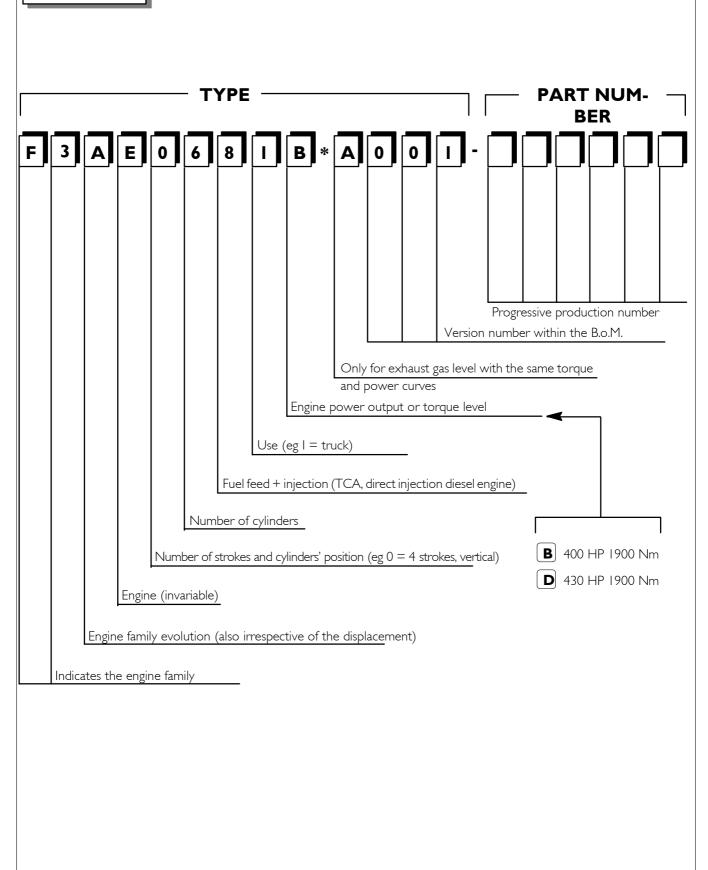




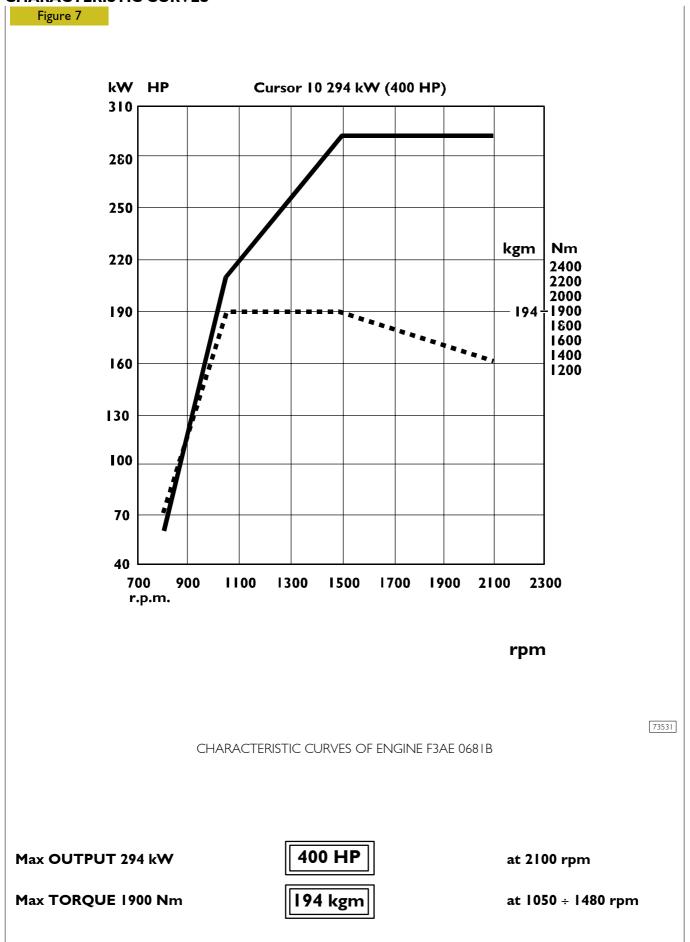


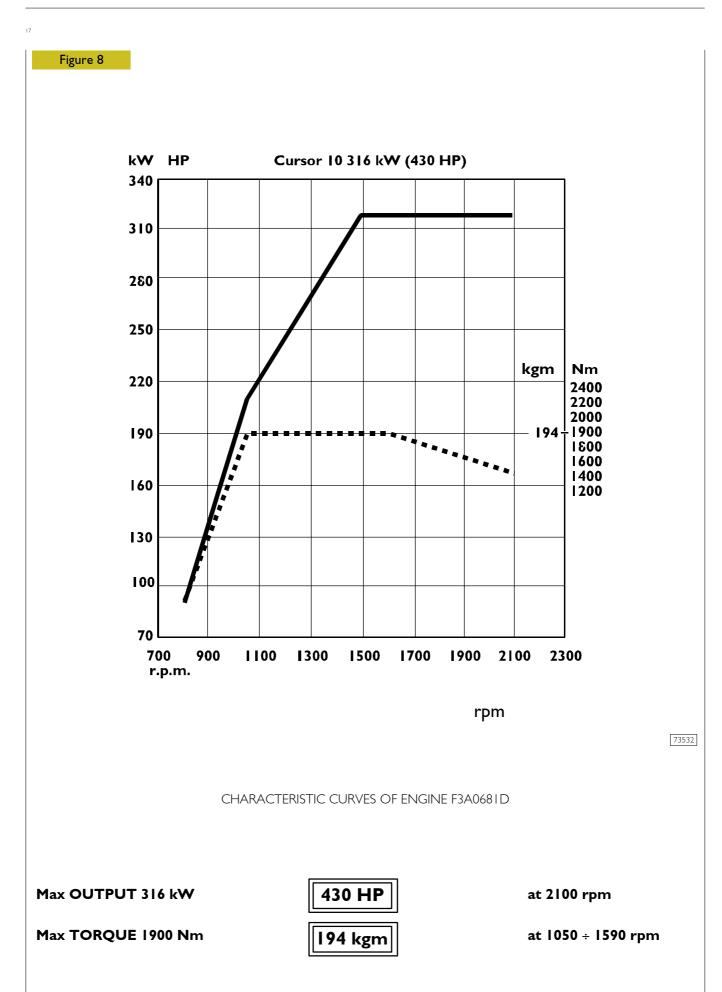
TECHNICAL DESIGNATION

ENGINE



CHARACTERISTIC CURVES





GENERAL CHARACTERISTICS

	Туре		F3AE0681B	F3AE0681D
A	Cycle		4-stroke Diesel engine	
	Fuel feed		Turbocharged w	vith aftercooler
	Injection		Dire	ect
	No. of cylinders		6 in	line
	Bore	mm	125	
	Stroke	mm	140	
	Total displacement	cm ³	10300	
Q	Compression ratio		17 ± 0.8	
	Max output	KW (HP)	294 (400)	316 (430)
		rpm	2100	2100
	Max. torque	Nm (kgm)	1900 (194)	1900 (194)
		rpm	1050 ÷ 1480	1050 ÷ 1590
	Engine idling speed, no load	rpm	550 :	±25
	Maximum engine speed, no load	rpm	2550	±20

	Туре		F3A
	VALVE TIMING opens before T.D.C. closes after B.D.C.	A B	16° 32°
	opens before B.D.C. closes after T.D.C.	D C	50° 9°
	For timing check X { Running X	mm mm mm	- - 0.35 to 0.45 0.45 to 0.55
	FEED Injection type: Bosch		Through fuel pump - filters With electronically regulated injectors PDE 31 pump injectors controlled by overhead camshaft
	Nozzle type		_
	Injection order		I - 4 - 2 - 6 - 3 - 5
bar	Injection pressure Injector calibration	bar bar	1500 290

	Туре	F3A
Î	SUPERCHARGING	
UP	Turbocharger type	Variable geometry Holset HY 55 V
	LUBRICATION	Forced by gear pump, pressure control valve, oil filter
bar	Oil pressure with hot engine (100°C ±5°C):	
	at idling speed bar	1.5
	at maximum rpm bar	5
	COOLING	By centrifugal pump, regulating thermostat, viscostatic fan, radiator and heat exchanger
	Water pump control	By belt
	Thermostat	N. I
	initial opening	~84°C ±2°C
	maximum opening	94°C ±2°C
	OIL FILLING	
	Total capacity at 1 st filling	
	litres	30
	kg	29.8
	Capacities	
	- engine sump min level litres	17
		15.3
Fiat Lubrificanti Urania Turbo LD	kg - engine sump max level	0.0
(according to	litres	25
E3-96 standard)	kg	22.5
Urania Turbo (according to E2-96 standard)	- quantity in circulation that does not flow back	
	to the engine sump	7
	litres	7
	kg	6.3
	 quantity contained in the cartridge filter (which has to be added to the 	
	cartridge filter refill)	
	litres	2.5
	kg	2.3

ASSEMBLY CLEARANCE DATA

	Туре	F3A	
	K AND ISM COMPONENTS	mm	1
	Bores for cylinder liners: upper Ø I lower	42.000 to 40.000 to	
L Ø2	Cylinder liners: external diameter: wper Ø2 lower length L	4 .96 to 4 .986 39.890 to 39.915	
	Cylinder liners - crankcase bores upper lower	0.014 to 0.064 0.085 to 0.135	
	External diameter Ø2		
Ø3 × Selection class	Cylinder sleeve inside diameter Ø3A* inside diameter Ø3B* Protrusion X	125.000 to 125.013 125.011 to 125.024 0.045 to 0.075	
	Pistons: measuring dimension X external diameter ØIA [●] external diameter ØIB [○] pin bore Ø2	NUERAL 18 124.884 to 124.896 124.895 to 124.907 50.010 to 1	MAHLE - MONDIAL 18 124.881 to 124.893 124.892 to 124.904 50.018
* Selection class	Piston - cylinder sleeve A* B*	0.104 to 0.129 0.093 to 0.118	0.107 to 0.132 0.096 to 0.131
PHRS A <	Piston diameter ØI		
X	Pistons protrusion X	0.23 to 0.53	
Ø3	Gudgeon pin Ø3	49.994 to 50.000	
	Gudgeon pin - pin housing	0.010 to	0.024

Class A pistons supplied as spares.

Class A pistons are fitted in production only and are not supplied as spares.